




FOSSIL & HYDRO ENGINEERING

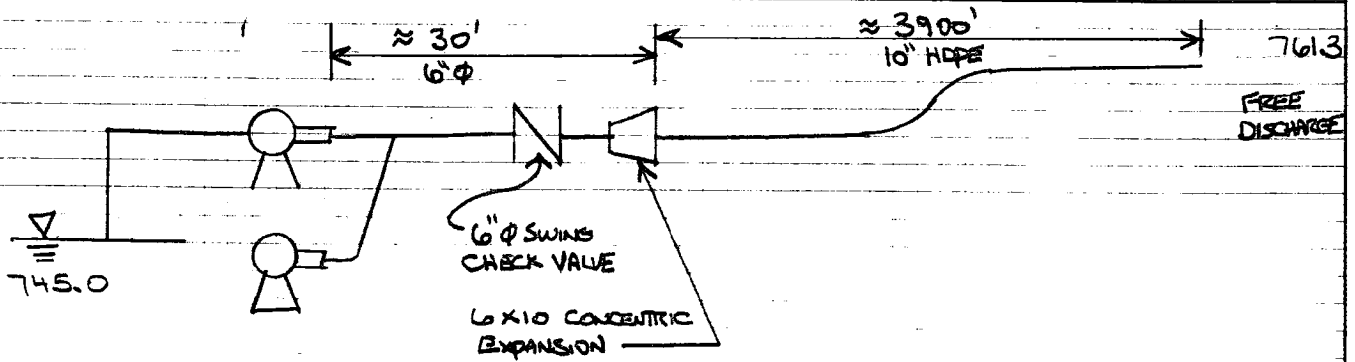
CALCULATION COVER SHEET

DEPT/SECTION IDENTIFIER	
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FOR TVA USE ONLY		
Each time these calculations are revised, preparers must ensure that the original (R0) RIMS accession number is filled in.		
REV	(for RIM'S use)	RIMS accession number
R0		
R_		
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R_		

NAME OF PLANT/UNIT	KINGSTON FOSSIL		ORGANIZATION		
			PE&E		
CALCULATION TITLE	COAL YARD PIPING - STORMWATER FORCE MAIN HYDRAULIC CALCULATIONS.		DEPARTMENT		
			CIVIL		
	SIGNATURE REQUIRED		INITIALS ONLY		
REVISION NO.	R0		R1	R2	R3
PREPARED					
CHECKED					
APPROVED					
DATE	7/6/00				
REMARKS/ABSTRACT	DETERMINE PUMP OPERATING POINT FOR SINGLE & DUAL PUMP OPERATION. THESE VALUES WILL BE USED TO DETERMINE POND DRAW DOWN TIME. (SEE CALC FOR FLOOD ANALYSIS BY PARSONS).				

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AUTHOR T BROWN				SHEET	OF



PURPOSE: EVALUATE PROPOSED PIPING SYSTEM AND PUMP SELECTION TO DETERMINE OPERATING POINT FOR SINGLE PUMP OPERATION AND DUAL-PARALLEL PUMP OPERATION.

① COMPUTE SYSTEM HEAD LOSSES

$$H_t = h_{stat} + h_{fs} + \sum h_{ms} + h_{f_{6''}} + h_{f_{10''}} + \sum h_{qm} \quad (\text{REF 1})$$

WHERE:  $H_t$  = TOTAL SYSTEM HEAD (ft)

$h_{stat}$  = STATIC HEAD (ft)

$h_{fs}$  = SUCTION HEAD LOSS (ft)

$\sum h_{ms}$  = SUM OF MINOR SUCTION LOSSES

$h_{f_{6''}}$  = HEAD LOSS IN 6" PIPE

$h_{f_{10''}}$  = HEAD LOSS IN 10" PIPE

$\sum h_{qm}$  = SUM OF MINOR HEAD LOSSES IN 6" & 10" PIPE

USE DARCY-WEISBACH EQUATION FOR HEAD LOSS DUE TO FRICTION

(PG 3-110, REF 2)

$$h_{fL} = f \frac{L}{D} \frac{V^2}{2g}$$

WHERE  $h_{fL}$  = FRICTION LOSS IN PIPE (ft)

$L$  = PIPE LENGTH (ft)

$D$  = PIPE DIAMETER (ft)

$V$  = VELOCITY (ft/sec)

$g$  = GRAVITATIONAL CONSTANT = 32.2 ft/sec<sup>2</sup>

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FOR 6" PIPE, USE SCH 40, STEEL, I.D. = 6.065 in = 0.5054 ft (pg 7-9, REF 2)  
 FOR 10" HDPE, SDR 17, I.D. = 9.41 in = 0.7842 ft (Bulletin 301, REF 3)

AREA 6" =  $\pi (0.5054)^2 / 4 = 0.2006 \text{ ft}^2$   
 AREA 10" =  $\pi (0.7842)^2 / 4 = 0.4830 \text{ ft}^2$

\* ASSUME SUCTION HEAD LOSS AND MINOR SUCTION LOSSES NEGLIGIBLE.

