

THE USE OF CARBON FIBERS IN WIND TURBINE BLADE DESIGN:

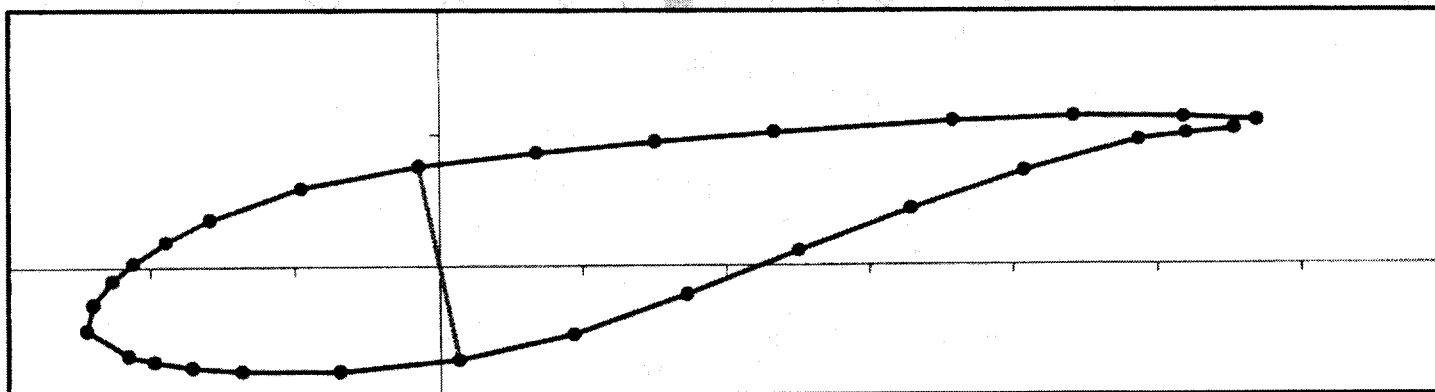
A SERI-8 BLADE EXAMPLE

Cheng-Huat Ong & Stephen W. Tsai

Department of Aeronautics & Astronautics

Stanford University

Stanford, CA 94305-4035



Prepared by
Sandia National Laboratories
Albuquerque, New Mexico 87185 and Livermore, California 94550

Sandia is a multiprogram laboratory operated by Sandia Corporation,
a Lockheed Martin Company, for the United States Department of Energy
under contract DE-AC04-94AL85000.

Approved for public release; further dissemination unlimited.

SAND2000-0478
Unlimited Release
Printed March 2000



SAND 2000-0478
Unlimited Release
Printed March 2000

The Use of Carbon Fibers in Wind Turbine Blade Design: a SERI-8 Blade Example

Cheng-Huat Ong & Stephen W. Tsai
Department of Aeronautics & Astronautics
Stanford University
Stanford CA 94305-4035

Sandia Contract: BE-6196

Abstract

The benefit of introducing carbon fibers in a wind turbine blade was evaluated. The SERI-8 wind turbine blade was used as a baseline for study. A model of the blade strength and stiffness properties was created using the 3D_Beam code; the predicted geometry and structural properties were validated against available data and static test results. Different enhanced models, which represent different volumes of carbon fibers in the blade, were also studied for two design options: with and without bend-twist coupling. Our studies indicate that hybrid blades have excellent structural properties compared to the all-glass SERI-8 blade. Recurring fabrication costs were also included in the study. The cost study highlights the importance of the labor-cost to material-cost ratio in the cost benefits and penalties of fabrication of a hybrid glass and carbon blade.

Acknowledgments

This project was supported by Sandia National Laboratories under Contract No. BE-6196. The Technical Monitor is Dr. Paul S. Veers. The authors would like to thank Paul Veers and Thomas Ashwill for their participation in many beneficial discussions. The authors would like to thank James Tangler for sharing the unpublished blade cost information.

Contents

1.	SERI-8 Blade Dimensions & Properties	1
2.	SERI-8 Blade Baseline Model	10
3.	SERI-8 Blade Enhanced Models	19
4.	Cost Estimates for the SERI-8 Blade	54
5.	Conclusions	66
	References	68



Chapter 1

SERI-8 Blade Dimensions & Properties

A SERI-8 blade is a 7.9-meter blade, which was designed by the Solar Energy Research Institute (SERI, now called the National Renewable Energy Laboratory, NREL). SERI developed a set of thin airfoils (S806A, S805A, S807 and S808) that were incorporated into a 7.9-meter blade, which was geometrically optimized for a 65KW wind turbine [1]. The SERI-8 blades were designed as replacement blades for Aerostar 7.5-meter blades, which are based on the NACA 44xx series airfoils. The differences in geometry between Aerostar blades and SERI-8 blades are given in Tangler et al. [1] and Keller and Smith [2].

We have summarized here pertinent data from Tangler et al. [1], Keller and Smith [2], Jackson [3] and Klingenstein [4], and it was used to generate the SERI-8 baseline finite element model (the code used is 3D-Beam [5]). The structural properties read out from the references were used to validate the SERI-8 finite element model.

1.1 Shape & Dimensions

A SERI-8 blade has various airfoil shapes along its span. The normalized coordinates of these thin airfoil shapes are given in Table 1.1. Table 1.2 provides the span location and chord length of a specific airfoil shape. The initial twist distribution is also given in Table 1.2. Figure 1.1 shows the general dimensions of a SERI-8 blade; the figure also highlights the shape change.

1.2 Section Structural Properties

The section structural properties of the SERI-8 blade are given in Table 1.3, which for the 12-station blade include mass distribution, EI (flapping), skin cross-section area, I_{xx} (torsion), I_{yy} (flapping), and I_{zz} (edgewise).

1.3 Fabrication Information

Figure 1.2 shows the general construction of SERI-8 blades. At the root was a steel root fitting that was about 16 inches long and attached to a filament wound transition spar that measured about 96 inches in length. A 142-inch shear web was then attached to the transition spar. The skins of the shear webs consisted of 6-ply $\pm 45^\circ$ E-glass material. The main skin of the blade was composed of MAT, TRIAX, Core and unidirectional-roving fibers. Detailed laminating information and material data could not be found for the as-built blade. We were forced to work from laminate design information that was substantially changed before fabrication and from back calculations of properties from static test results

(Rotor hub radius = 0.6m, 23.62 in, 1 ft 11.62 in; Twist axis at 30% chord)

Blade Station No.	Blade Radius (in)	Blade Chord c. (in)	Blade Twist B. (deg)	Blade Thickness t. (in)	Rotor Radius r. (in)	Rotor Radius r. (ft)	Rotor Radius r. (m)	%Rotor Radius r/R	Chord/ Radius c/R	Airfoil Section/ Notes
-1	-1.38	n/a	n/a	n/a	23.62	1.97	0.60	0.07	n/a	→ Rotor hub radius
0	0	16.50	n/a	16.50	25.00	2.08	0.63	0.07	0.0490	→ Blade root flange thk
1	6	17.13	n/a	17.13	31.00	2.58	0.79	0.09	0.0508	
2	12	17.83	n/a	17.83	37.00	3.08	0.94	0.11	0.0529	
3	18	18.50	n/a	18.50	43.00	3.58	1.09	0.13	0.0549	
4	24	22.15	29.85	17.18	49.00	4.08	1.24	0.15	0.0657	
5	30	25.79	28.03	15.85	55.00	4.58	1.40	0.16	0.0765	
6	36	29.43	26.28	14.53	61.00	5.08	1.55	0.18	0.0873	
7	42	33.08	24.60	13.21	67.00	5.58	1.70	0.20	0.0982	
8	48	36.72	23.00	11.89	73.00	6.08	1.85	0.22	0.1090	
9	54	40.36	21.46	10.56	79.00	6.58	2.01	0.23	0.1198	
10	60	44.00	20.00	9.24	85.00	7.08	2.16	0.25	0.1306	S808 (@ 60.0°) Max c
11	66	43.99	18.61	8.95	91.00	7.58	2.31	0.27	0.1302	
12	72	43.68	17.28	8.64	97.00	8.08	2.46	0.29	0.1296	
13	78	43.41	16.01	8.32	103.00	8.58	2.62	0.31	0.1288	
14	84	43.09	14.81	7.99	109.00	9.08	2.77	0.32	0.1279	S807 (@ 88.6°)
15	90	42.73	13.67	7.68	115.00	9.58	2.92	0.34	0.1268	
16	96	42.33	12.59	7.52	121.00	10.08	3.07	0.36	0.1256	
17	102	41.89	11.57	7.36	127.00	10.58	3.23	0.38	0.1243	
18	108	41.42	10.61	7.19	133.00	11.08	3.38	0.39	0.1229	
19	114	40.92	9.70	7.01	139.00	11.58	3.53	0.41	0.1214	
20	120	40.40	8.85	6.84	145.00	12.08	3.68	0.43	0.1199	
21	126	39.84	8.04	6.66	151.00	12.58	3.84	0.45	0.1182	
22	132	39.27	7.29	6.48	157.00	13.08	3.99	0.47	0.1165	
23	138	38.66	6.58	6.30	163.00	13.58	4.14	0.48	0.1147	
24	144	38.03	5.93	6.12	169.00	14.08	4.29	0.50	0.1128	
25	150	37.38	5.31	5.94	175.00	14.58	4.44	0.52	0.1109	
26	156	36.71	4.74	5.75	181.00	15.08	4.60	0.54	0.1089	S805A/7 (@ 160.4°)
27	162	36.02	4.22	5.57	187.00	15.58	4.75	0.55	0.1069	
28	168	35.30	3.73	5.40	193.00	16.08	4.90	0.57	0.1047	
29	174	34.57	3.28	5.23	199.00	16.58	5.05	0.59	0.1026	
30	180	33.81	2.87	5.06	205.00	17.08	5.21	0.61	0.1003	
31	186	33.04	2.50	4.89	211.00	17.58	5.36	0.63	0.0980	
32	192	32.25	2.16	4.72	217.00	18.08	5.51	0.64	0.0957	
33	198	31.44	1.85	4.55	223.00	18.58	5.66	0.66	0.0933	
34	204	30.61	1.57	4.38	229.00	19.08	5.82	0.68	0.0908	
35	210	29.76	1.33	4.21	235.00	19.58	5.97	0.70	0.0883	

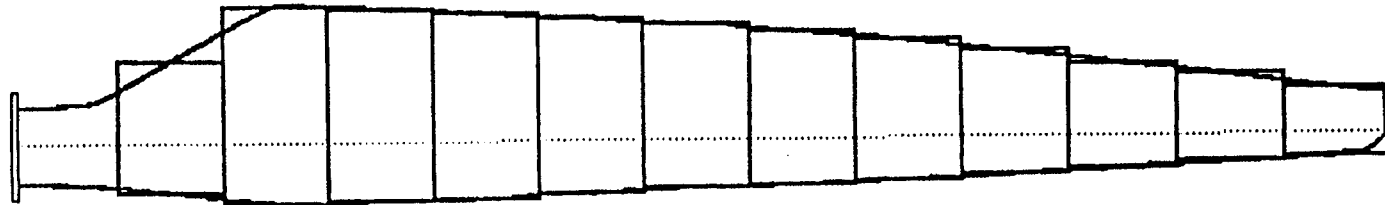
Table 1.2 SERI-8 Blade Planform Geometry Data [2]

(Rotor hub radius = 0.6m, 23.62 in, 1 ft 11.62 in; Twist axis at 30% chord)

Blade Station No.	Blade Radius (in)	Blade Chord c, (in)	Blade Twist B, (deg)	Blade Thickness t, (in)	Rotor Radius r, (in)	Rotor Radius r, (ft)	Rotor Radius r, (m)	%Rotor Radius r/R	Chord/Radius c/R	Airfoil Section/ Notes
36	216	28.90	1.11	4.03	241.00	20.08	6.12	0.72	0.0858	
37	222	28.02	0.91	3.86	247.00	20.58	6.27	0.73	0.0831	
38	228	27.13	0.74	3.70	253.00	21.08	6.43	0.75	0.0805	S805A (@ 232.2")
39	234	26.21	0.59	3.53	259.00	21.58	6.58	0.77	0.0778	
40	240	25.29	0.47	3.35	265.00	22.08	6.73	0.79	0.0750	
41	246	24.34	0.36	3.18	271.00	22.58	6.88	0.80	0.0722	
42	252	23.38	0.27	3.01	277.00	23.08	7.04	0.82	0.0694	Tip slittine at 252"
43	258	22.41	0.20	2.85	283.00	23.58	7.19	0.84	0.0665	
44	264	21.42	0.14	2.68	289.00	24.08	7.34	0.86	0.0636	S805A/6A (@ 263.8")
45	270	20.42	0.09	2.52	295.00	24.58	7.49	0.88	0.0606	
4E	276	19.40	0.06	2.35	301.00	25.08	7.65	0.89	0.0576	
47	282	18.37	0.03	2.19	307.00	25.58	7.80	0.91	0.0545	
48	288	17.32	0.02	2.04	313.00	26.08	7.95	0.93	0.0514	
49	294	16.25	0.01	1.88	319.00	26.58	8.10	0.95	0.0482	S806A (@ 296.0")
50	300	15.19	0.00	1.75	325.00	27.08	8.25	0.96	0.0451	
51	306	14.10	0.00	1.62	331.00	27.58	8.41	0.98	0.0418	
52	312	13.00	0.00	1.50	337.00	28.08	8.56	1.00	0.0386	S806A (@ 312.0")

Table 1.2 (continue) SERI-8 Blade Planform Geometry Data [2]

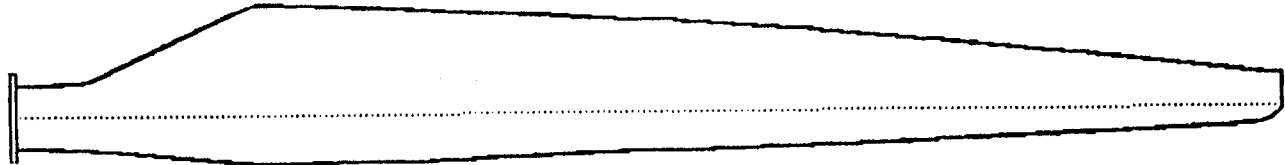
SERI 7.9m Thin Airfoil Blade



Part	1	2	3	4	5	6	7	8	9	10	11	12	13
Station, in	12	36	60	84	108	132	156	180	204	228	252	276	300
Min, m	0.000	0.610	1.219	1.829	2.438	3.048	3.658	4.267	4.877	5.486	6.096	6.706	7.315
Mid, m	0.305	0.914	1.524	2.134	2.743	3.353	3.962	4.572	5.182	5.791	6.401	7.010	7.620
Max, m	0.610	1.219	1.829	2.438	3.048	3.658	4.267	4.877	5.486	6.096	6.706	7.315	7.925
Mass, lbm	184.09	67.05	62.64	49.13	36.38	31.10	28.72	25.75	27.75	38.41	53.85	27.18	10.35
Mass, kg	83.50	30.41	28.41	22.28	16.50	14.11	13.03	11.68	12.59	17.42	24.43	12.33	4.69
Chord, in	17.83	29.43	44.00	43.09	41.42	39.27	36.71	33.81	30.61	27.13	23.38	19.40	15.19
Chord, m	0.453	0.748	1.118	1.094	1.052	0.997	0.932	0.859	0.777	0.689	0.594	0.493	0.386
Thickness, m	0.453	0.369	0.235	0.203	0.183	0.165	0.146	0.129	0.111	0.094	0.076	0.060	0.044
Half Thick, m	0.226	0.185	0.117	0.101	0.091	0.082	0.073	0.064	0.056	0.047	0.038	0.030	0.022
Thick/Chord	1.000	0.494	0.210	0.185	0.174	0.165	0.157	0.150	0.143	0.136	0.129	0.121	0.115
Lead Edge, m	0.226	0.289	0.346	0.331	0.319	0.300	0.269	0.258	0.242	0.208	0.181	0.162	0.123
Trail Edge, m	0.226	0.459	0.772	0.763	0.733	0.697	0.663	0.601	0.535	0.481	0.413	0.331	0.263
Twist, deg	29.85	26.28	20.00	14.81	10.61	7.29	4.74	2.87	1.57	0.74	0.27	0.06	0.00
90-Twist	60.15	63.72	70.00	75.19	79.39	82.71	85.26	87.13	88.43	89.26	89.73	89.94	90.00
Top-TE-Yr	0.084	0.329	0.685	0.712	0.704	0.681	0.655	0.597	0.534	0.480	0.413	0.331	0.263
Top-TE-Zr	-0.309	-0.368	-0.374	-0.293	-0.225	-0.170	-0.128	-0.094	-0.070	-0.053	-0.040	-0.030	-0.022

Table 1.3 SERI-8 Section Structural Properties [2]