$H_T = h_{STAT} + h_{f6} + h_{f10} + \Sigma h_{LM}$

FOR MINOR LOSSES, THE FOLLOWING ARE USED.

$h_{LM} = K \frac{V^2}{2g}$  (pg 3-110, REF 2)

WHERE K = RESISTANCE COEFFICIENT

- FOR 6" Y CONNECTION, USE STD TEE  $K = 0.90$  (SECTION 3, REF 2)
- FOR 6" CHECK VALVE,  $K = 1.5$
- FOR 6" → 10" ENLARGEMENT (GRADUAL)  $K = \left(1 - \frac{d_1^2}{d_2^2}\right)^2$   
 $K = \left(1 - \frac{(0.5054)^2}{(0.7842)^2}\right)^2$   
 $K = 0.34$
- FOR 10" PIPE EXIT  $K = 1.0$

$h_{LM} = (0.90 + 1.5 + 0.34) \frac{V_6^2}{2g} + 1.0 \frac{V_{10}^2}{2g}$

$h_{STAT} = h_2 - h_1 = 761.3 - 745.0 = 16.3 \text{ ft}$

$h_{f6}$  &  $h_{f10}$  AND  $h_{LM}$  DEPEND ON THE RATE OF FLOW. IN ORDER TO DEVELOP A SYSTEM CURVE, SEVERAL FLOW RATES MUST BE EXAMINED TO ESTABLISH THE SYSTEM RELATIONSHIP BETWEEN FLOW RATE & HEAD LOSS.

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FOR ALL CALCULATIONS, IT IS ASSUMED THAT THE SYSTEM WILL CONVEY FRESH WATER AS 60°F, WITH NEGLIGIBLE SOLIDS CONTENT. THE KINEMATIC VISCOSITY  $\nu = 1.2 \times 10^{-5} \text{ ft}^2/\text{SEC}$  (REF 2). FOR RELATIVE ROUGHNESS ( $e/D$ ) THE FOLLOWING VALUES WILL BE USED FOR THE 6" & 10" PIPE

FOR 6" STL PIPE,  $e = 0.00015$ ,  $e/D = 0.00015/0.5054 = 0.0003$   
 FOR 10" HDPE  $e = 0.000005$   $e/D = 0.000005/0.7842 = 0.000006$

FOR  $Q = 1000 \text{ GPM} = 2.228 \text{ ft}^3/\text{s}$   $R = \text{REYNOLD'S NUMBER} = \frac{Dv}{\nu}$   
 $v_6 = \frac{Q}{A} = \frac{2.228 \text{ ft}^3/\text{s}}{0.2006 \text{ ft}^2} = 11.1 \text{ ft/s}$   $R = \frac{0.5054 \text{ ft} \times 11.1 \text{ ft/s}}{1.2 \times 10^{-5} \text{ ft}^2/\text{s}} = 467495$

FROM MOODY DIAGRAM  $f_6 = 0.0165$  (REF 4)

$v_{10} = \frac{Q}{A} = \frac{2.228 \text{ ft}^3/\text{s}}{0.4830 \text{ ft}^2} = 4.61 \text{ ft/s}$   $R = \frac{0.7842 \times 4.61 \text{ ft/s}}{1.2 \times 10^{-5} \text{ ft}^2/\text{s}} = 301264$   
 $f_{10} \approx 0.0143$

then  $h_{f6} = f \frac{L}{D} \frac{v^2}{2g} = 0.0165 \times \frac{30 \text{ ft}}{0.5054} \times \frac{(11.1)^2}{2 \times 32.2} = 1.87 \text{ ft}$

$h_{f10} = f \frac{L}{D} \frac{v^2}{2g} = 0.0143 \times \frac{3900 \text{ ft}}{0.7842} \times \frac{(4.61)^2}{2 \times 32.2} = 23.47 \text{ ft}$

$h_{em6} = (0.9 + 1.5 + 0.34) \frac{v_6^2}{2 \times 32.2} = 2.74 \frac{(11.1)^2}{64.4} = 5.24 \text{ ft}$

$h_{em10} = 1.0 \times \frac{v_{10}^2}{64.4} = \frac{(4.61)^2}{64.4} = 0.33 \text{ ft}$

$H_T @ Q = 1000 \text{ gpm} = 16.3 \text{ ft} + 1.87 \text{ ft} + 23.47 \text{ ft} + 5.24 \text{ ft} + 0.33 \text{ ft} = 47.21 \text{ ft}$

$H_T @ 1000 \text{ gpm} = 47.21 \text{ ft}$

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For Q = 1200 gpm, =  $1200 \frac{\text{gal}}{\text{min}} \times \frac{1 \text{ ft}^3}{7.48 \text{ gal}} \times \frac{1 \text{ min}}{60 \text{ sec}} = 2.67 \text{ ft}^3/\text{sec}$

For 6"  $V_6 = \frac{Q}{A} = \frac{2.67 \text{ ft}^3/\text{sec}}{0.2006 \text{ ft}^2} = 13.31 \text{ ft}/\text{sec}$   $R = \frac{0.5054 \text{ ft} \times 13.31 \text{ ft}/\text{sec}}{1.2 \times 10^{-5} \text{ ft}^2/\text{sec}}$

R = 560573

FROM MOODY DIAGRAM  $f \approx 0.0161$

(REF 4)

For 10"  $V_{10} = \frac{Q}{A} = \frac{2.67 \text{ ft}^3/\text{sec}}{0.4830 \text{ ft}^2} = 5.53 \text{ ft}/\text{sec}$   $R = \frac{0.7842 \text{ ft} \times 5.53 \text{ ft}/\text{sec}}{1.2 \times 10^{-5} \text{ ft}^2/\text{sec}} = 361386$

FROM MOODY DIAGRAM  $f \approx 0.0141$

then  $h_{f6} = f \frac{L}{D} \frac{V^2}{2g} = 0.0161 \times \frac{20 \text{ ft}}{0.5054 \text{ ft}} \times \frac{(13.31 \text{ ft}/\text{sec})^2}{2 \times 32.2 \text{ ft}/\text{sec}^2} = 2.63 \text{ ft}$

$h_{f10} = f \frac{L}{D} \frac{V^2}{2g} = 0.0141 \times \frac{2900 \text{ ft}}{0.7842 \text{ ft}} \times \frac{(5.53 \text{ ft}/\text{sec})^2}{2 \times 32.2 \text{ ft}/\text{sec}^2} = 33.30 \text{ ft}$

$h_{lm6} = \sum K \frac{V_6^2}{2g} = 2.74 \frac{(13.31 \text{ ft}/\text{sec})^2}{2 \times 32.2 \text{ ft}/\text{sec}^2} = 7.54 \text{ ft}$

$h_{lm10} = \sum K \frac{V_{10}^2}{2g} = 1.0 \frac{(5.53 \text{ ft}/\text{sec})^2}{2 \times 32.2 \text{ ft}/\text{sec}^2} = 0.48 \text{ ft}$

$H_T @ 1200 \text{ gpm} = 16.3 + 2.63 + 33.3 + 7.54 + 0.48 = 60.25 \text{ ft}$

$H_T @ 1200 \text{ gpm} = 60.25 \text{ ft}$

For 1400 gpm =  $1400 \frac{\text{gal}}{\text{min}} \times \frac{1 \text{ ft}^3}{7.48 \text{ gal}} \times \frac{1 \text{ min}}{60 \text{ sec}} = 3.12 \text{ ft}^3/\text{sec}$

For 6"  $V_6 = \frac{Q}{A} = \frac{3.12 \text{ ft}^3/\text{sec}}{0.2006 \text{ ft}^2} = 15.55 \text{ ft}/\text{sec}$   $R = \frac{0.5054 \text{ ft} \times 15.55 \text{ ft}/\text{sec}}{1.2 \times 10^{-5} \text{ ft}^2/\text{sec}}$

R = 654914

FROM MOODY'S DIAGRAM.  $f \approx 0.0160$

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FOR 10"  $V_{10} = \frac{Q}{A} = \frac{3.12 \text{ ft}^3/\text{sec}}{0.4830 \text{ ft}^2} = 6.46 \text{ ft/sec}$

$R = \frac{0.7842 \text{ ft} \times 6.46 \text{ ft/sec}}{1.2 \times 10^{-5} \text{ ft}^2/\text{sec}} = 422161$

FROM MOODY'S DIAGRAM

$f \approx 0.014$

THEN  $h_{fL6} = f \frac{L}{D} \frac{V^2}{2g} = 0.0160 \times \frac{30 \text{ ft}}{0.5054 \text{ ft}} \times \frac{(15.55 \text{ ft/sec})^2}{2 \times 32.2 \text{ ft/sec}^2} = 3.57 \text{ ft}$

$h_{fL10} = f \frac{L}{D} \frac{V^2}{2g} = 0.014 \times \frac{3900 \text{ ft}}{0.7842 \text{ ft}} \times \frac{(6.46 \text{ ft/sec})^2}{2 \times 32.2 \text{ ft/sec}^2} = 45.12 \text{ ft}$

$h_{Lm6} = \sum K \frac{V^2}{2g} = 2.74 \frac{(15.55 \text{ ft/sec})^2}{2 \times 32.2 \text{ ft/sec}^2} = 10.29 \text{ ft}$

$h_{Lm10} = \sum K \frac{V^2}{2g} = 1.0 \frac{(6.46 \text{ ft/sec})^2}{2 \times 32.2 \text{ ft/sec}^2} = 0.65 \text{ ft}$

$H_T @ 1400 \text{ gpm} = 16.3 + 3.57 + 45.12 + 10.29 + 0.65 \text{ ft} = 75.93 \text{ ft}$

$H_T @ 1400 \text{ gpm} = 75.93 \text{ ft}$

FOR  $Q = 1600 \text{ gpm} = 1600 \text{ gal/min} \times \frac{1 \text{ ft}^3}{7.48 \text{ gal}} \times \frac{1 \text{ min}}{60 \text{ sec}} = 3.57 \text{ ft}^3/\text{sec}$