SERI 7.9m Thin Airfoil Blade Stiffness Properties



Part	1	2	3	4	5	6	7	8	9	10	11	12	13
Beam mid, in	0	24	48	72	96	120	144	168	192	216	240	264	288
Beam mid, m	-0.305	0.305	0.914	1.524	2.134	2.743	3.353	3.962	4.572	5.182	5.791	6.401	7.010
Area, in ²	312.72	298.72	342.73	296.26	249.88	216.92	182.70	149.64	119.49	91.43	66.51	45.06	27.74
Area, m ²	0.2018	0.1927	0.2211	0.1911	0.1612	0.1399	0.1179	0.0965	0.0771	0.0590	0.0429	0.0291	0.0179
EI-Flap, lbf-in ² x 10 ⁺⁷	137.00	331.00	375.00	114.00	69.30	52.00	42.70	34.60	23.30	10.50	2.17	2.17	2.17
EI-Flap, N-m ² x 10 ⁺⁵	39.32	94.99	107.62	32.72	19.89	14.92	12.25	9.93	6.69	3.01	0.62	0.62	0.62
Half Chord, m	0.226	0.374	0.559	0.547	0.526	0.499	0.466	0.429	0.389	0.345	0.297	0.246	0.193
Half Thick, m	0.226	0.185	0.117	0.101	0.091	0.082	0.073	0.064	0.056	0.047	0.038	0.030	0.022
# Skin Layers	n/a	7.00	10.50	11.50	11.00	7.50	7.00	6.00	5.50	5.00	3.75	3.80	3.60
Skin Thick, m	0.011	0.007	0.011	0.012	0.012	0.008	0.007	0.006	0.006	0.005	0.004	0.004	0.004
Skin X-Area, m ²	0.0154	0.0128	0.0232	0.0243	0.0222	0.0143	0.0124	0.0097	0.0080	0.0064	0.0041	0.0034	0.0025
Ixx-Tors, m ⁴ x 10 ⁻⁶	752.95	972.90	2567.78	2466.36	2032.59	1173.60	875.43	577.20	385.67	239.53	113.24	63.72	28.25
Iyy-Flap, m ⁴ x 10 ⁻⁶	376.47	245.69	193.47	149.79	110.25	59.84	40.79	24.96	15.40	8.76	3.78	1.87	0.74
Izz-Edge, m ⁴ x 10 ⁻⁶	376.47	727.21	2374.31	2316.57	1922.34	1113.76	834.64	552.23	370.27	230.78	109.47	61.85	27.51
E, GPa	10.44	38.66	55.63	21.84	18.04	24.94	30.04	39.78	43.43	34.40	16.49	33.37	84.63
E, psi x 10 ⁺⁶	15.15	56.08	80.68	31.68	26.16	36.17	43.57	57.69	62.99	49.90	23.92	48.40	122.75
G, GPa	12.00	12.00	12.00	12.00	12.00	12.00	12.00	12.00	12.00	12.00	12.00	12.00	12.00
Twist, deg	29.85	26.28	20.00	14.81	10.61	7.29	4.74	2.87	1.57	0.74	0.27	0.06	0.00
90-Twist	60.15	63.72	70.00	75.19	79.39	82.71	85.26	87.13	88.43	89.26	89.73	89.94	90.00
Twist, rad	0.52	0.46	0.35	0.26	0.19	0.13	0.08	0.05	0.03	0.01	0.00	0.00	0.00
Beam Length, m	0.362	0.732	0.610	0.610	0.610	0.610	0.610	0.610	0.610	0.610	0.610	0.610	0.610

Table 1.3 (continued) SERI-8 Section Structural Properties [2]

Figure 1.1 SERI-8 General Dimensions [3]

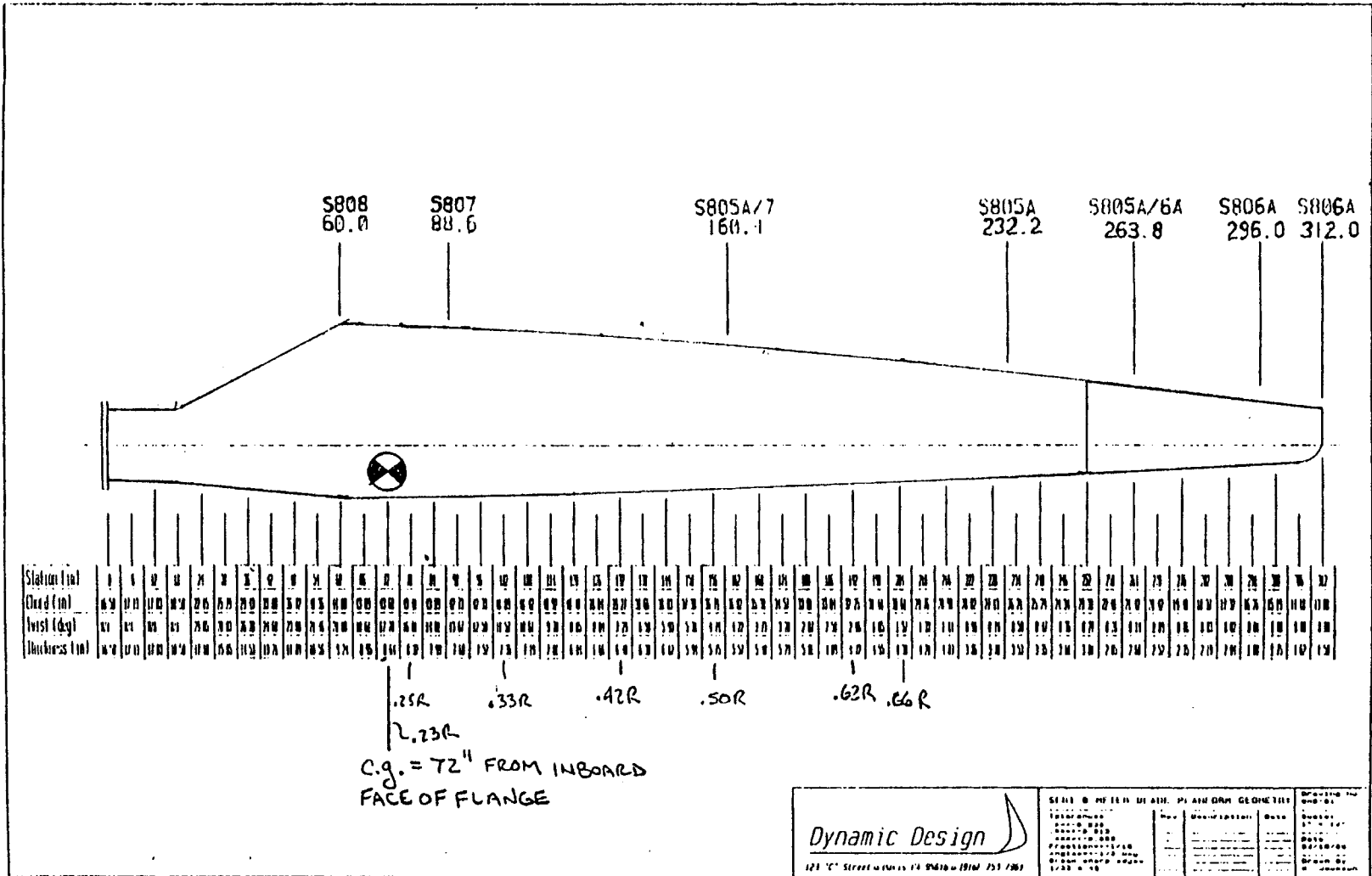
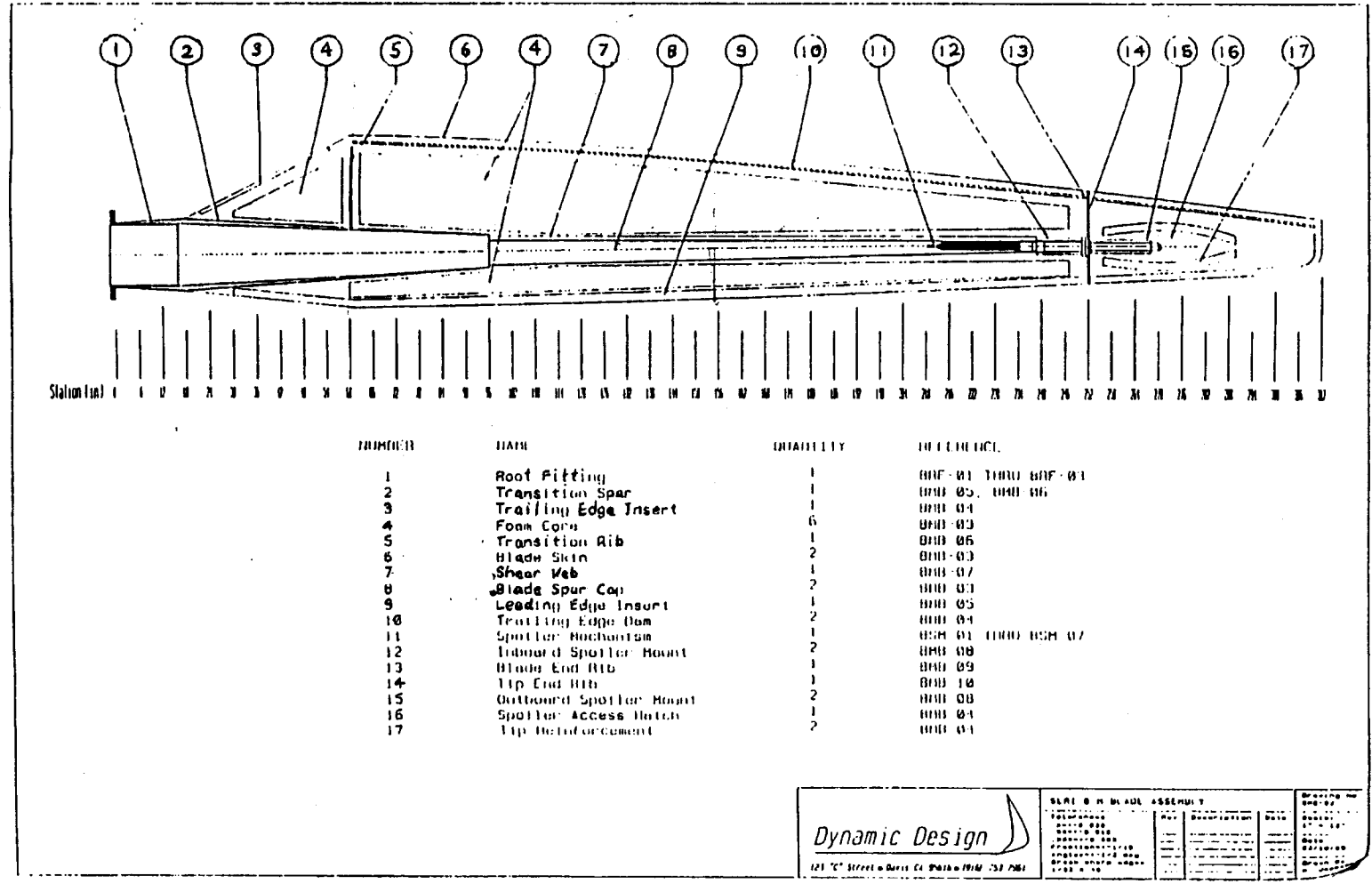


Figure 1.2 SERI-8 Blade Construction [3]



Chapter 2

SERI-8 Blade Baseline Model

The 3D-Beam [5] code was used to create the SERI-8 blade baseline finite element model. This code was also used to study enhanced models: a model with glass fibers being incrementally replaced by carbon fibers and one in which carbon fibers are placed at an off-axis direction for bend-induced-twist design.

In this chapter, we highlight our model's constraints and assumptions, and compare its predictions with data given in Keller and Smith [2], Jackson [3], and Klingenstein [4].

2.1 Modeling Constraints and Assumptions

The modeling constraints and assumptions are as follows:

- a. Because data on lamination, such as schedule and material properties, is incomplete in Keller and Smith [2] and Jackson [3], we made a reasonable guess about both. As indicated in Jackson [3], the materials used were MAT, TRIAX and unidirectional-roving fibers; the material properties of these materials were not given in Jackson [3]. We used the material properties given by TPI [6], which is blade manufacturer. The material properties of three essential materials: MAT, TRIAX and unidirectional fiber, are shown in Table 2.1. We estimated the laminate schedule for the blade skin by matching the geometry and structural properties given in the works cited here.
- b. The original design of the SERI-8 blade had foam core placed at various locations along the skin (see Figure 1.2) to increase local EI properties and to increase buckling strength; however, the foam core has negligible effect on overall EI properties and the baseline model does not include it.

c. The original blade design also had a 96-inch long filament wound transition spar to which a 142-inch shear web was then attached. The skins of the shear webs consisted of 6-ply $\pm 45^\circ$ E-glass material. For the baseline blade model, a spar is assumed to be placed at the blade axis from station 12 to the end of the blade; the skin of the spar is assumed to have four layers of TRIAX material (see Table 2.1 also).

d. The original blade design had two ribs, located at stations 60 and 252, which are a transition rib and a blade/tip end rib, respectively. They were not modeled because of the 3D_Beam's limitations. In addition, the baseline model assumes that the blade structure is continuous; however, the actual blade construction was made up of two components: the main blade and the spoiler. The discontinuity occurred at station 252.

e. The blade model is divided into twelve equal-span 24-inch components, the same division as in Keller and Smith [2]. The cross-section shape and chordwise length of each component are assumed to be the same at the mid-section of that component.

2.2 Modeling Results and Comparisons

Comparisons between the 3D_Beam prediction and the reference data were made in three general areas: geometric properties, structural properties, and results of static tests. The geometric properties being studied are skin cross-section area and area moment of inertia (I_{xx} , I_{yy} & I_{zz}); all these properties do not require material information. The structural properties being studied are EI_{yy} (flapping) and mass distribution along the blade span. Estimations of these properties require both material and geometric properties.

The estimated geometric properties -- skin cross-section area, I_{yy} , I_{zz} and I_{xx} -- are shown in Figures 2.1 to 2.4 respectively. The estimated skin cross-section area and I_{yy} agree very well with the results given in Keller and Smith [2] and Jackson [3]; however, the estimated I_{xx} and I_{zz} are higher than the results reported there.

The estimated structural properties -- EI_{yy} and weight distribution -- are shown in Figures 2.5 and 2.6 respectively. The major differences between estimated and reported

results are near the root and at station 60, where there was a rib that was not modeled by the 3D_Beam code and may be the reason the estimated EI_{yy} is lower than the reported value. It is also unclear whether the reported structural properties considered the effect of the steel root fitting. Inclusion of a steel root fitting in the estimate increases both weight and EI_{yy} near the root area. The weight comparison near the root improves after the effect of a root fitting is included; however, there is a huge EI_{yy} difference near the root.

After having reasonably matched geometric and structural properties, the estimated static deflection and that reported in Klingenstein [4] were compared. The load was applied at the tip (station 312), and the root was clamped. The comparisons also included the estimated results without root fitting, though it is believed the test was done on a complete SERI-8 blade (with root fitting). The estimated results (with root fitting) agree well with the reported test results (see Figure 2.7).

2.3 Laminate Schedule for Baseline SERI-8 Model

The final laminate schedule for the baseline SERI-8 model is shown on Table 2.2. The composition of the lay-up is about 80% unidirectional-roving fibers, 17% TRIAX, and 3% MAT.

Table 2.1 Material Properties Data [6]

E_1 = Longitudinal Modulus (msi)		X_T = Axial Failure Stress - Tension (ksi)	
E_2 = Transverse Modulus (msi)		X_C = Axial Failure Stress - Compression (ksi)	
G_{12} = In-plane Shear Modulus (msi)		Y_T = Transverse Failure Stress - Tension (ksi)	
ν_{12} = Poisson's Ratio		Y_C = Transverse Failure Stress - Compression (ksi)	
t = thickness (10^{-3} in)		S = Shear Failure Stress (ksi)	
Material	DDB340 (TRIAX)	C260 (Unidirectional)	MAT
E_1	3.93	6.14	1.1
E_2	1.64	1.41	1.1
G_{12}	0.94	0.94	0.94
ν_{12}	0.3	0.3	0.3
X_T	88.2	103	19
X_C	53.1	49.8	20
Y_T	13.6	2.3	19
Y_C	15	2.3	20
S	15	3.6	13
t	15	5	5

Table 2.2 Laminate Lay-up for SERI-8 Baseline Model

Station	Laminate Lay-up*	Volume Fraction of C260 Material	Volume Fraction of TRIAX	Volume Fraction of MAT
0 - 24	2 x MAT; 4 x TRIAX; 75 x C260 (90°)	84.3%	13.5%	2.2%
24 - 48	2 x MAT; 4 x TRIAX; 40 x C260 (0°)	74.1%	22.2%	3.7%
48 - 72	2 x MAT; 4 x TRIAX; 60 x C260 (0°)	81.1%	16.2%	2.7%
72 - 96	2 x MAT; 3 x TRIAX; 80 x C260 (0°)	87.9%	9.9%	2.2%
96 - 120	2 x MAT; 3 x TRIAX; 70 x C260 (0°)	86.4%	11.1%	2.5%
120 - 168	2 x MAT; 2 x TRIAX; 55 x C260 (0°)	87.3%	9.5%	3.2%
168 - 192	2 x MAT; 2 x TRIAX; 42 x C260 (0°)	84.0%	12.0%	4.0%
192 - 240	2 x MAT; 2 x TRIAX; 30 x C260 (0°)	78.9%	15.8%	5.3%
240 - 288	2 x MAT; 2 x TRIAX; 25 x C260 (0°)	75.8%	18.2%	6.1%
288 - 312	2 x MAT; 6 x TRIAX	0.0%	90.0%	10.0%
Spar	4 x TRIAX	0.0%	100.0%	0.0%

*: The first number is the number of layers (i.e. 2 x MAT means 2 layers of MAT material).