FOR 6"  $V_6 = \frac{Q}{A} = \frac{3.57 \text{ ft}^3/\text{sec}}{0.2006 \text{ ft}^2} = 17.8 \text{ ft/sec}$   $R = \frac{0.5054 \text{ ft} \times 17.8 \text{ ft/sec}}{1.2 \times 10^{-5} \text{ ft}^2/\text{sec}} = 749677$

FROM MOODY DIAGRAM

$f \approx 0.0159$

FOR 10"  $V_{10} = \frac{Q}{A} = \frac{3.57 \text{ ft}^3/\text{sec}}{0.4830 \text{ ft}^2} = 7.39 \text{ ft/sec}$   $R = \frac{0.7842 \text{ ft} \times 7.39 \text{ ft/sec}}{1.2 \times 10^{-5} \text{ ft}^2/\text{sec}} = 482937$

FROM MOODY DIAGRAM

$f \approx 0.0139$

THEN  $h_{fL6} = f \frac{L}{D} \frac{V^2}{2g} = 0.0159 \times \frac{30 \text{ ft}}{0.5054 \text{ ft}} \times \frac{(17.8 \text{ ft/sec})^2}{2 \times 32.2 \text{ ft/sec}^2} = 4.64 \text{ ft}$

$h_{fL10} = f \frac{L}{D} \frac{V^2}{2g} = 0.0139 \times \frac{3900 \text{ ft}}{0.7842 \text{ ft}} \times \frac{(7.39 \text{ ft/sec})^2}{2 \times 32.2 \text{ ft/sec}^2} = 58.62 \text{ ft}$

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$$h_{Lmb} = \sum K \frac{V^2}{2g} = 2.74 \frac{(17.8 \text{ ft/sec})^2}{2 \times 32.2 \text{ ft/sec}^2} = 13.48 \text{ ft}$$

$$h_{Lm10} = \sum K \frac{V^2}{2g} = 1.0 \frac{(7.39 \text{ ft/sec})^2}{2 \times 32.2 \text{ ft/sec}^2} = 0.85 \text{ ft}$$

$$H_T @ 1600 \text{ gpm} = 16.3 + 4.64 + 58.62 + 13.48 + 0.85 = 93.89 \text{ ft}$$

$$H_T @ 1600 \text{ gpm} = 93.89 \text{ ft}$$

$$\text{FOR } Q = 1800 \text{ gpm} = 1800 \text{ gal/min} \times 1 \text{ ft}^3 / 7.48 \text{ gal} \times 1 \text{ min} / 60 \text{ sec} = 4.01 \text{ ft}^3/\text{sec}$$

$$\text{FOR } 6" \quad V_6 = Q/A = \frac{4.01 \text{ ft}^3/\text{sec}}{0.2006 \text{ ft}^2} = 19.99 \text{ ft/sec} \quad R = \frac{0.5054 \text{ ft} \times 19.99 \text{ ft/sec}}{1.2 \times 10^{-5} \text{ ft}^2/\text{sec}} = 841912$$

FROM MOODY DIAGRAM  $f \approx 0.0157$

$$\text{FOR } 10" \quad V_{10} = Q/A = \frac{4.01 \text{ ft}^3/\text{sec}}{0.4830 \text{ ft}^2} = 8.3 \text{ ft/sec} \quad R = \frac{0.7842 \text{ ft} \times 8.30 \text{ ft/sec}}{1.2 \times 10^{-5} \text{ ft}^2/\text{sec}} = 542405$$

FROM MOODY DIAGRAM  $f \approx 0.0136$

$$h_{fL6} = f \frac{L}{D} \frac{V^2}{2g} = 0.0157 \times \frac{30 \text{ ft}}{0.5054 \text{ ft}} \times \frac{(19.99 \text{ ft/sec})^2}{2 \times 32.2 \text{ ft/sec}^2} = 5.78 \text{ ft}$$

$$h_{fL10} = f \frac{L}{D} \frac{V^2}{2g} = 0.0136 \times \frac{3900 \text{ ft}}{0.7842 \text{ ft}} \times \frac{(8.3 \text{ ft/sec})^2}{2 \times 32.2 \text{ ft/sec}^2} = 72.35 \text{ ft}$$

$$h_{Lmb} = \sum K \frac{V^2}{2g} = 2.74 \times \frac{(19.99 \text{ ft/sec})^2}{2 \times 32.2 \text{ ft/sec}^2} = 17.00 \text{ ft}$$

$$h_{Lm10} = \sum K \frac{V^2}{2g} = 1.0 \times \frac{(8.3 \text{ ft/sec})^2}{2 \times 32.2 \text{ ft/sec}^2} = 1.07 \text{ ft}$$

$$H_T @ 1800 \text{ gpm} = 16.3 + 5.78 + 72.35 + 17.0 + 1.07 = 112.5 \text{ ft}$$

$$H_T @ 1800 \text{ gpm} = 112.5 \text{ ft}$$

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FOR Q = 2000 gpm =  $2000 \text{ gal/min} \times \frac{1 \text{ ft}^3}{7.48 \text{ gal}} \times \frac{1 \text{ min}}{60 \text{ sec}} = 4.46 \text{ ft}^3/\text{sec}$