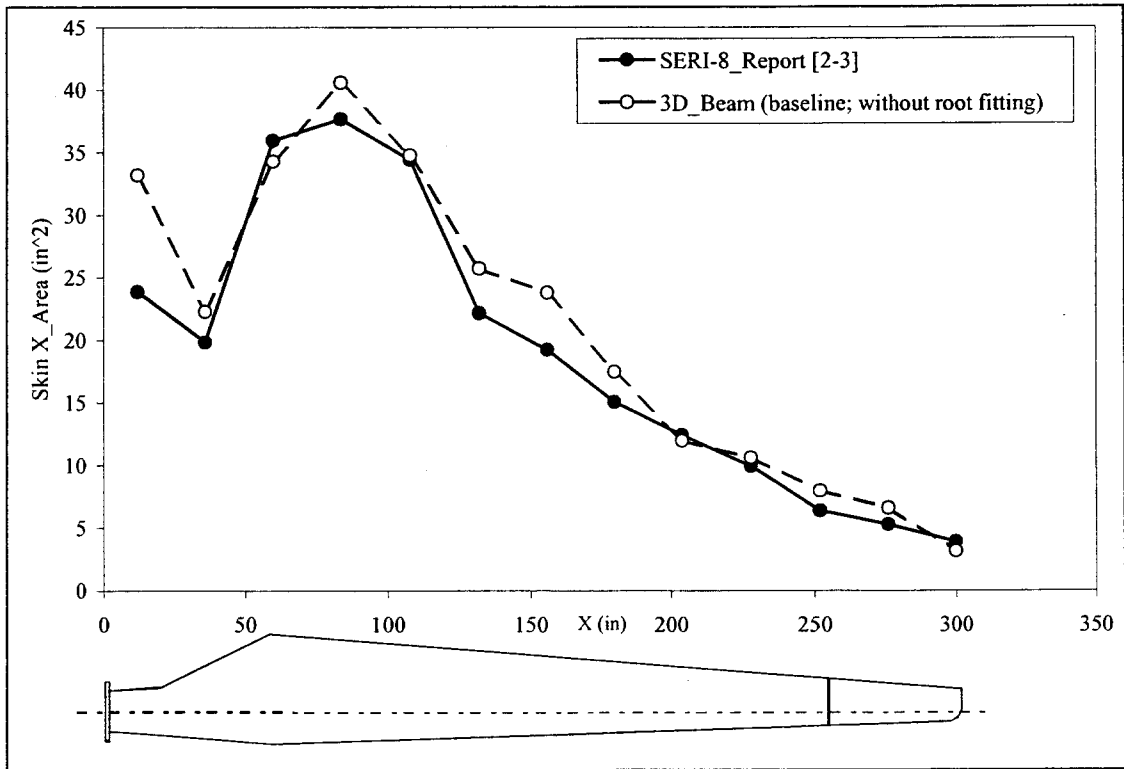


Figure 2.1 Skin Cross-section Area Distribution along a SERI-8 Blade

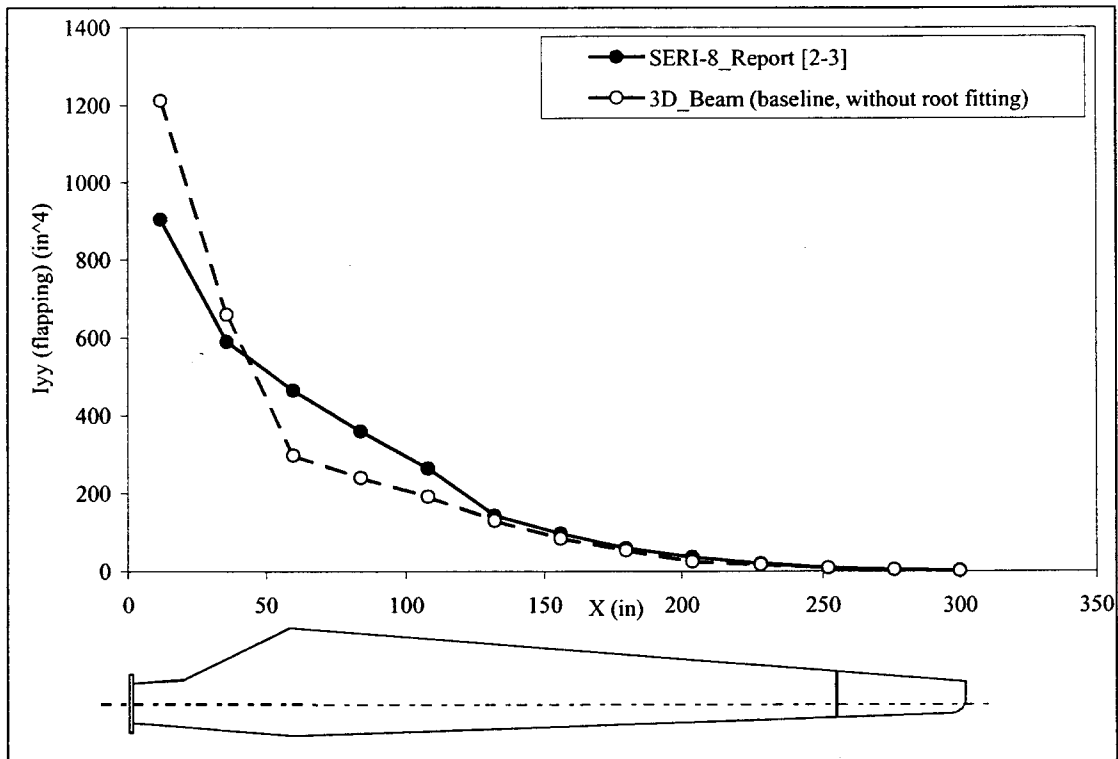


Figure 2.2 Iyy Distribution along a SERI-8 Blade

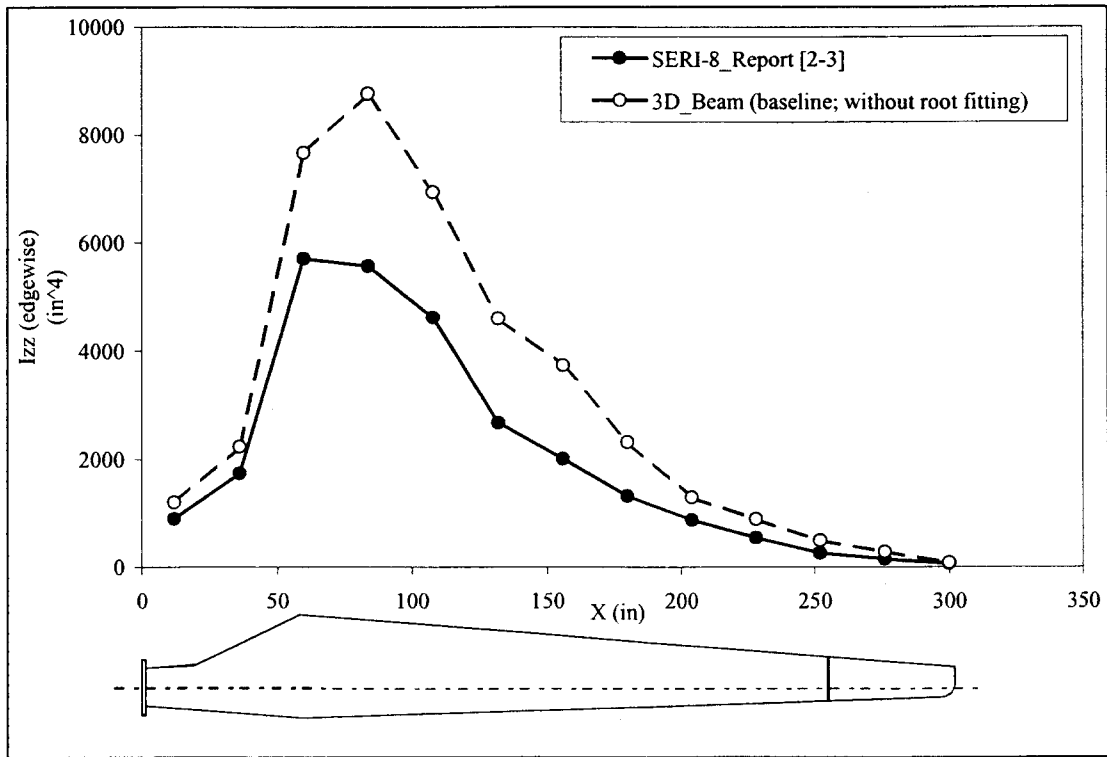


Figure 2.3 I_{zz} Distribution along a SERI-8 Blade

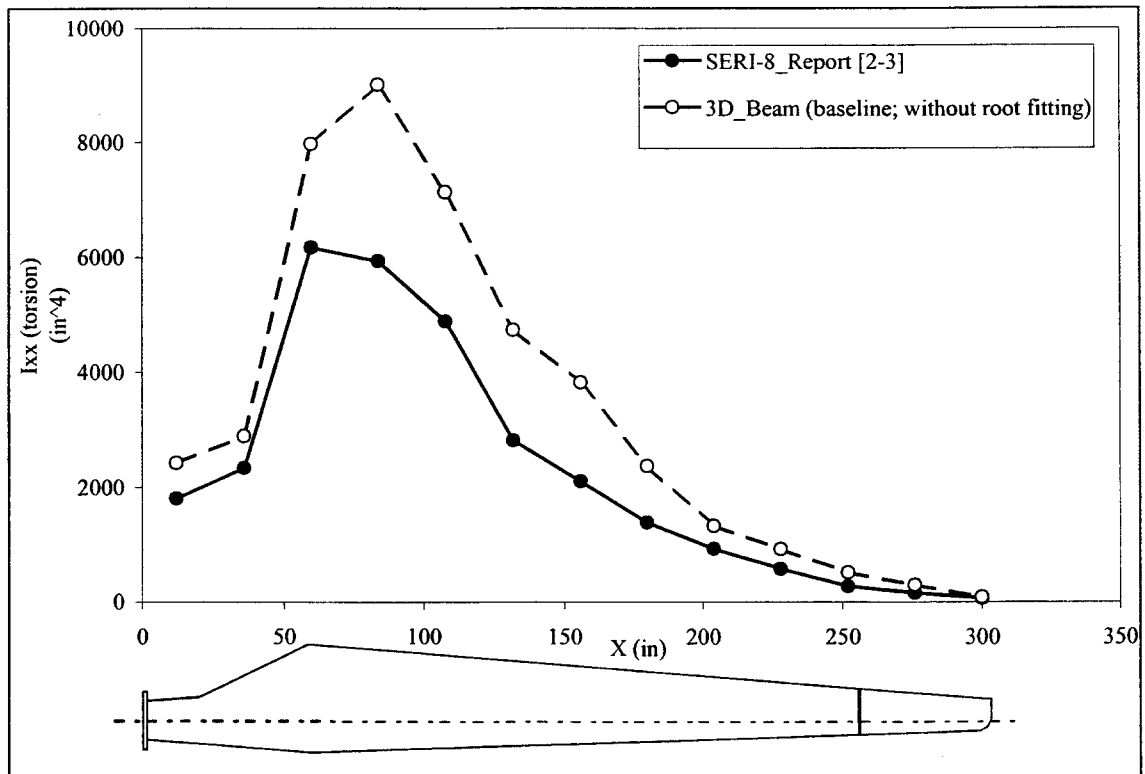


Figure 2.4 I_{xx} Distribution along a SERI-8 Blade

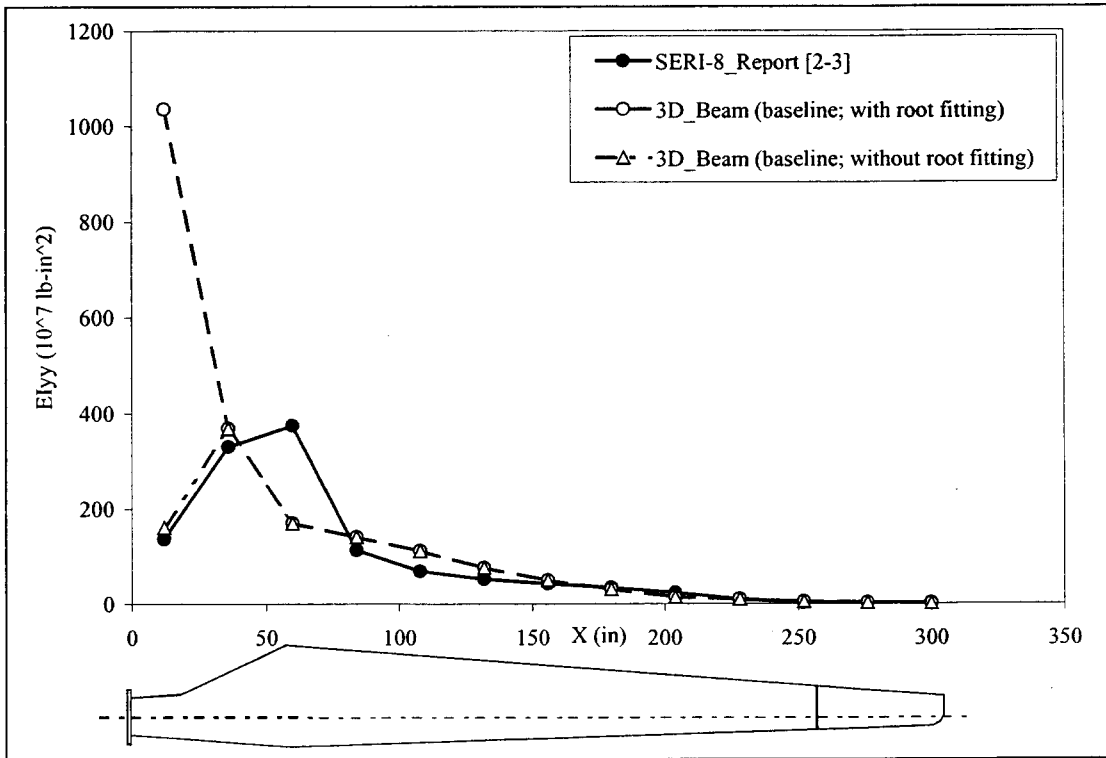


Figure 2.5 EI_{yy} Distribution along a SERI-8 Blade

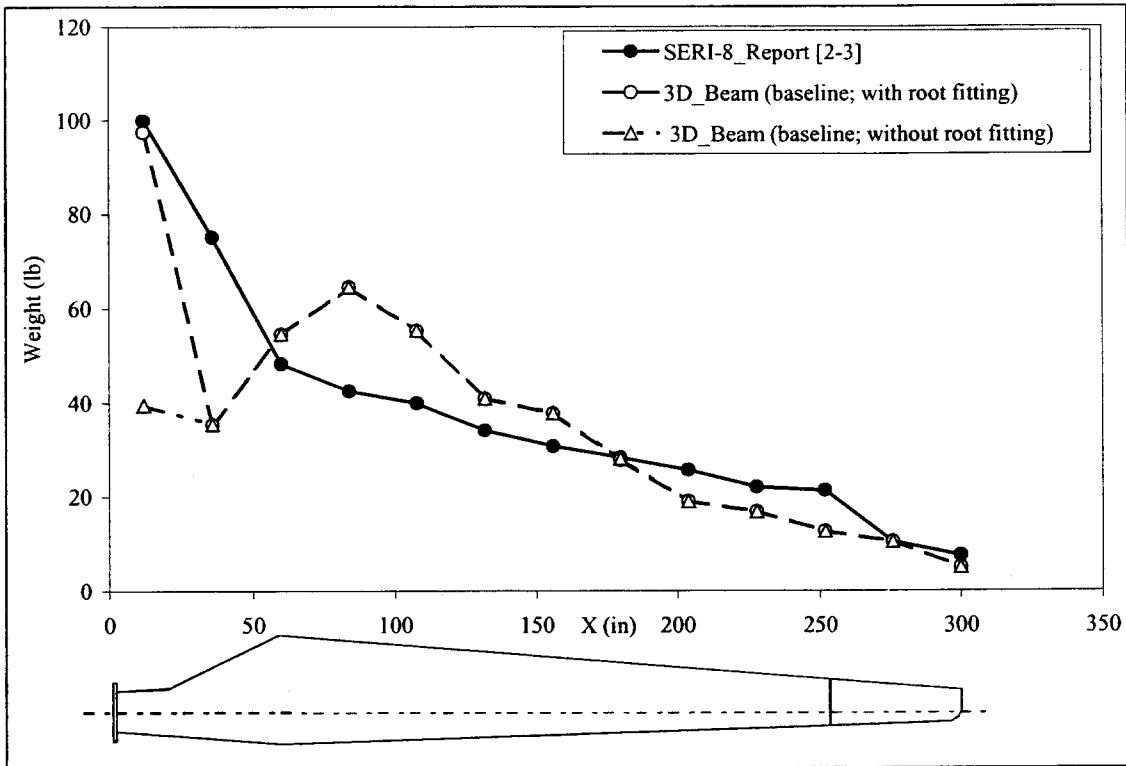


Figure 2.6 Weight Distribution along a SERI-8 Blade

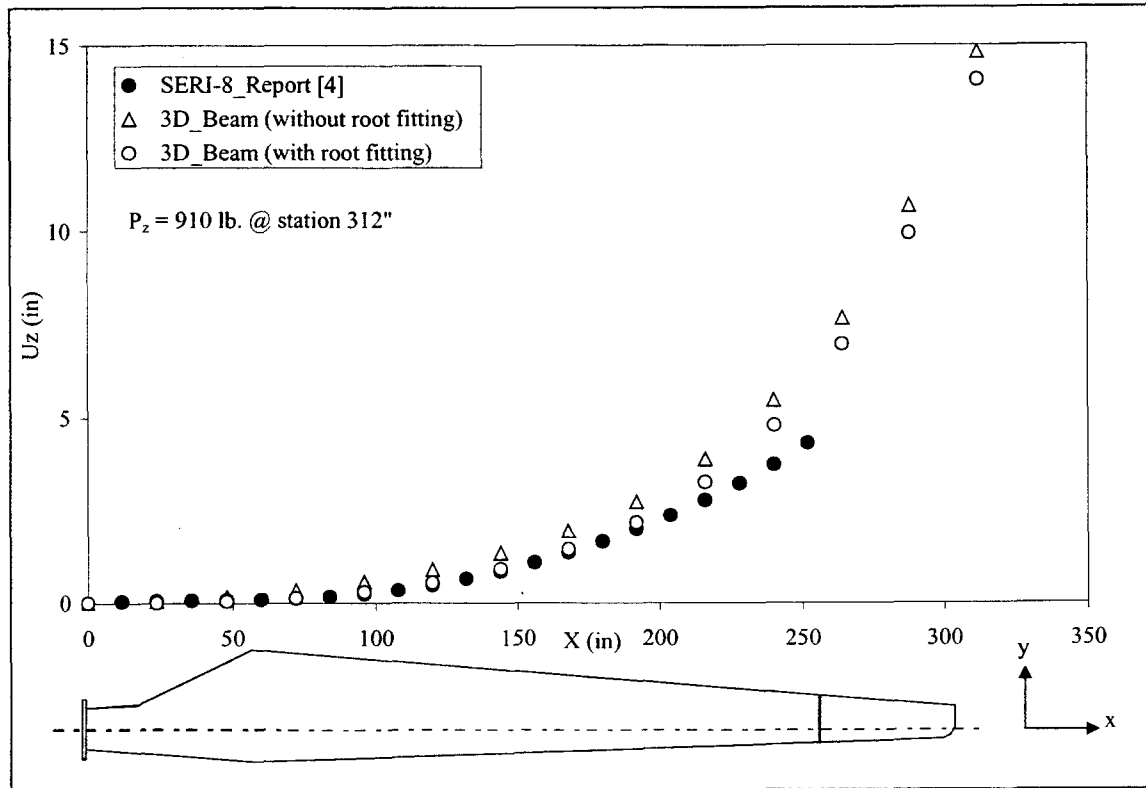


Figure 2.7 Vertical Deflection along the Blade for a Point Load of 910 lbs. applied at Station 312.

Chapter 3

SERI-8 Blade Enhanced Models

The baseline model, which is discussed in the previous chapter, was modified to study the effect of using carbon fibers with and without bend-twist-coupled design. The evaluation parameters not only included geometric and structural properties, but also a cost model that is discussed in a subsequent chapter.

The laminate lay-up for the baseline model is composed of roughly 80% unidirectional glass fibers, 17% TRIAX and 3% MAT (see Table 2.2). The criterion for transforming the baseline model to an enhanced model (with and without bend-twist-coupled design) is to maintain the same flapping stiffness (EI_{yy} , see Figure 3.1). To achieve the design criterion for the enhanced models, it is more efficient to replace unidirectional glass fibers with unidirectional carbon fibers and leave the TRIAX and MAT materials untouched.

3.1 Generation of Enhanced Models

The procedures to generate an enhanced model (with and without bend-twist-coupled design) are as follows:

- a. Only unidirectional glass fibers are replaced.
- b. The blade is divided into 13 equal lengths of 24-inch blade elements. The laminate lay-up for the first (station 0 - 24) and last (station 288 - 312) blade elements remains unchanged. The last blade element only has MAT and TRIAX materials. The first blade element is a part of the transition spar in the real blade design.
- c. An incremental volume fraction (based on the volume of individual blade elements) of unidirectional glass fibers of each blade element is removed from the baseline model.

- d. An equivalent amount of volume of unidirectional carbon fibers (material properties are given in Table 3.1) is substituted for the removed glass fibers. The volume of carbon fibers required for each blade element is determined by maintaining the same EI_{yy} for that blade element.
- e. In bend-twist-coupled design, all unidirectional carbon fibers are orientated from 0 degrees to 20 degrees to achieve higher bend-twist coupling. Additional carbon fibers are added to maintain the same EI_{yy} .

3.2 Loading Conditions for Failure Analysis

In addition, to evaluate the geometric and structural properties, it is also important to know the enhanced model is able to withstand extreme wind conditions. The extreme wind design load generally used for wind turbine blade design is 70 meters/second wind speed, with $C_d = 1.7$.

As the current 3D_Beam model cannot handle a distributed load (such as pressure load), we must transform the distributed load to nodal forces using an energy method. Figure 3.2 shows the nodal force distribution along the blade after the transformation.

3.3 Results & Discussions

Tables 3.2 - 3.11 show the laminate lay-up for various blade configurations, which incrementally replace unidirectional glass fibers with unidirectional carbon fibers. Configurations shown in Tables 3.2 - 3.6 are for blade models designed without bend-twist-coupling. Configurations shown in Tables 3.7 - 3.11 are for blade models designed for bend-twist-coupling.

The main advantage of replacing glass fibers with carbon fibers is reduction in blade weight, as shown in Figure 3.3 - 3.4. A lighter blade design could have many advantages, including ease of handling and transportation, as well as lowering tower head mass. The skin cross-section area (Figure 3.5) and area moments of inertia (Figures 3.6 - 3.8) are also reduced. The same reductions are also seen in blade configurations designed for bend-twist coupling. Additional parameters being evaluated in bend-twist-coupled

configurations were the cross-coupling parameter (α) [7], bend-induced twist distribution and vertical deflection (U_z) under the design wind condition.

Figure 3.9 shows the cross-coupling parameter (α) distribution for various bend-twist-coupled configurations. The cross-coupling parameter is seen to be nearly constant for a bend-twist-coupled configuration, although the volumes of carbon fibers for each blade element change along the blade (see Tables 3.7 -3.11). This observation reinforces one of the findings in a previous project [7] -- the cross-coupling parameter depends on the volume fraction of the anisotropy layers and does not depend on the actual volume of the anisotropy layers.

A bend-twist-coupled blade is more flexible than a normal blade without bend-twist-coupled design, although both blades have the same flapping stiffness property (EI_{yy}). Consequently, a bend-twist-coupled blade will have a higher deflection than a normal blade. Figure 3.10 shows the vertical deflection distribution along SERI-8 blades for various bend-twist-coupled configurations subjected to a wind load of 70 meters/second. The bend-induced twist distribution for the same wind condition for various bend-twist-coupled configurations is shown on Figure 3.11.

One bend-twist-coupled blade configuration was chosen for failure analysis and the selection criterion is based primarily on the highest bend-induced twist angle. The selected configuration was the blade model having 100% unidirectional glass fibers replacement; in addition, failure analysis was carried out for the baseline configuration for comparison's sake.

Since each blade element (total of 13 blade elements) has different geometry and structural properties, the failure may not occur at the root area. Thus, it was necessary to carry out a failure analysis for each blade element for the baseline configuration and the selected bend-twist-coupled (twist-to-feather) configuration shown in Figures 3.12 and 3.13. Failure occurs when the failure index is more than one [5]. Each blade element is modeled by 28 brick-elements for the skin and 1 brick-element for the spar. Except blade element number '1' (between station 0 and 24), the brick-element number for the spar (the spar begins from station 24) is designated as number '29'. Both figures indicate that the likely failure location for both configurations (baseline and the selected bend-twist-coupled configurations) is near the root area. In addition, the factor of safety, which is in

inverse proportion to the failure index, is about two for the two configurations under the same severe wind condition.

Figures 3.14 - 3.26 show the estimated failure index (top diagram), and the shape and brick-element numbering (bottom diagram) in each blade element for the baseline configuration as well as both twist-to-stall and twist-to-feather coupled designs. The highest overall failure index (most likely to fail) occurs in the baseline design at element 9. The failure index is 0.50 resulting in a design factor of safety of 2.0. The maximum failure index for the twist-to-stall design is 0.32 at element 2, and for the twist-to-feather design is 0.41 at element 6. In every case except one, the twist-to-stall maximum failure index is lower than the baseline; the exception is at element 2 (see Figure 3.15). The twist to feather failure index is higher than the baseline for elements 2-7 (see Figures 3.15 - 3.20), and lower for elements 8-12 (see Figures 3.21 - 3.25). The failure indices for most of the "Brick" elements all along the blade are higher for the bend-twist-coupled blades than that of the baseline configuration, although well below the local maximums.

The bend-twist-coupled designs also shift the location of maximum failure index at each section. The highest index occurs at the leading edge brick-element for the baseline and twist-to-feather configurations, and is usually at the middle brick-element for the twist-to-stall configuration.

The failure prediction assumes that the joint between the two halves of the bend-twist blade is seamless. In fact, there is a discontinuity of the 20-degree fibers at the seam. The failure index similarly treats all "Brick" elements as equally strong and does not recognize the weak connection at the joint. To strengthen the joint at the seam, it is suggested that a staggered overlap joining method [7] or some other joint reinforcement technique be adopted.

Table 3.1 Material Properties Data for CFRP AS/H3501

E_x = Longitudinal Modulus (msi)	X_T = Axial Failure Stress - Tension (ksi)
E_y = Transverse Modulus (msi)	X_C = Axial Failure Stress - Compression (ksi)
G_{xy} = In-plane Shear Modulus (msi)	Y_T = Transverse Failure Stress - Tension (ksi)
ν_x = Poisson's Ratio	Y_C = Transverse Failure Stress - Compression (ksi)
t = thickness (10^{-3} in)	S = Shear Failure Stress (ksi)

Material	CFRP AS/H3501
E_x	20.0
E_y	1.3
G_{xy}	1.03
ν_x	0.3
X_T	209.8
X_C	209.8
Y_T	7.5
Y_C	29.9
S	13.5
t	4.925

Table 3.2 Laminate Lay-up for SERI-8 Enhanced Model (without bend-twist-coupling) with 20% Unidirectional Glass Replacement

Station	Laminate Lay-up*	Volume Fraction of AS/H3501	Volume Fraction of C260 Material	Volume Fraction of TRIAX	Volume Fraction of MAT
0 - 24	2 x MAT; 4 x TRIAX; 75 x C260 (90°)	0.0%	84.3%	13.5%	2.2%
24 - 48	2 x MAT; 4 x TRIAX; 32 x C260 (0°); 2 x AS/H3501 (0°)	4.2%	66.7%	25.0%	4.2%
48 - 72	2 x MAT; 4 x TRIAX; 48 x C260 (0°); 3 x AS/H3501 (0°)	4.6%	73.8%	18.5%	3.1%
72 - 96	2 x MAT; 3 x TRIAX; 64 x C260 (0°); 4 x AS/H3501 (0°)	5.1%	81.0%	11.4%	2.5%
96 - 120	2 x MAT; 3 x TRIAX; 56 x C260 (0°); 3 x AS/H3501 (0°)	4.3%	80.0%	12.9%	2.9%
120 - 144	2 x MAT; 2 x TRIAX; 44 x C260 (0°); 3 x AS/H3501 (0°)	5.5%	80.0%	10.9%	3.6%
144 - 168	2 x MAT; 2 x TRIAX; 44 x C260 (0°); 3 x AS/H3501 (0°)	5.5%	80.0%	10.9%	3.6%
168 - 192	2 x MAT; 2 x TRIAX; 34 x C260 (0°); 2 x AS/H3501 (0°)	4.5%	77.3%	13.6%	4.5%
192 - 216	2 x MAT; 2 x TRIAX; 24 x C260 (0°); 2 x AS/H3501 (0°)	5.9%	70.6%	17.6%	5.9%
216 - 240	2 x MAT; 2 x TRIAX; 24 x C260 (0°); 2 x AS/H3501 (0°)	5.9%	70.6%	17.6%	5.9%
240 - 264	2 x MAT; 2 x TRIAX; 20 x C260 (0°); 1 x AS/H3501 (0°)	3.4%	69.0%	20.7%	6.9%
264 - 288	2 x MAT; 2 x TRIAX; 20 x C260 (0°); 1 x AS/H3501 (0°)	3.4%	69.0%	20.7%	6.9%
288 - 312	2 x MAT; 6 x TRIAX	0.0%	0.0%	90.0%	10.0%
Spar	4 x TRIAX	0.0%	0.0%	90.0%	10.0%

*: The first number is the number of layers (i.e. 2 x MAT means 2 layers of MAT material).

Table 3.3 Laminate Lay-up for SERI-8 Enhanced Model (without bend-twist-coupling) with 40% Unidirectional Glass Replacement

Station	Laminate Lay-up*	Volume Fraction of AS/H3501	Volume Fraction of C260 Material	Volume Fraction of TRIAX	Volume Fraction of MAT
0 - 24	2 x MAT; 4 x TRIAX; 75 x C260 (90°)	0.0%	84.3%	13.5%	2.2%
24 - 48	2 x MAT; 4 x TRIAX; 24 x C260 (0°); 5 x AS/H3501 (0°)	11.6%	55.8%	27.9%	4.7%
48 - 72	2 x MAT; 4 x TRIAX; 36 x C260 (0°); 5 x AS/H3501 (0°)	9.1%	65.5%	21.8%	3.6%
72 - 96	2 x MAT; 3 x TRIAX; 48 x C260 (0°); 8 x AS/H3501 (0°)	11.9%	71.6%	13.4%	3.0%
96 - 120	2 x MAT; 3 x TRIAX; 42 x C260 (0°); 7 x AS/H3501 (0°)	11.7%	70.0%	15.0%	3.3%
120 - 144	2 x MAT; 2 x TRIAX; 33 x C260 (0°); 7 x AS/H3501 (0°)	14.6%	68.8%	12.5%	4.2%
144 - 168	2 x MAT; 2 x TRIAX; 33 x C260 (0°); 7 x AS/H3501 (0°)	14.6%	68.8%	12.5%	4.2%
168 - 192	2 x MAT; 2 x TRIAX; 25 x C260 (0°); 5 x AS/H3501 (0°)	13.2%	65.8%	15.8%	5.3%
192 - 216	2 x MAT; 2 x TRIAX; 18 x C260 (0°); 4 x AS/H3501 (0°)	13.3%	60.0%	20.0%	6.7%
216 - 240	2 x MAT; 2 x TRIAX; 18 x C260 (0°); 4 x AS/H3501 (0°)	13.3%	60.0%	20.0%	6.7%
240 - 264	2 x MAT; 2 x TRIAX; 15 x C260 (0°); 4 x AS/H3501 (0°)	14.8%	55.6%	22.2%	7.4%
264 - 288	2 x MAT; 2 x TRIAX; 15 x C260 (0°); 4 x AS/H3501 (0°)	14.8%	55.6%	22.2%	7.4%
288 - 312	2 x MAT; 6 x TRIAX	0.0%	0.0%	90.0%	10.0%
Spar	4 x TRIAX	0.0%	0.0%	90.0%	10.0%

*: The first number is the number of layers (i.e. 2 x MAT means 2 layers of MAT material).

Table 3.4 Laminate Lay-up for SERI-8 Enhanced Model (without bend-twist-coupling) with 60% Unidirectional Glass Replacement

Station	Laminate Lay-up*	Volume Fraction of AS/H3501	Volume Fraction of C260 Material	Volume Fraction of TRIAX	Volume Fraction of MAT
0 - 24	2 x MAT; 4 x TRIAX; 75 x C260 (90°)	0.0%	84.3%	13.5%	2.2%
24 - 48	2 x MAT; 4 x TRIAX; 16 x C260 (0°); 8 x AS/H3501 (0°)	21.1%	42.1%	31.6%	5.3%
48 - 72	2 x MAT; 4 x TRIAX; 24 x C260 (0°); 12 x AS/H3501 (0°)	24.0%	48.0%	24.0%	4.0%
72 - 96	2 x MAT; 3 x TRIAX; 32 x C260 (0°); 16 x AS/H3501 (0°)	27.1%	54.2%	15.3%	3.4%
96 - 120	2 x MAT; 3 x TRIAX; 28 x C260 (0°); 14 x AS/H3501 (0°)	26.4%	52.8%	17.0%	3.8%
120 - 144	2 x MAT; 2 x TRIAX; 22 x C260 (0°); 11 x AS/H3501 (0°)	26.8%	53.7%	14.6%	4.9%
144 - 168	2 x MAT; 2 x TRIAX; 22 x C260 (0°); 11 x AS/H3501 (0°)	26.8%	53.7%	14.6%	4.9%
168 - 192	2 x MAT; 2 x TRIAX; 17 x C260 (0°); 8 x AS/H3501 (0°)	24.2%	51.5%	18.2%	6.1%
192 - 216	2 x MAT; 2 x TRIAX; 12 x C260 (0°); 6 x AS/H3501 (0°)	23.1%	46.2%	23.1%	7.7%
216 - 240	2 x MAT; 2 x TRIAX; 12 x C260 (0°); 6 x AS/H3501 (0°)	23.1%	46.2%	23.1%	7.7%
240 - 264	2 x MAT; 2 x TRIAX; 10 x C260 (0°); 5 x AS/H3501 (0°)	21.7%	43.5%	26.1%	8.7%
264 - 288	2 x MAT; 2 x TRIAX; 10 x C260 (0°); 5 x AS/H3501 (0°)	21.7%	43.5%	26.1%	8.7%
288 - 312	2 x MAT; 6 x TRIAX	0.0%	0.0%	90.0%	10.0%
Spar	4 x TRIAX	0.0%	0.0%	90.0%	10.0%

*: The first number is the number of layers (i.e. 2 x MAT means 2 layers of MAT material).

Table 3.5 Laminate Lay-up for SERI-8 Enhanced Model (without bend-twist-coupling) with 80% Unidirectional Glass Replacement

Station	Laminate Lay-up*	Volume Fraction of AS/H3501	Volume Fraction of C260 Material	Volume Fraction of TRIAX	Volume Fraction of MAT
0 - 24	2 x MAT; 4 x TRIAX; 75 x C260 (90°)	0.0%	84.3%	13.5%	2.2%
24 - 48	2 x MAT; 4 x TRIAX; 8 x C260 (0°); 10 x AS/H3501 (0°)	31.3%	25.0%	37.5%	6.3%
48 - 72	2 x MAT; 4 x TRIAX; 12 x C260 (0°); 15 x AS/H3501 (0°)	36.6%	29.3%	29.3%	4.9%
72 - 96	2 x MAT; 3 x TRIAX; 16 x C260 (0°); 20 x AS/H3501 (0°)	42.6%	34.0%	19.1%	4.3%
96 - 120	2 x MAT; 3 x TRIAX; 14 x C260 (0°); 18 x AS/H3501 (0°)	41.9%	32.6%	20.9%	4.7%
120 - 144	2 x MAT; 2 x TRIAX; 11 x C260 (0°); 14 x AS/H3501 (0°)	42.4%	33.3%	18.2%	6.1%
144 - 168	2 x MAT; 2 x TRIAX; 11 x C260 (0°); 14 x AS/H3501 (0°)	42.4%	33.3%	18.2%	6.1%
168 - 192	2 x MAT; 2 x TRIAX; 8 x C260 (0°); 10 x AS/H3501 (0°)	38.5%	30.8%	23.1%	7.7%
192 - 216	2 x MAT; 2 x TRIAX; 6 x C260 (0°); 7 x AS/H3501 (0°)	33.3%	28.6%	28.6%	9.5%
216 - 240	2 x MAT; 2 x TRIAX; 6 x C260 (0°); 7 x AS/H3501 (0°)	33.3%	28.6%	28.6%	9.5%
240 - 264	2 x MAT; 2 x TRIAX; 5 x C260 (0°); 6 x AS/H3501 (0°)	31.6%	26.3%	31.6%	10.5%
264 - 288	2 x MAT; 2 x TRIAX; 5 x C260 (0°); 6 x AS/H3501 (0°)	31.6%	26.3%	31.6%	10.5%
288 - 312	2 x MAT; 6 x TRIAX	0.0%	0.0%	90.0%	10.0%
Spar	4 x TRIAX	0.0%	0.0%	90.0%	10.0%

*: The first number is the number of layers (i.e. 2 x MAT means 2 layers of MAT material).