FOR 6"  $V_6 = \frac{Q}{A} = \frac{4.46 \text{ ft}^3/\text{sec}}{0.2006 \text{ ft}^2} = 22.23 \text{ ft/sec}$

$R = \frac{0.5054 \text{ ft} \times 22.23 \text{ ft/sec}}{1.2 \times 10^{-5} \text{ ft}^2/\text{sec}} = 936254$

FROM MOODY DIAGRAM,  $f \approx 0.0154$

FOR 10"  $V_{10} = \frac{Q}{A} = \frac{4.46 \text{ ft}^3/\text{sec}}{0.4830 \text{ ft}^2} = 9.23 \text{ ft/sec}$

$R = \frac{0.7842 \text{ ft} \times 9.23 \text{ ft/sec}}{1.2 \times 10^{-5} \text{ ft}^2/\text{sec}} = 603181$

FROM MOODY DIAGRAM  $f \approx 0.0130$

$h_{fL_6} = f \frac{L}{D} \frac{V^2}{2g} = 0.0154 \times \frac{30 \text{ ft}}{0.5054 \text{ ft}} \times \frac{(22.23 \text{ ft/sec})^2}{2 \times 32.2 \text{ ft/sec}^2} = 7.02 \text{ ft}$

$h_{fL_{10}} = f \frac{L}{D} \frac{V^2}{2g} = 0.0130 \times \frac{3900 \text{ ft}}{0.7842 \text{ ft}} \times \frac{(9.23 \text{ ft/sec})^2}{2 \times 32.2 \text{ ft/sec}^2} = 85.53 \text{ ft}$

$h_{Lm_6} = \sum K \frac{V^2}{2g} = 2.74 \times \frac{(22.23 \text{ ft/sec})^2}{2 \times 32.2 \text{ ft/sec}^2} = 21.03 \text{ ft}$

$h_{Lm_{10}} = \sum K \frac{V^2}{2g} = 1.0 \times \frac{(9.23 \text{ ft/sec})^2}{2 \times 32.2 \text{ ft/sec}^2} = 1.32 \text{ ft}$

$H_T @ 2000 \text{ gpm} = 16.3 + 7.02 + 85.53 + 21.03 + 1.32 = 131.2 \text{ ft}$

$H_T @ 2000 \text{ gpm} = 131.2 \text{ ft}$



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WITH SYSTEM HEAD LOSSES KNOWN OVER A RANGE OF FLOW CONDITIONS, PLOT SYSTEM HEAD CURVE ON PUMP CURVE TO DETERMINE OPERATING POINT FOR PUMP. SINCE 2 PUMPS MAY BE USED SIMULTANEOUSLY IN PARALLEL, ALSO PLOT SYSTEM HEAD CURVE ON 2-PUMP CURVE TO DETERMINE OPERATING POINT USING 2-PUMPS IN PARALLEL OPERATION.

TO CONSTRUCT PUMP CURVE FOR PARALLEL OPERATION, ADD PUMP CAPACITIES FOR GIVEN HEAD, NEGLECTING SUCTION LOSSES.

SUMMARY OF RESULTS:

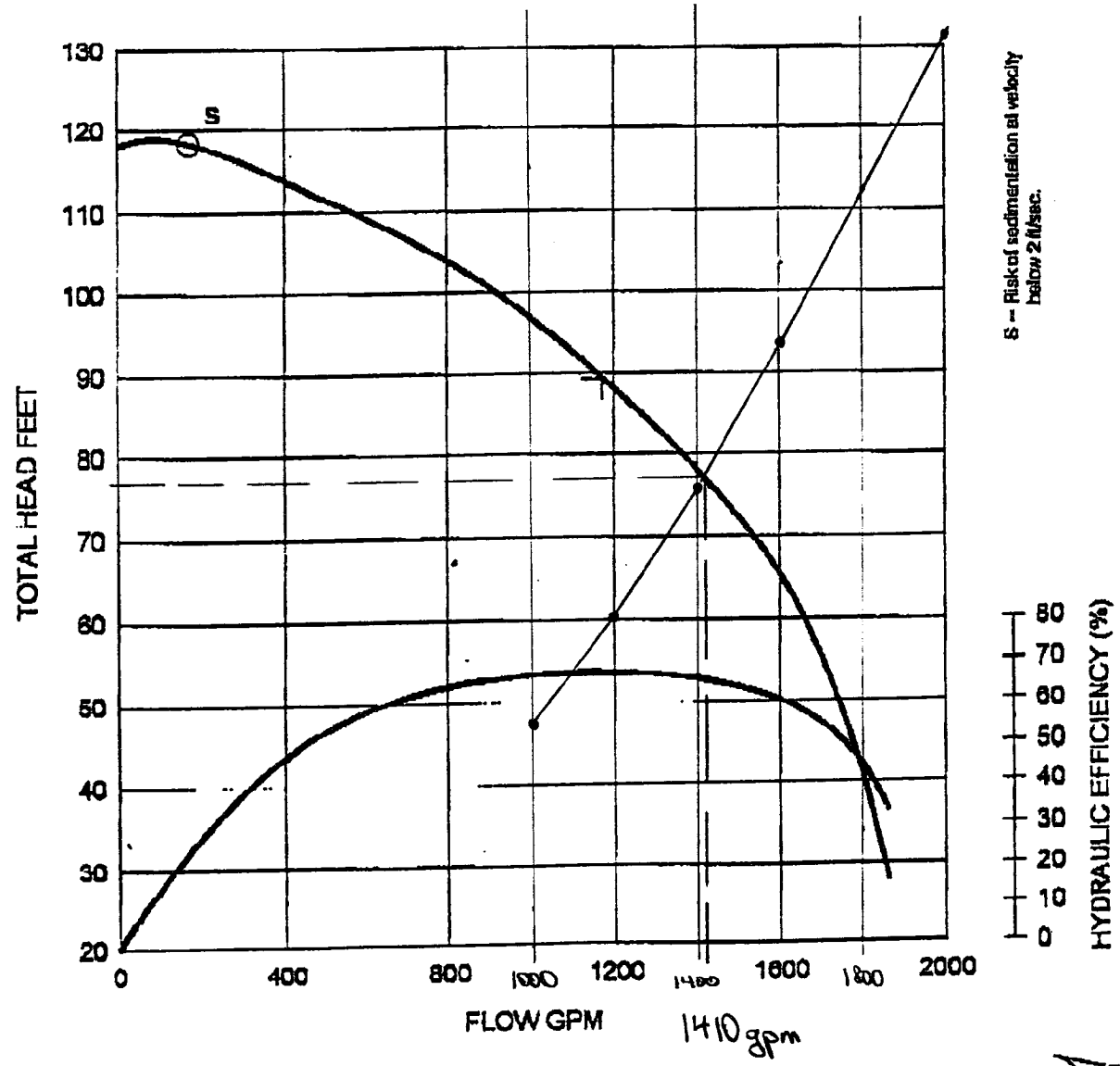
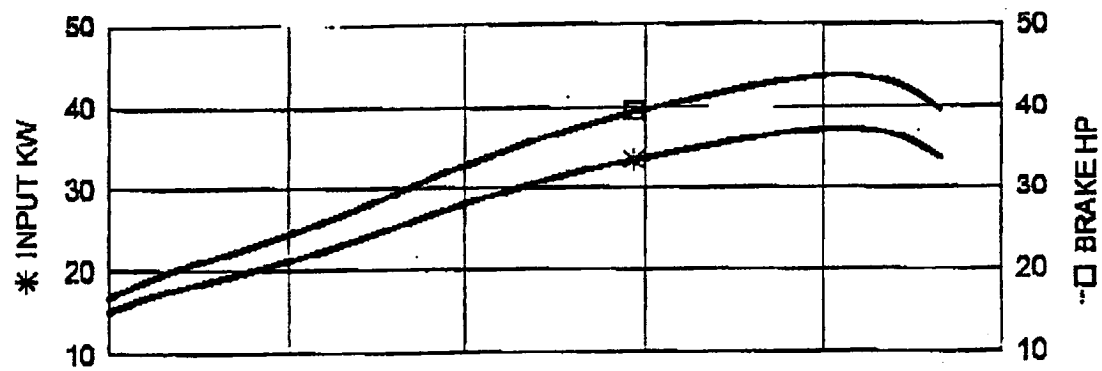
SINGLE PUMP OPERATING PT = 1410 gpm @ 77-ft TDH

2 PUMPS IN PARALLEL, OPERATING PT = 1700 gpm @ 102.7-ft TDH

ADDING 2ND PUMP INCREASES FLOW BY  $(1700 - 1410) / 1410 = 20.6\%$

CONFIG		<b>C-3201</b> <b>462 Impeller</b>	SECTION	PAGE
HP/HS			<b>3</b>	<b>9</b>
PHASE	VANES		SUPERSEDES	ISSUED
3	3	2/88	6/94	

SINGLE PUMP



S - Risk of sedimentation at velocity below 2 ft/sec.

HYDRAULIC EFFICIENCY (%)