Table 3.6 Laminate Lay-up for SERI-8 Enhanced Model (without bend-twist-coupling) with 100% Unidirectional Glass Replacement

Station	Laminate Lay-up*	Volume Fraction of AS/H3501	Volume Fraction of C260 Material	Volume Fraction of TRIAX	Volume Fraction of MAT
0 - 24	2 x MAT; 4 x TRIAX; 75 x C260 (90°)	0.0%	84.3%	13.5%	2.2%
24 - 48	2 x MAT; 4 x TRIAX; 12 x AS/H3501 (0°)	46.2%	0.0%	46.2%	7.7%
48 - 72	2 x MAT; 4 x TRIAX; 19 x AS/H3501 (0°)	57.6%	0.0%	36.4%	6.1%
72 - 96	2 x MAT; 3 x TRIAX; 25 x AS/H3501 (0°)	69.4%	0.0%	25.0%	5.6%
96 - 120	2 x MAT; 3 x TRIAX; 22 x AS/H3501 (0°)	66.7%	0.0%	27.3%	6.1%
120 - 144	2 x MAT; 2 x TRIAX; 17 x AS/H3501 (0°)	68.0%	0.0%	24.0%	8.0%
144 - 168	2 x MAT; 2 x TRIAX; 17 x AS/H3501 (0°)	68.0%	0.0%	24.0%	8.0%
168 - 192	2 x MAT; 2 x TRIAX; 13 x AS/H3501 (0°)	61.9%	0.0%	28.6%	9.5%
192 - 216	2 x MAT; 2 x TRIAX; 9 x AS/H3501 (0°)	52.9%	0.0%	35.3%	11.8%
216 - 240	2 x MAT; 2 x TRIAX; 9 x AS/H3501 (0°)	52.9%	0.0%	35.3%	11.8%
240 - 264	2 x MAT; 2 x TRIAX; 7 x AS/H3501 (0°)	46.7%	0.0%	40.0%	13.3%
264 - 288	2 x MAT; 2 x TRIAX; 7 x AS/H3501 (0°)	46.7%	0.0%	40.0%	13.3%
288 - 312	2 x MAT; 6 x TRIAX	0.0%	0.0%	90.0%	10.0%
Spar	4 x TRIAX	0.0%	0.0%	90.0%	10.0%

*: The first number is the number of layers (i.e. 2 x MAT means 2 layers of MAT material).

Table 3.7 Laminate Lay-up for SERI-8 Enhanced Model (with bend-twist-coupling) with 20% Unidirectional Glass Replacement

Station	Laminate Lay-up*	Volume Fraction of AS/H3501	Volume Fraction of C260 Material	Volume Fraction of TRIAX	Volume Fraction of MAT
0 - 24	2 x MAT; 4 x TRIAX; 75 x C260 (90°)	0.0%	84.3%	13.5%	2.2%
24 - 48	2 x MAT; 4 x TRIAX; 32 x C260 (0°); 4 x AS/H3501 (20°)	8.0%	64.0%	24.0%	4.0%
48 - 72	2 x MAT; 4 x TRIAX; 48 x C260 (0°); 5 x AS/H3501 (20°)	7.5%	71.6%	17.9%	3.0%
72 - 96	2 x MAT; 3 x TRIAX; 64 x C260 (0°); 6 x AS/H3501 (20°)	7.4%	79.0%	11.1%	2.5%
96 - 120	2 x MAT; 3 x TRIAX; 56 x C260 (0°); 5 x AS/H3501 (20°)	6.9%	77.8%	12.5%	2.8%
120 - 144	2 x MAT; 2 x TRIAX; 44 x C260 (0°); 5 x AS/H3501 (20°)	8.8%	77.2%	10.5%	3.5%
144 - 168	2 x MAT; 2 x TRIAX; 44 x C260 (0°); 5 x AS/H3501 (20°)	8.8%	77.2%	10.5%	3.5%
168 - 192	2 x MAT; 2 x TRIAX; 34 x C260 (0°); 4 x AS/H3501 (20°)	8.7%	73.9%	13.0%	4.3%
192 - 216	2 x MAT; 2 x TRIAX; 24 x C260 (0°); 3 x AS/H3501 (20°)	8.6%	68.6%	17.1%	5.7%
216 - 240	2 x MAT; 2 x TRIAX; 24 x C260 (0°); 3 x AS/H3501 (20°)	8.6%	68.6%	17.1%	5.7%
240 - 264	2 x MAT; 2 x TRIAX; 20 x C260 (0°); 2 x AS/H3501 (20°)	6.7%	66.7%	20.0%	6.7%
264 - 288	2 x MAT; 2 x TRIAX; 20 x C260 (0°); 2 x AS/H3501 (20°)	6.7%	66.7%	20.0%	6.7%
288 - 312	2 x MAT; 6 x TRIAX	0.0%	0.0%	90.0%	10.0%
Spar	4 x TRIAX	0.0%	0.0%	90.0%	10.0%

*: The first number is the number of layers (i.e. 2 x MAT means 2 layers of MAT material).

Table 3.8 Laminate Lay-up for SERI-8 Enhanced Model (with bend-twist-coupling) with 40% Unidirectional Glass Replacement

Station	Laminate Lay-up*	Volume Fraction of AS/H3501	Volume Fraction of C260 Material	Volume Fraction of TRIAX	Volume Fraction of MAT
0 - 24	2 x MAT; 4 x TRIAX; 75 x C260 (90°)	0.0%	84.3%	13.5%	2.2%
24 - 48	2 x MAT; 4 x TRIAX; 24 x C260 (0°); 7 x AS/H3501 (20°)	15.6%	53.3%	26.7%	4.4%
48 - 72	2 x MAT; 4 x TRIAX; 36 x C260 (0°); 10 x AS/H3501 (20°)	16.7%	60.0%	20.0%	3.3%
72 - 96	2 x MAT; 3 x TRIAX; 48 x C260 (0°); 13 x AS/H3501 (20°)	18.1%	66.7%	12.5%	2.8%
96 - 120	2 x MAT; 3 x TRIAX; 42 x C260 (0°); 12 x AS/H3501 (20°)	18.5%	64.6%	13.8%	3.1%
120 - 144	2 x MAT; 2 x TRIAX; 33 x C260 (0°); 9 x AS/H3501 (20°)	18.0%	66.0%	12.0%	4.0%
144 - 168	2 x MAT; 2 x TRIAX; 33 x C260 (0°); 9 x AS/H3501 (20°)	18.0%	66.0%	12.0%	4.0%
168 - 192	2 x MAT; 2 x TRIAX; 25 x C260 (0°); 7 x AS/H3501 (20°)	17.5%	62.5%	15.0%	5.0%
192 - 216	2 x MAT; 2 x TRIAX; 18 x C260 (0°); 5 x AS/H3501 (20°)	16.1%	58.1%	19.4%	6.5%
216 - 240	2 x MAT; 2 x TRIAX; 18 x C260 (0°); 5 x AS/H3501 (20°)	16.1%	58.1%	19.4%	6.5%
240 - 264	2 x MAT; 2 x TRIAX; 15 x C260 (0°); 5 x AS/H3501 (20°)	17.9%	53.6%	21.4%	7.1%
264 - 288	2 x MAT; 2 x TRIAX; 15 x C260 (0°); 5 x AS/H3501 (20°)	17.9%	53.6%	21.4%	7.1%
288 - 312	2 x MAT; 6 x TRIAX	0.0%	0.0%	90.0%	10.0%
Spar	4 x TRIAX	0.0%	0.0%	90.0%	10.0%

*: The first number is the number of layers (i.e. 2 x MAT means 2 layers of MAT material).

Table 3.9 Laminate Lay-up for SERI-8 Enhanced Model (with bend-twist-coupling)
with 60% Unidirectional Glass Replacement

Station	Laminate Lay-up*	Volume Fraction of AS/H3501	Volume Fraction of C260 Material	Volume Fraction of TRIAX	Volume Fraction of MAT
0 - 24	2 x MAT; 4 x TRIAX; 75 x C260 (90°)	0.0%	84.3%	13.5%	2.2%
24 - 48	2 x MAT; 4 x TRIAX; 16 x C260 (0°); 11 x AS/H3501 (20°)	26.8%	39.0%	29.3%	4.9%
48 - 72	2 x MAT; 4 x TRIAX; 24 x C260 (0°); 20 x AS/H3501 (20°)	34.5%	41.4%	20.7%	3.4%
72 - 96	2 x MAT; 3 x TRIAX; 32 x C260 (0°); 20 x AS/H3501 (20°)	31.7%	50.8%	14.3%	3.2%
96 - 120	2 x MAT; 3 x TRIAX; 28 x C260 (0°); 18 x AS/H3501 (20°)	31.6%	49.1%	15.8%	3.5%
120 - 144	2 x MAT; 2 x TRIAX; 22 x C260 (0°); 14 x AS/H3501 (20°)	31.8%	50.0%	13.6%	4.5%
144 - 168	2 x MAT; 2 x TRIAX; 22 x C260 (0°); 14 x AS/H3501 (20°)	31.8%	50.0%	13.6%	4.5%
168 - 192	2 x MAT; 2 x TRIAX; 17 x C260 (0°); 11 x AS/H3501 (20°)	30.6%	47.2%	16.7%	5.6%
192 - 216	2 x MAT; 2 x TRIAX; 12 x C260 (0°); 8 x AS/H3501 (20°)	28.6%	42.9%	21.4%	7.1%
216 - 240	2 x MAT; 2 x TRIAX; 12 x C260 (0°); 8 x AS/H3501 (20°)	28.6%	42.9%	21.4%	7.1%
240 - 264	2 x MAT; 2 x TRIAX; 10 x C260 (0°); 7 x AS/H3501 (20°)	28.0%	40.0%	24.0%	8.0%
264 - 288	2 x MAT; 2 x TRIAX; 10 x C260 (0°); 7 x AS/H3501 (20°)	28.0%	40.0%	24.0%	8.0%
288 - 312	2 x MAT; 6 x TRIAX	0.0%	0.0%	90.0%	10.0%
Spar	4 x TRIAX	0.0%	0.0%	90.0%	10.0%

*: The first number is the number of layers (i.e. 2 x MAT means 2 layers of MAT material).

Table 3.10 Laminate Lay-up for SERI-8 Enhanced Model (with bend-twist-coupling) with 80% Unidirectional Glass Replacement

Station	Laminate Lay-up*	Volume Fraction of AS/H3501	Volume Fraction of C260 Material	Volume Fraction of TRIAX	Volume Fraction of MAT
0 - 24	2 x MAT; 4 x TRIAX; 75 x C260 (90°)	0.0%	84.3%	13.5%	2.2%
24 - 48	2 x MAT; 4 x TRIAX; 8 x C260 (0°); 14 x AS/H3501 (20°)	38.9%	22.2%	33.3%	5.6%
48 - 72	2 x MAT; 4 x TRIAX; 12 x C260 (0°); 20 x AS/H3501 (20°)	43.5%	26.1%	26.1%	4.3%
72 - 96	2 x MAT; 3 x TRIAX; 16 x C260 (0°); 26 x AS/H3501 (20°)	49.1%	30.2%	17.0%	3.8%
96 - 120	2 x MAT; 3 x TRIAX; 14 x C260 (0°); 23 x AS/H3501 (20°)	47.9%	29.2%	18.8%	4.2%
120 - 144	2 x MAT; 2 x TRIAX; 11 x C260 (0°); 19 x AS/H3501 (20°)	50.0%	28.9%	15.8%	5.3%
144 - 168	2 x MAT; 2 x TRIAX; 11 x C260 (0°); 18 x AS/H3501 (20°)	48.6%	29.7%	16.2%	5.4%
168 - 192	2 x MAT; 2 x TRIAX; 8 x C260 (0°); 14 x AS/H3501 (20°)	46.7%	26.7%	20.0%	6.7%
192 - 216	2 x MAT; 2 x TRIAX; 6 x C260 (0°); 11 x AS/H3501 (20°)	44.0%	24.0%	24.0%	8.0%
216 - 240	2 x MAT; 2 x TRIAX; 6 x C260 (0°); 10 x AS/H3501 (20°)	41.7%	25.0%	25.0%	8.3%
240 - 264	2 x MAT; 2 x TRIAX; 5 x C260 (0°); 9 x AS/H3501 (20°)	40.9%	22.7%	27.3%	9.1%
264 - 288	2 x MAT; 2 x TRIAX; 5 x C260 (0°); 8 x AS/H3501 (20°)	38.1%	23.8%	28.6%	9.5%
288 - 312	2 x MAT; 6 x TRIAX	0.0%	0.0%	90.0%	10.0%
Spar	4 x TRIAX	0.0%	0.0%	90.0%	10.0%

*: The first number is the number of layers (i.e. 2 x MAT means 2 layers of MAT material).

Table 3.11 Laminate Lay-up for SERI-8 Enhanced Model (with bend-twist-coupling) with 100% Unidirectional Glass Replacement

Station	Laminate Lay-up*	Volume Fraction of AS/H3501	Volume Fraction of C260 Material	Volume Fraction of TRIAX	Volume Fraction of MAT
0 - 24	2 x MAT; 4 x TRIAX; 75 x C260 (90°)	0.0%	84.3%	13.5%	2.2%
24 - 48	2 x MAT; 4 x TRIAX; 18 x AS/H3501 (20°)	56.3%	0.0%	37.5%	6.3%
48 - 72	2 x MAT; 4 x TRIAX; 26 x AS/H3501 (20°)	65.0%	0.0%	30.0%	5.0%
72 - 96	2 x MAT; 3 x TRIAX; 33 x AS/H3501 (20°)	75.0%	0.0%	20.5%	4.5%
96 - 120	2 x MAT; 3 x TRIAX; 29 x AS/H3501 (20°)	72.5%	0.0%	22.5%	5.0%
120 - 144	2 x MAT; 2 x TRIAX; 23 x AS/H3501 (20°)	74.2%	0.0%	19.4%	6.5%
144 - 168	2 x MAT; 2 x TRIAX; 23 x AS/H3501 (20°)	74.2%	0.0%	19.4%	6.5%
168 - 192	2 x MAT; 2 x TRIAX; 18 x AS/H3501 (20°)	69.2%	0.0%	23.1%	7.7%
192 - 216	2 x MAT; 2 x TRIAX; 13 x AS/H3501 (20°)	61.9%	0.0%	28.6%	9.5%
216 - 240	2 x MAT; 2 x TRIAX; 13 x AS/H3501 (20°)	61.9%	0.0%	28.6%	9.5%
240 - 264	2 x MAT; 2 x TRIAX; 11 x AS/H3501 (20°)	57.9%	0.0%	31.6%	10.5%
264 - 288	2 x MAT; 2 x TRIAX; 10 x AS/H3501 (20°)	55.6%	0.0%	33.3%	11.1%
288 - 312	2 x MAT; 6 x TRIAX	0.0%	0.0%	90.0%	10.0%
Spar	4 x TRIAX	0.0%	0.0%	90.0%	10.0%

*: The first number is the number of layers (i.e. 2 x MAT means 2 layers of MAT material).

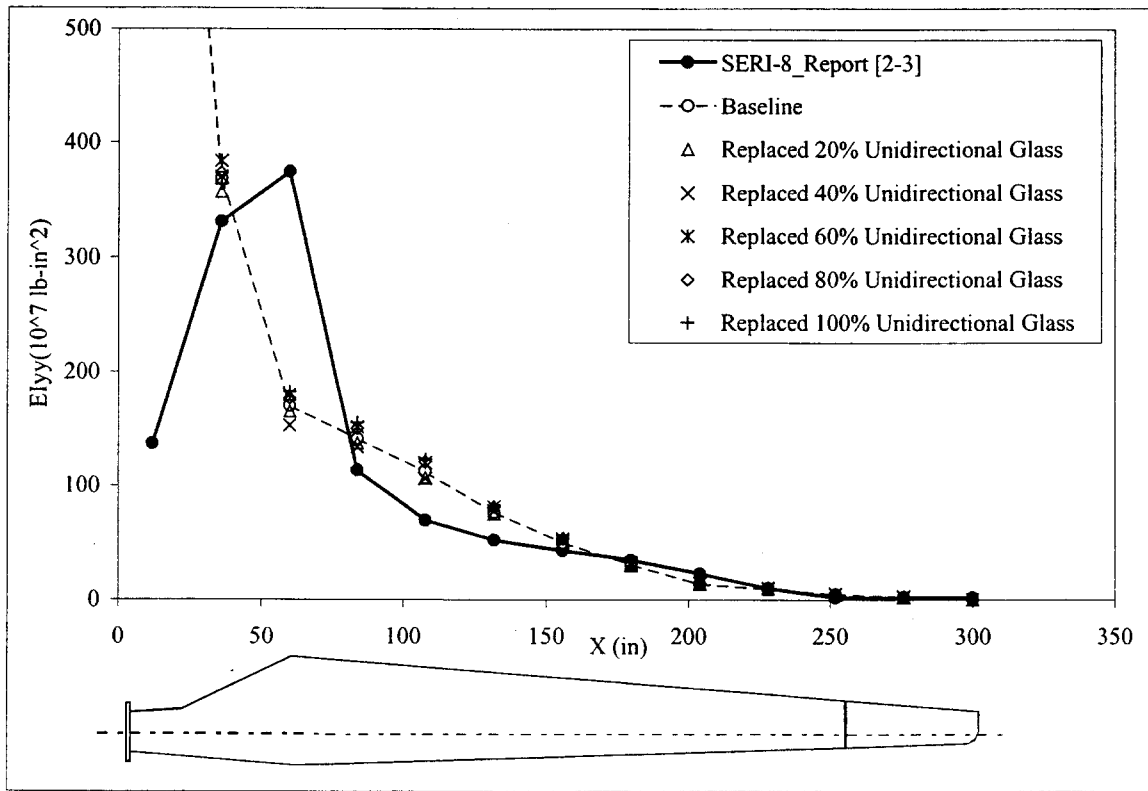


Figure 3.1 Enhanced Models Having the Same EI_{yy} Distribution

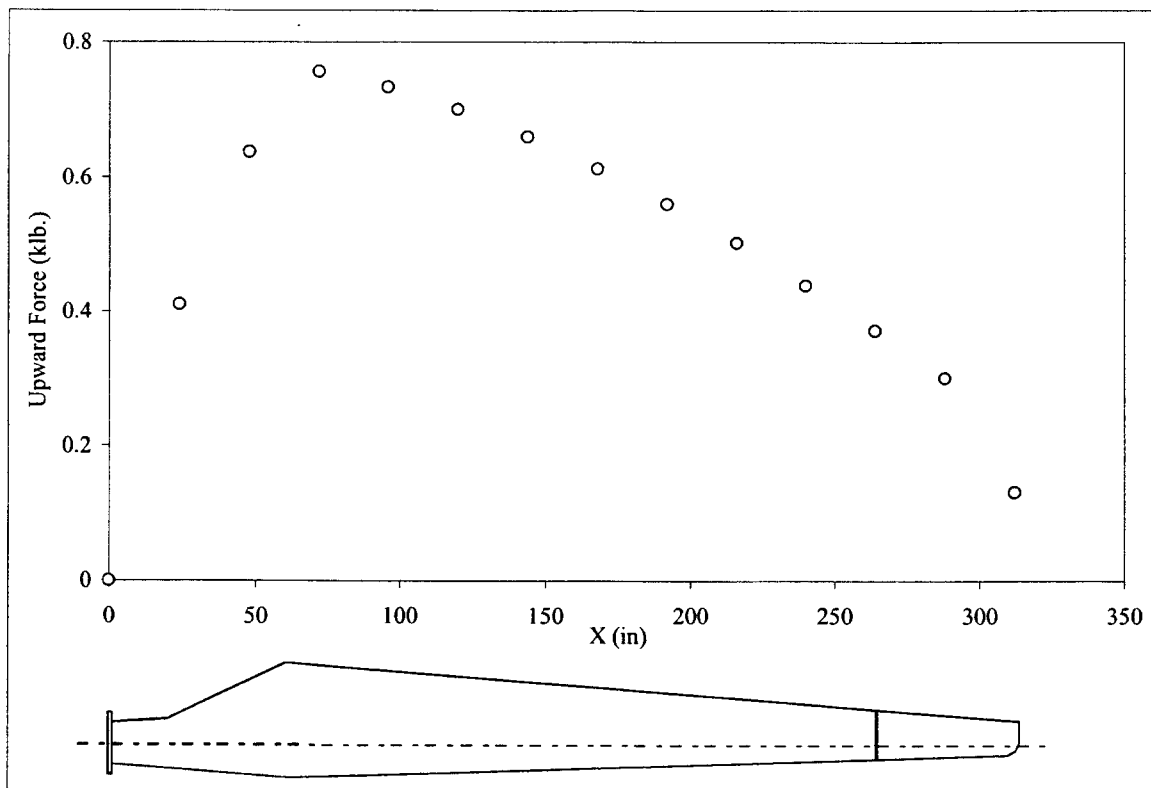


Figure 3.2 Nodal Force Distribution Input for 3D_Beam Code

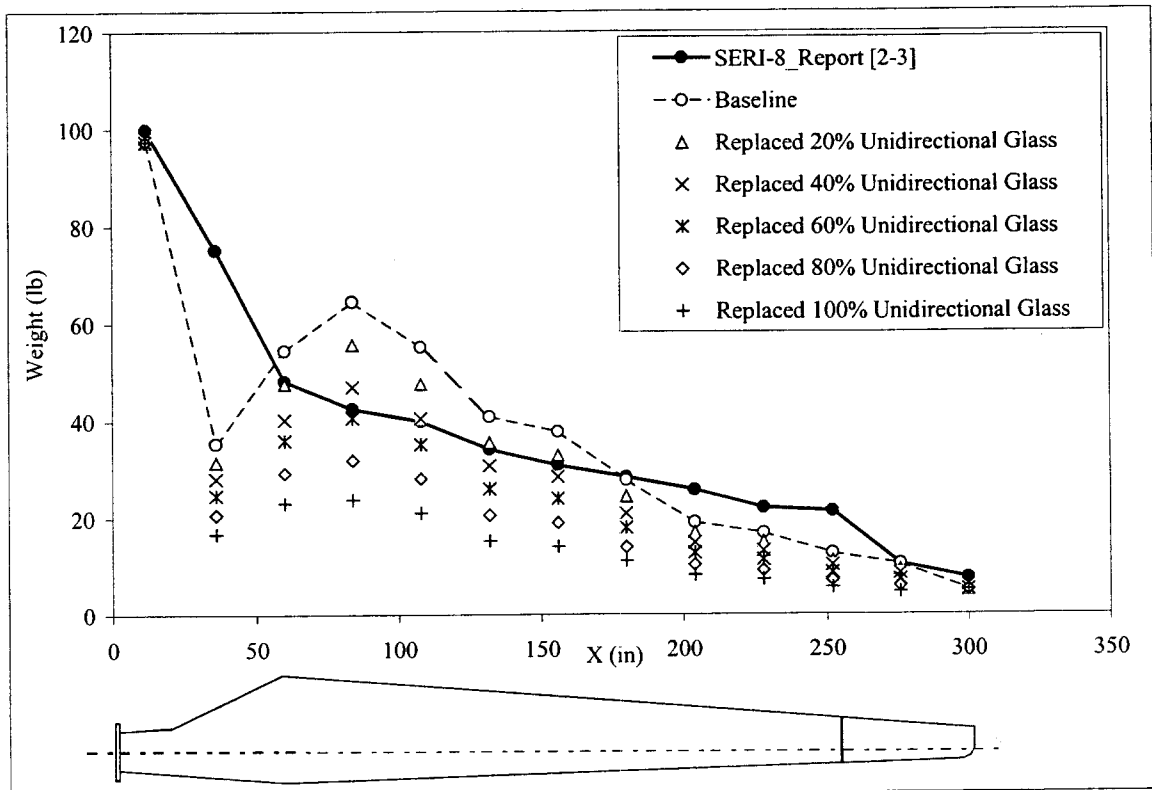


Figure 3.3 Weight Distribution for Various Configurations

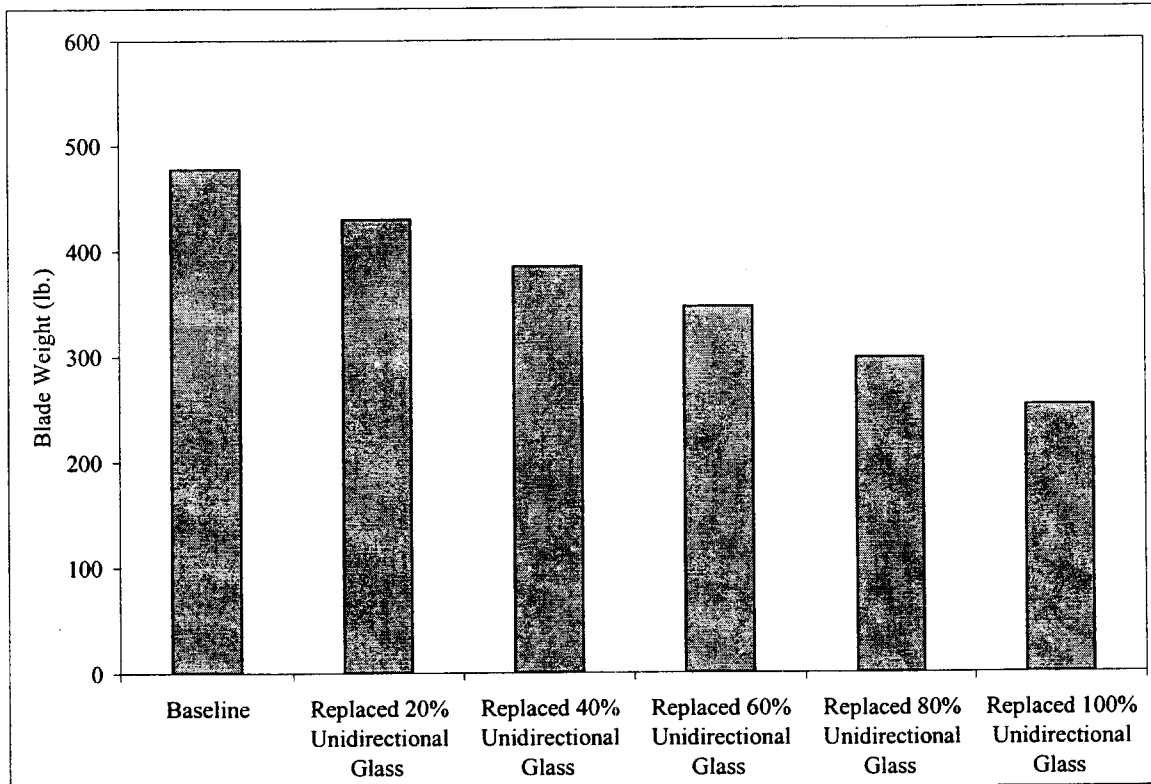


Figure 3.4 Reduction in Total Blade Weight as More Carbon Fibers are used

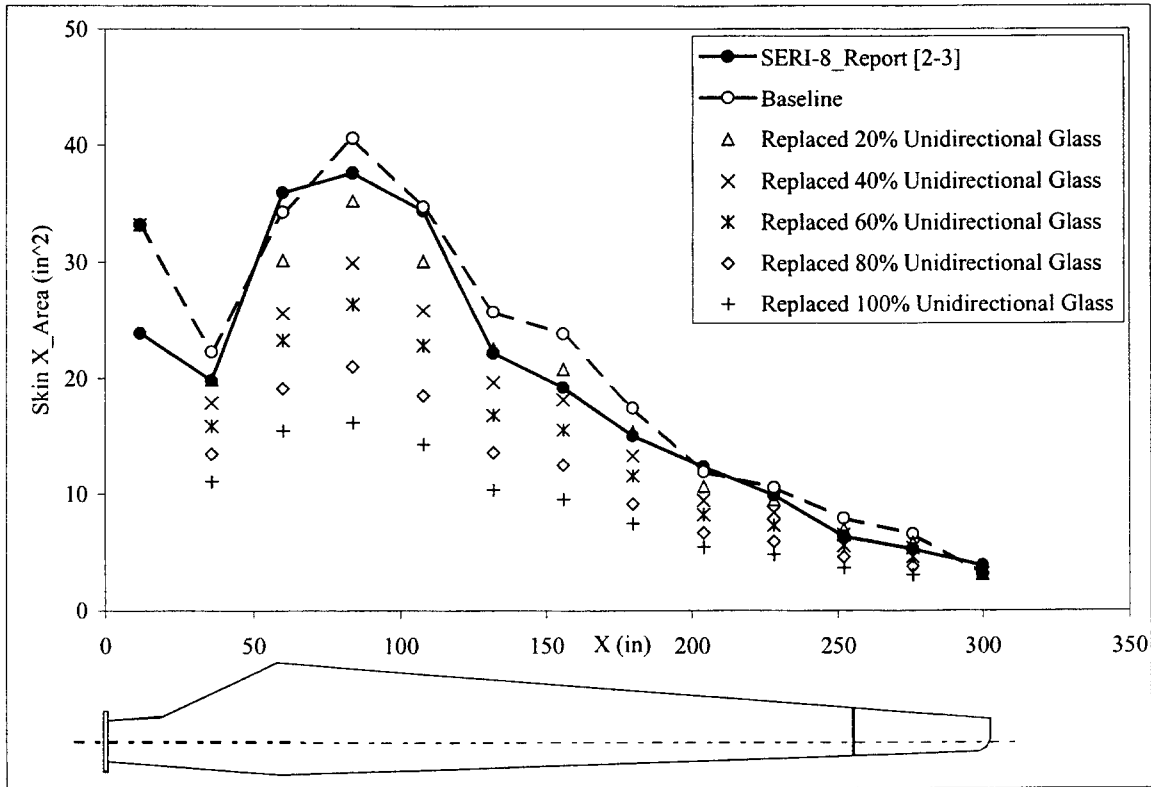


Figure 3.5 Skin Cross-section Area Distribution for Various Configurations

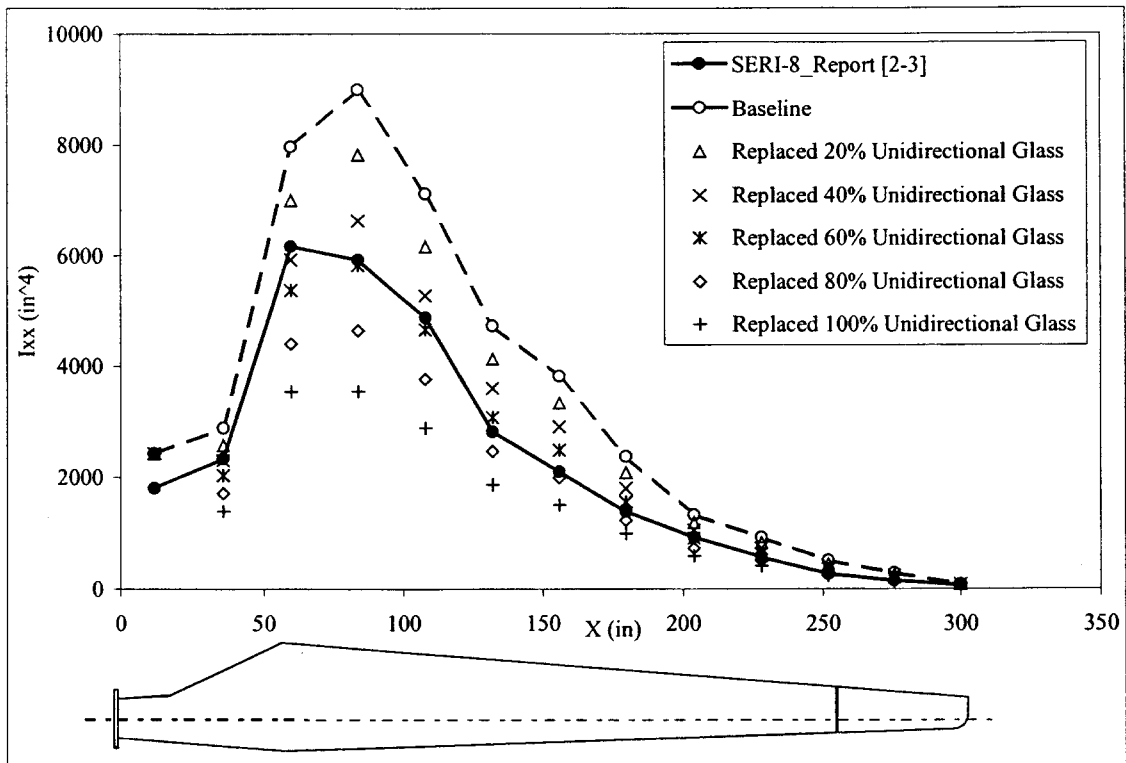


Figure 3.6 I_{xx} Distribution for Various Configurations

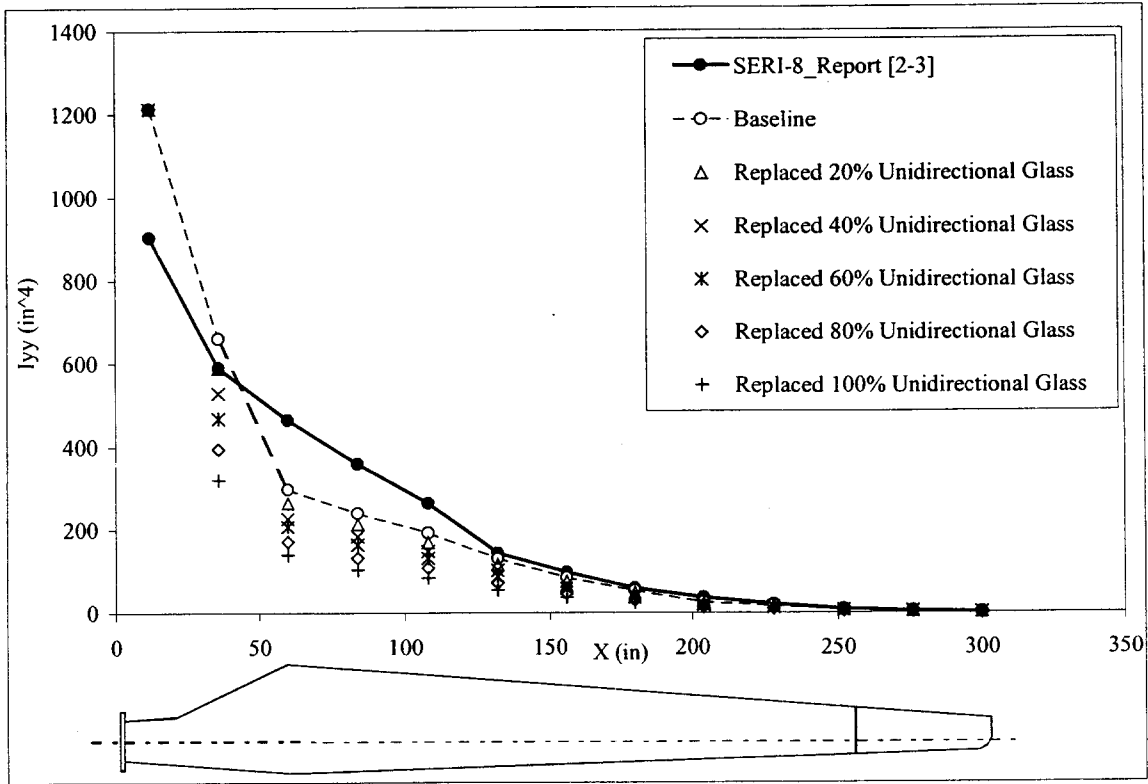


Figure 3.7 I_{yy} Distribution for Various Configurations

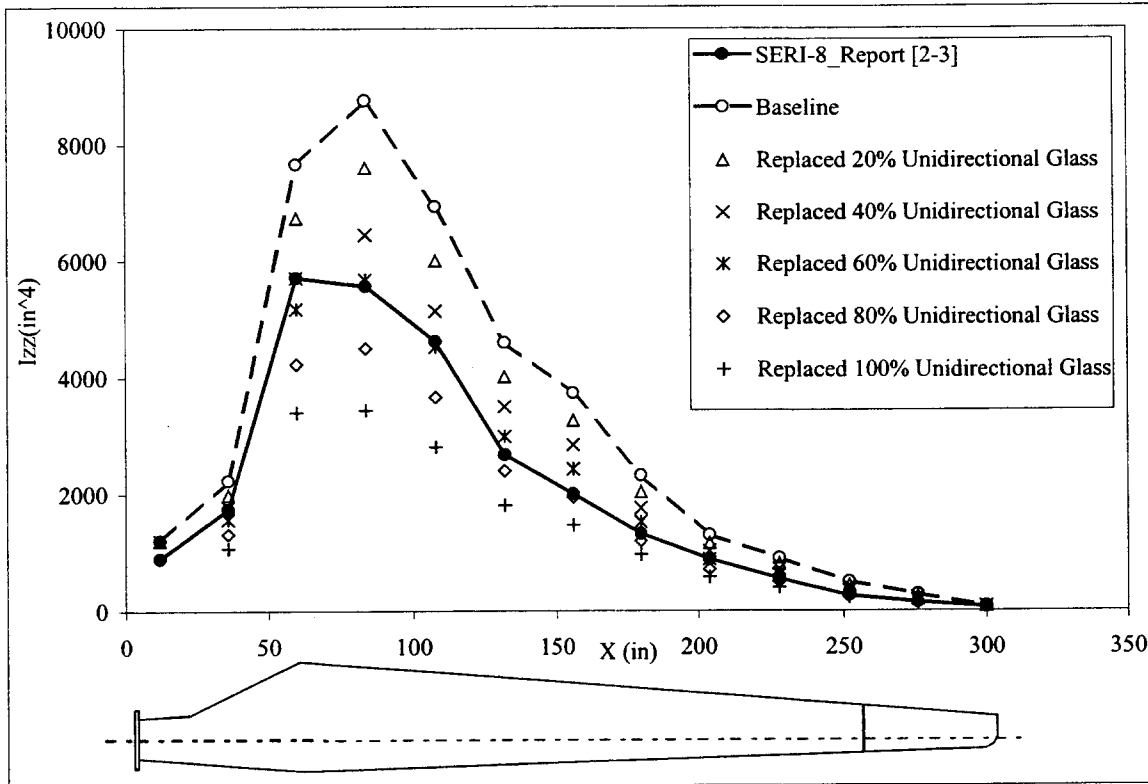


Figure 3.8 I_{zz} Distribution for Various Configurations