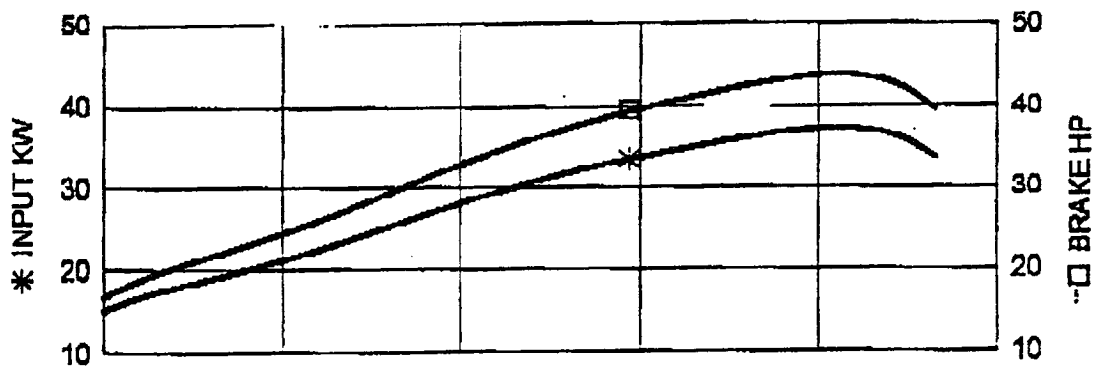
1410 gpm



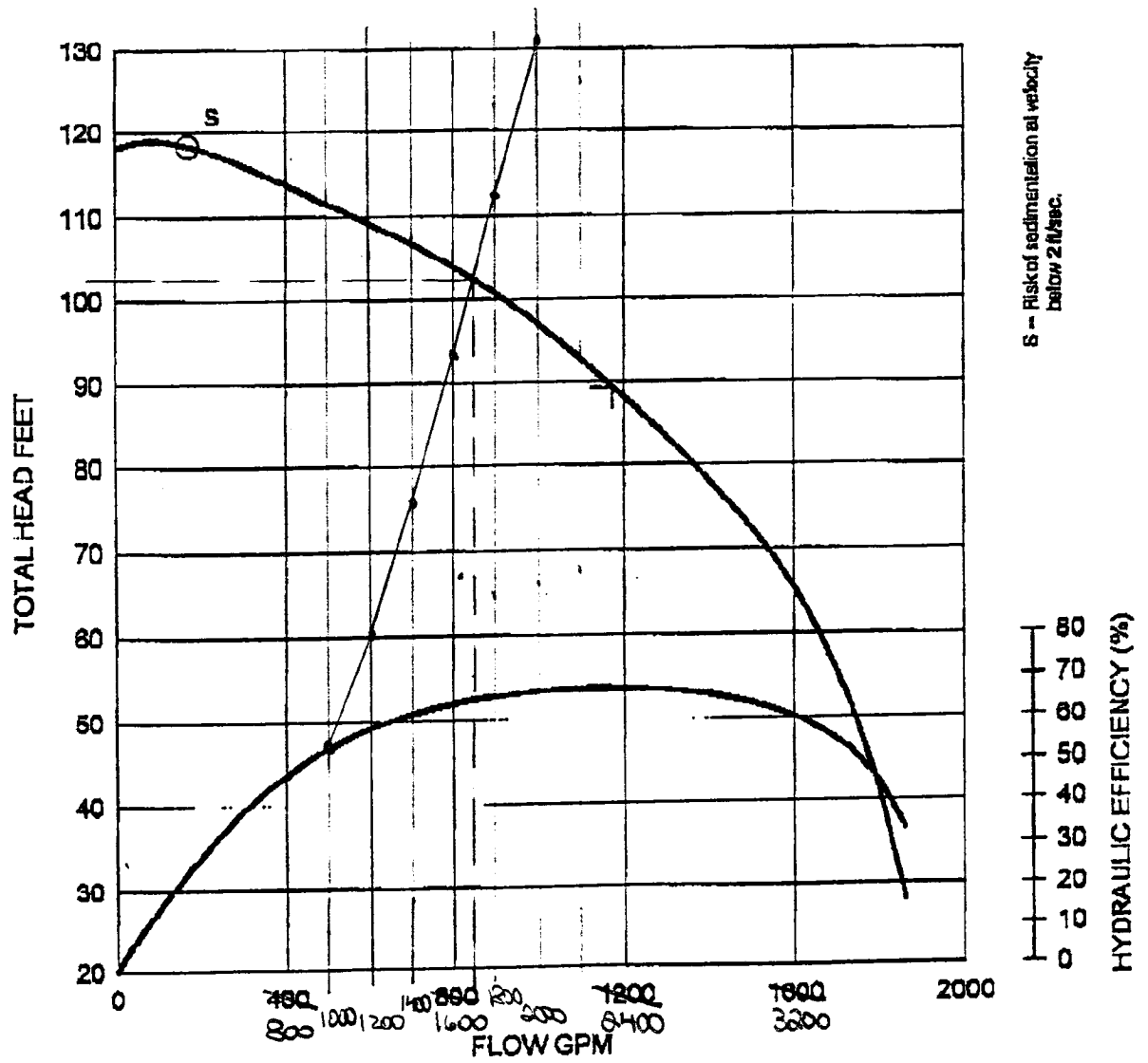
REF 5

CONFIG		<b>C-3201</b> <b>462 Impeller</b>	SECTION	PAGE
HP/HS			<b>3</b>	<b>9</b>
PHASE	VANES		SUPERSEDES	ISSUED
<b>3</b>	<b>3</b>	<b>2/88</b>	<b>6/94</b>	

2 PUMPS IN PARALLEL



\* SECTION LOSSES NOT INCLUDED



S - Risk of sedimentation at velocity below 2 ft/sec.