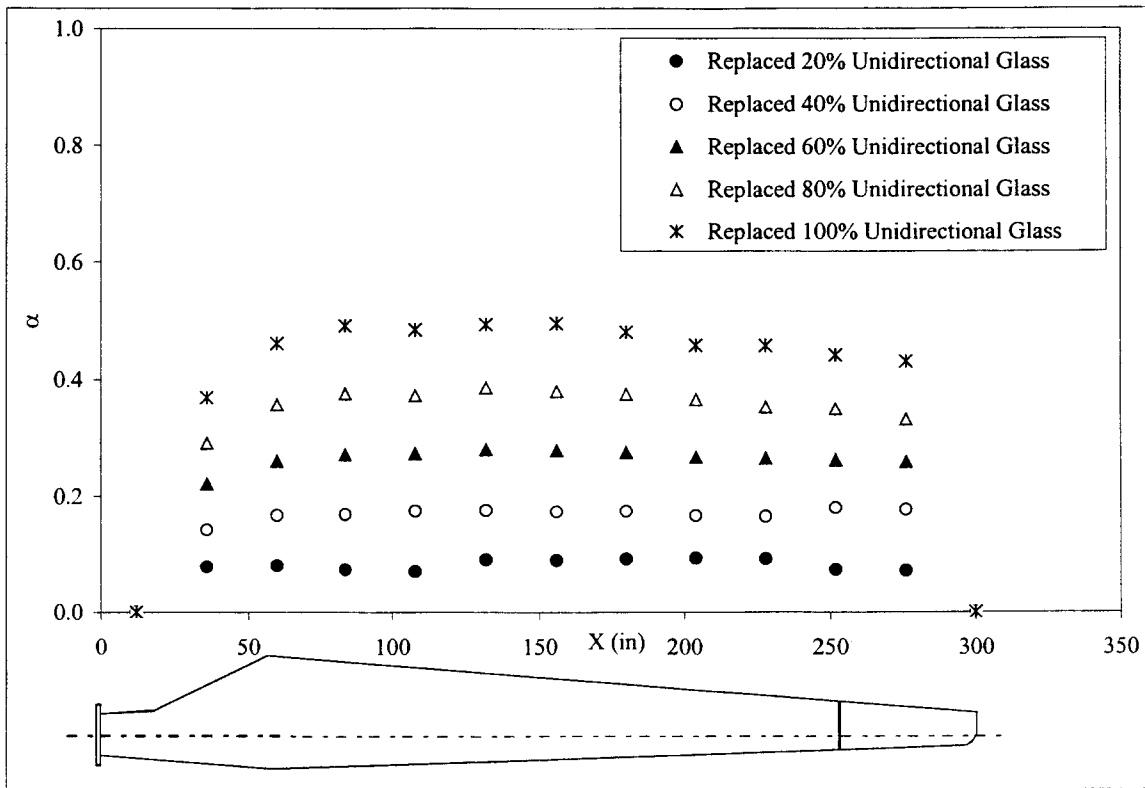


Figure 3.9 α Distribution for Various Configurations with Bend-Twist Coupling

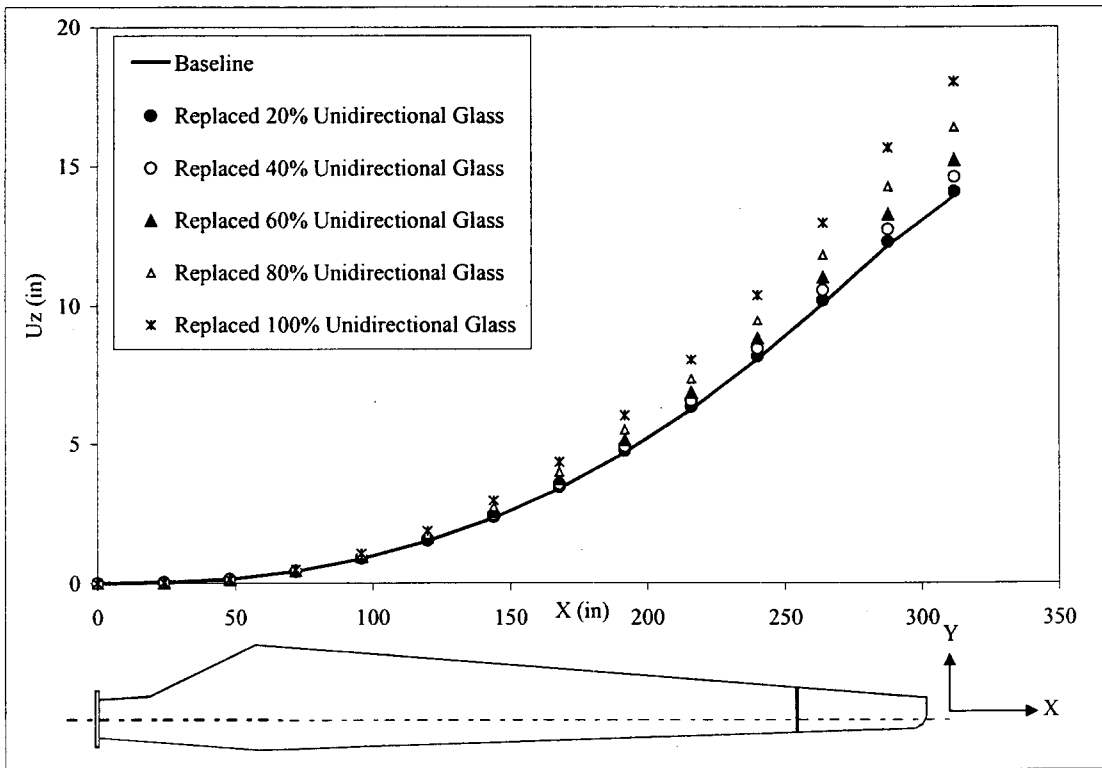


Figure 3.10 Vertical Deflection along the Blade for Various Configurations with Bend-Twist Coupling. Wind Load is 70 m/s and C_d is 1.7.

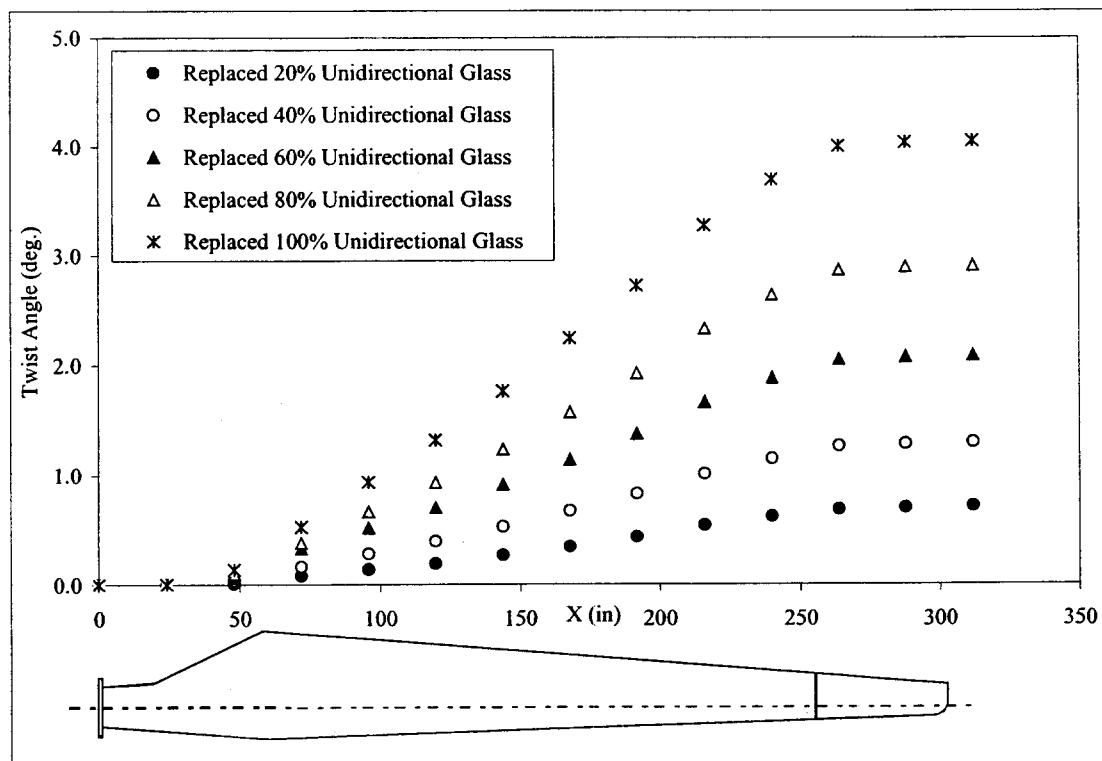


Figure 3.11 Bend-induced Twist along the Blade for Various Configurations with Bend-Twist Coupling. Wind Load is 70 m/s and C_d is 1.7.

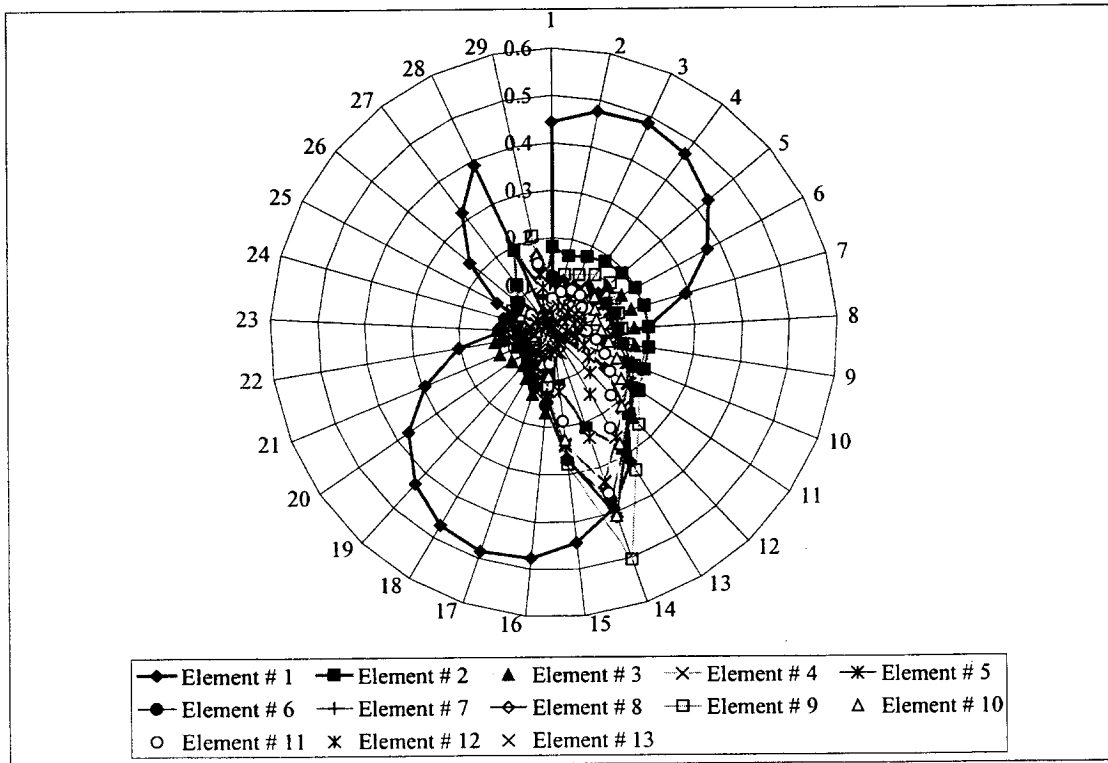


Figure 3.12 Failure Index along the "Brick Elements" for the Respective Blade Element (Baseline Configuration)

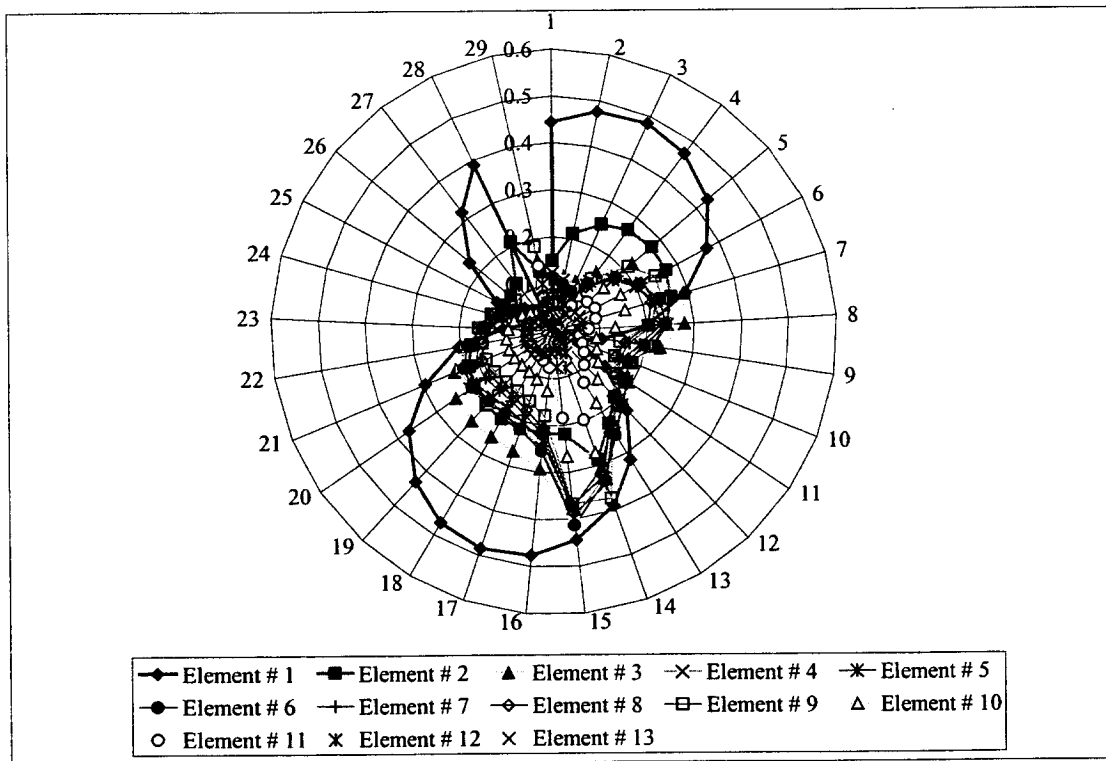


Figure 3.13 Failure Index along the "Brick Elements" for the Respective Blade Element (Twist-to-Feather; 100% Unidirectional Glass Replacement)