2 pumps @ 1/2 1700 gpm

HYDRAULIC EFFICIENCY (%)

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REFERENCES:

- ① "WASTEWATER ENGINEERING: COLLECTION AND PUMPING OF WASTEWATER", METCALF & EDDY, INC., MCGRAW-HILL 1981.
- ② "CAMERON HYDRAULIC DATA", INGERSOLL-RAND COMPANY, 17th ed., 1988
- ③ "BULLETIN No. 301" PLEXCO PIPE, REV. 3-93.
- ④ DAUGHERTY, R.L., FRANZINI, J.B., FENNEMORE, E.J., "FLUID MECHANICS WITH ENGINEERING APPLICATIONS", 8th ed, MCGRAW-HILL, 1985.
- ⑤ FAX TRANSMISSION FROM JOHN BOUCHARD & SONS CO., PUMP CURVE FOR FLYGHT C-3901 PUMP. MAY 5, 2000

# PE 3408 Industrial Piping System Pipe Data and Pressure Ratings Bulletin No. 301



(Pipe weights are calculated in accordance with PPI TR-7)  
Average inside diameter calculated on minimum wall plus 6%.

Pressure Ratings are for water at 73 F. For other fluids and service temperatures ratings may differ,  
refer to Application Note No. 6 Chemical and Environmental Considerations.

IPS* Pipe Size	Nominal OD (in.)	Minimum Wall (in.)	Average ID (in.)	Weight LB/FT	Minimum Well (in.)	Average ID (in.)	Weight LB/FT	Minimum Well (in.)	Average ID (in.)	Weight LB/FT	Minimum Well (in.)	Average ID (in.)	Weight LB/FT	Minimum Well (in.)	Average ID (in.)	Weight LB/FT	IPS* Pipe Size
1 1/4"																	1 1/4"
1 1/2"																	1 1/2"
2"	2.375	0.140	2.078	0.43													2"
3"	3.500	0.206	3.063	0.93													3"
4"	4.500	0.265	3.938	1.54	0.214	4.046	1.26										4"
5"	5.375	0.316	4.705	2.20	0.256	4.832	1.80	0.207	4.936	1.47							5"
5 1/2"	5.563	0.327	4.870	2.35	0.265	5.001	1.93	0.214	5.109	1.58							5 1/2"
6"	6.625	0.390	5.798	3.34	0.315	5.957	2.73	0.255	6.084	2.23	0.204	6.193	1.80				6"
7"	7.125	0.419	6.237	3.86	0.339	6.406	3.16	0.274	6.544	2.58	0.219	6.661	2.08				7"
8"	8.625	0.507	7.550	5.65	0.411	7.754	4.64	0.332	7.921	3.79	0.265	8.063	3.05				8"
10"	10.750	0.632	9.410	8.78	0.512	9.665	7.21	0.413	9.874	5.87	0.331	10.048	4.75				10"
12"	12.750	0.750	11.160	12.36	0.607	11.463	10.13	0.490	11.711	8.26	0.392	11.919	6.67				12"
13 1/2"	13.375	0.787	11.707	13.60	0.637	12.025	11.15	0.514	12.285	9.09	0.412	12.502	7.35				13 1/2"
14"	14.000	0.824	12.253	14.91	0.667	12.586	12.22	0.538	12.859	9.96	0.431	13.086	8.05				14"
16"	16.000	0.941	14.005	19.46	0.762	14.385	15.97	0.615	14.696	13.02	0.492	14.957	10.51				16"
18"	18.000	1.059	15.755	24.65	0.857	16.183	20.19	0.692	16.533	16.48	0.554	16.826	13.29				18"
20"	20.000	1.176	17.507	30.42	0.952	17.982	24.92	0.769	18.370	20.34	0.615	18.696	16.41				20"
22"	22.000	1.294	19.257	36.81	1.048	19.778	30.19	0.846	20.206	24.62	0.677	20.565	19.87				22"
24"	24.000	1.412	21.007	43.82	1.143	21.577	35.92	0.923	22.043	29.29	0.738	22.435	23.62				24"
26"	26.000	1.529	22.759	51.40	1.238	23.375	42.13	1.000	23.880	34.39	0.800	24.304	27.74				26"
28"	28.000	1.647	24.508	59.62	1.333	25.174	48.86	1.077	25.717	39.89	0.862	26.173	32.20				28"
30"	30.000	1.765	26.258	68.45	1.429	26.971	56.13	1.154	27.554	45.78	0.923	28.043	36.92				30"
32"	32.000	1.882	28.010	77.86	1.524	28.769	63.83	1.231	29.390	52.10	0.985	29.912	42.04				32"
34"	34.000	2.000	29.760	87.91	1.619	30.568	72.06	1.308	31.227	58.79	1.046	31.782	47.44				34"
36"	36.000	2.118	31.510	98.56	1.714	32.366	80.79	1.385	33.064	65.93	1.108	33.651	53.18				36"
42"	42.000				2.000	37.760	109.97	1.615	38.576	89.71	1.292	39.261	72.40				42"
48"	48.000				2.286	43.154	143.64	1.846	44.086	117.20	1.477	44.869	94.58				48"
54"	54.000				2.571	48.549	181.74	2.077	49.597	148.35	1.662	50.477	119.72				54"

\* Industrial PE (polyethylene) pipe sizes are identified by IPS (iron pipe size) diameters which designate the nominal diameter for 12 IPS AND SMALLER PIPE, AND O.D. (outside diameter) for 14 IPS and larger pipe.

† SUBJECT TO MINIMUM ORDER QUANTITIES, AND AVAILABILITY OF TOOLING.

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