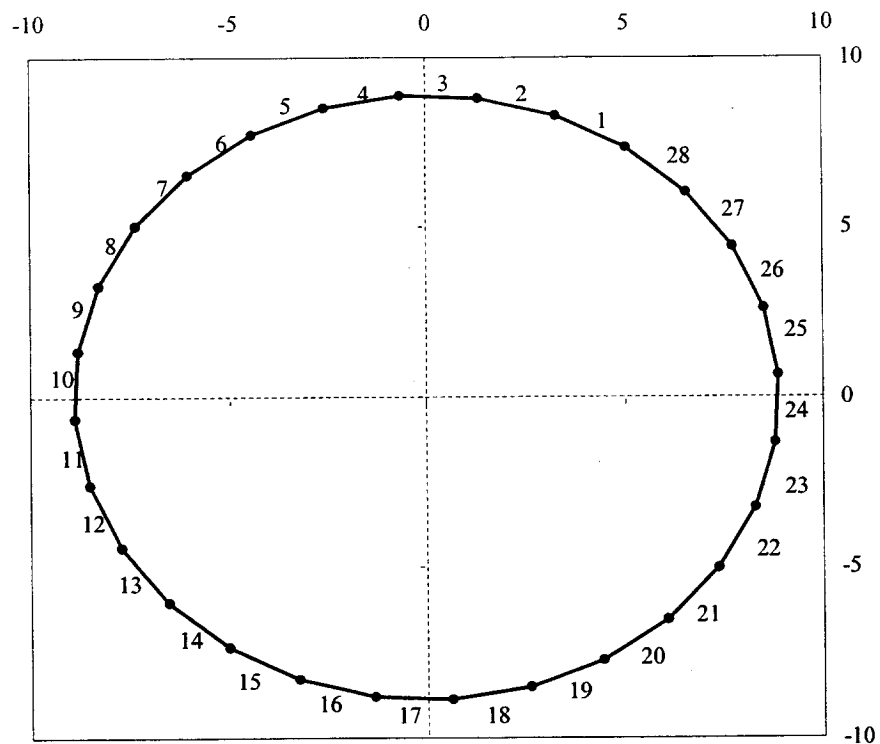
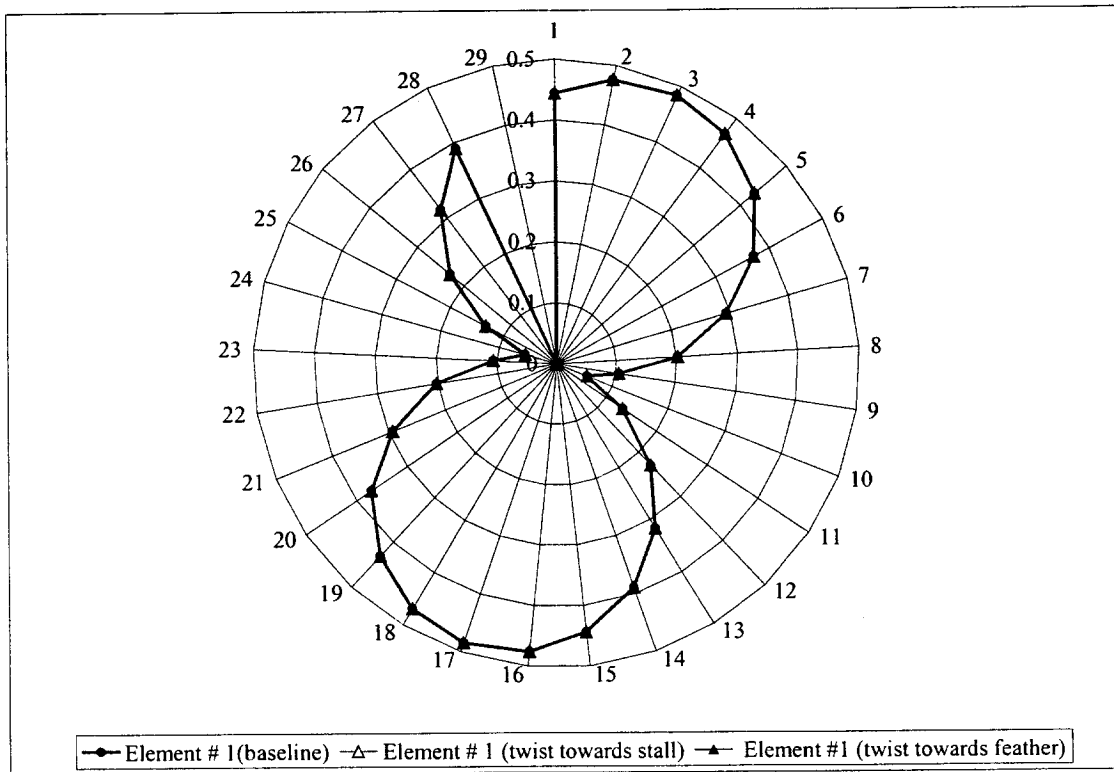


Figure 3.14 Top: Failure Index along the "Brick Elements" for Blade Element No. 1 (Station 0 - 24) for the Baseline Configuration and Configuration with 100% Unidirectional Glass Replacement (Wind Load is 70 m/s; C_d is 1.7).
Bottom: Brick Element Numbering for Blade Element No. 1.

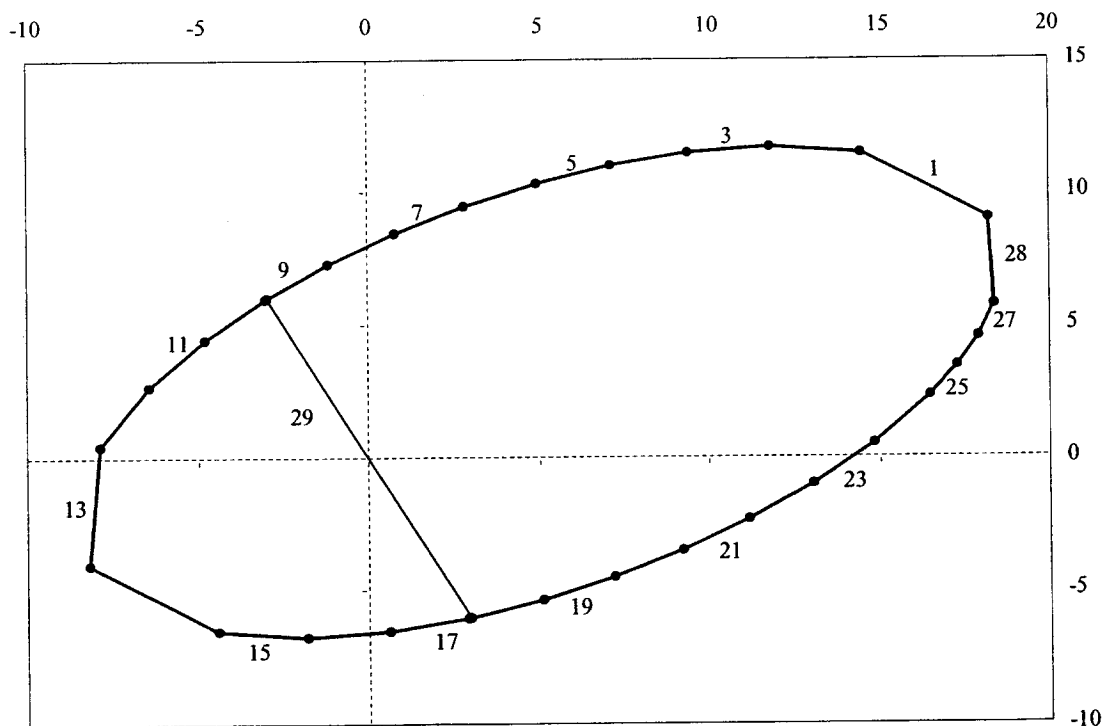
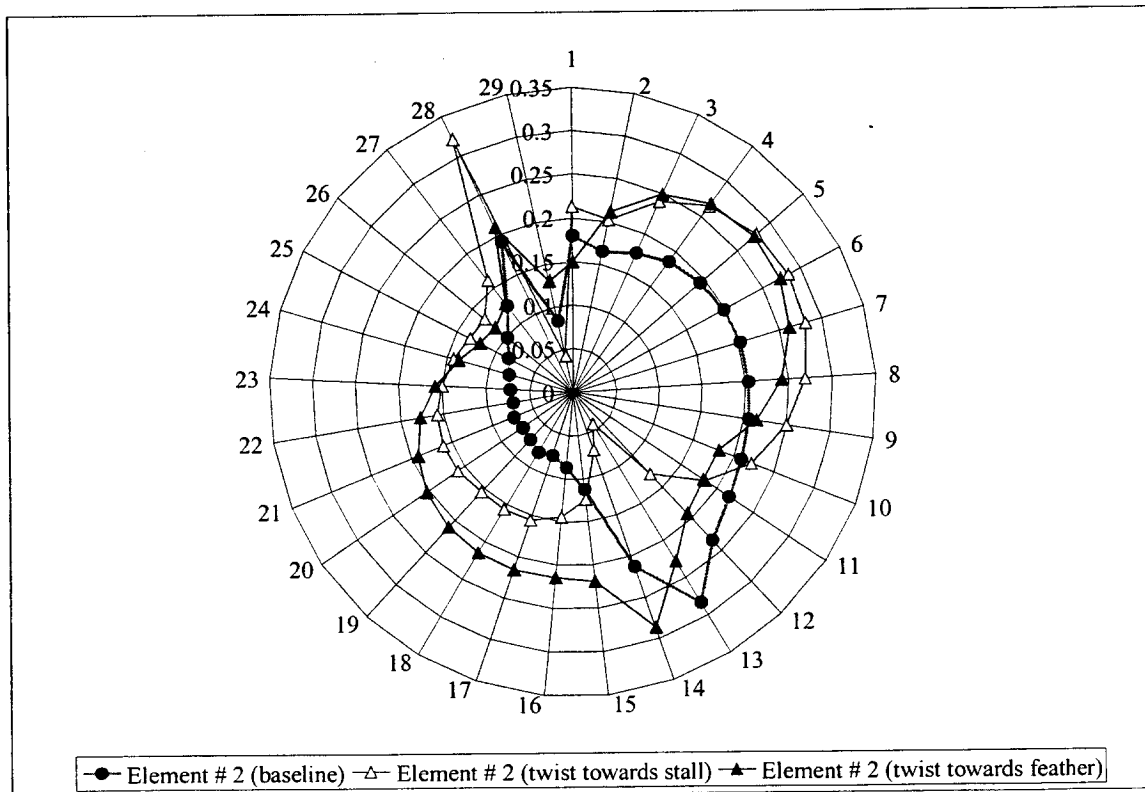


Figure 3.15 Top: Failure Index along the "Brick Elements" for Blade Element No. 2 (Station 24 - 48) for the Baseline Configuration and Configuration with 100% Unidirectional Glass Replacement (Wind Load is 70 m/s; C_d is 1.7).
Bottom: Brick Element Numbering for Blade Element No. 2.

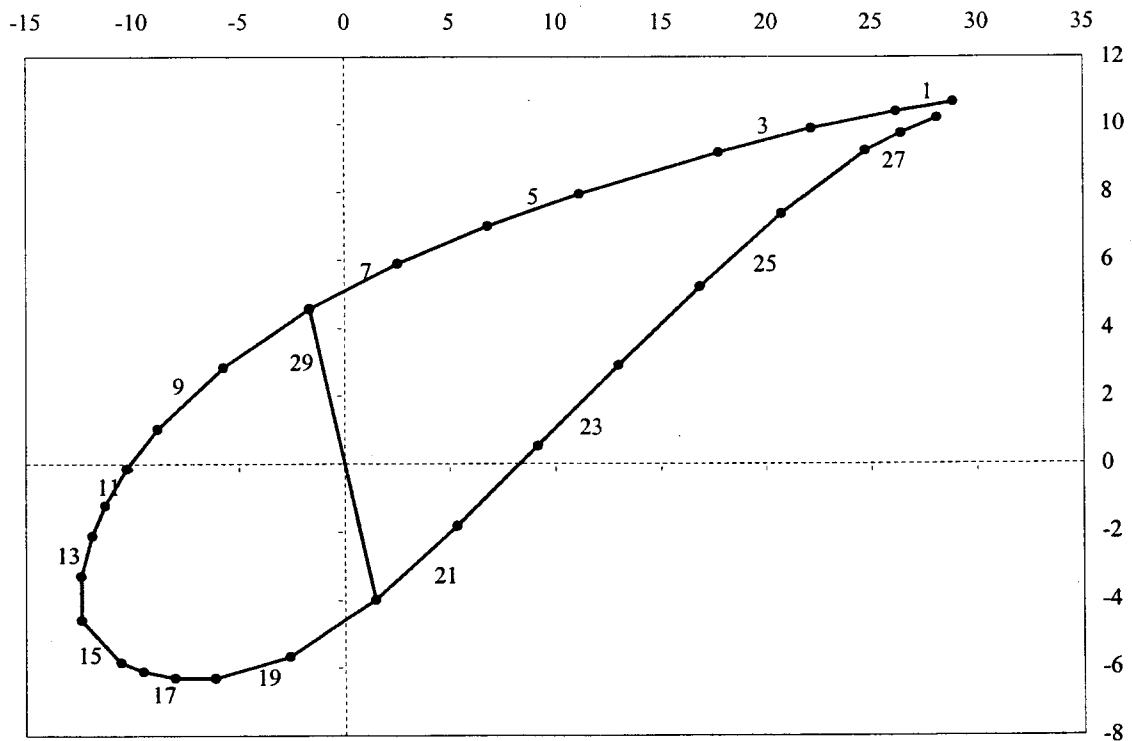
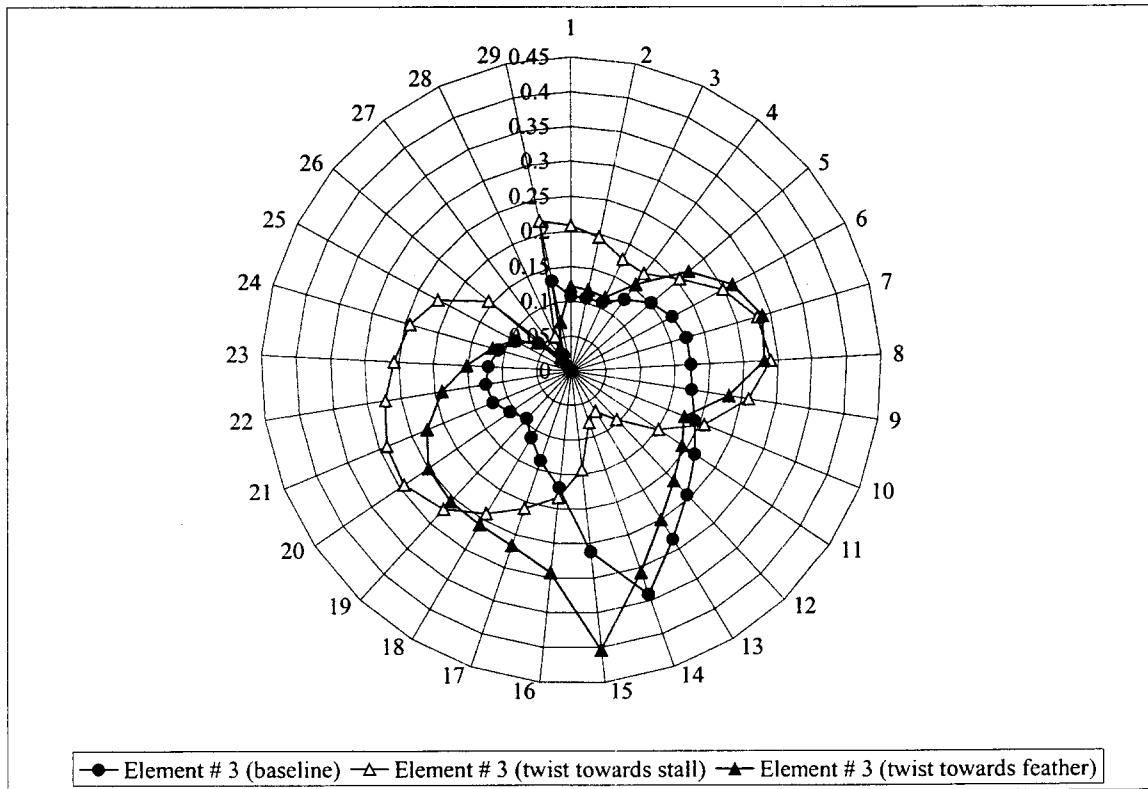


Figure 3.16 **Top:** Failure Index along the "Brick Elements" for Blade Element No. 3 (Station 48 - 72) for the Baseline Configuration and Configuration with 100% Unidirectional Glass Replacement (Wind Load is 70 m/s; C_d is 1.7).

Bottom: Brick Element Numbering for Blade Element No. 3.

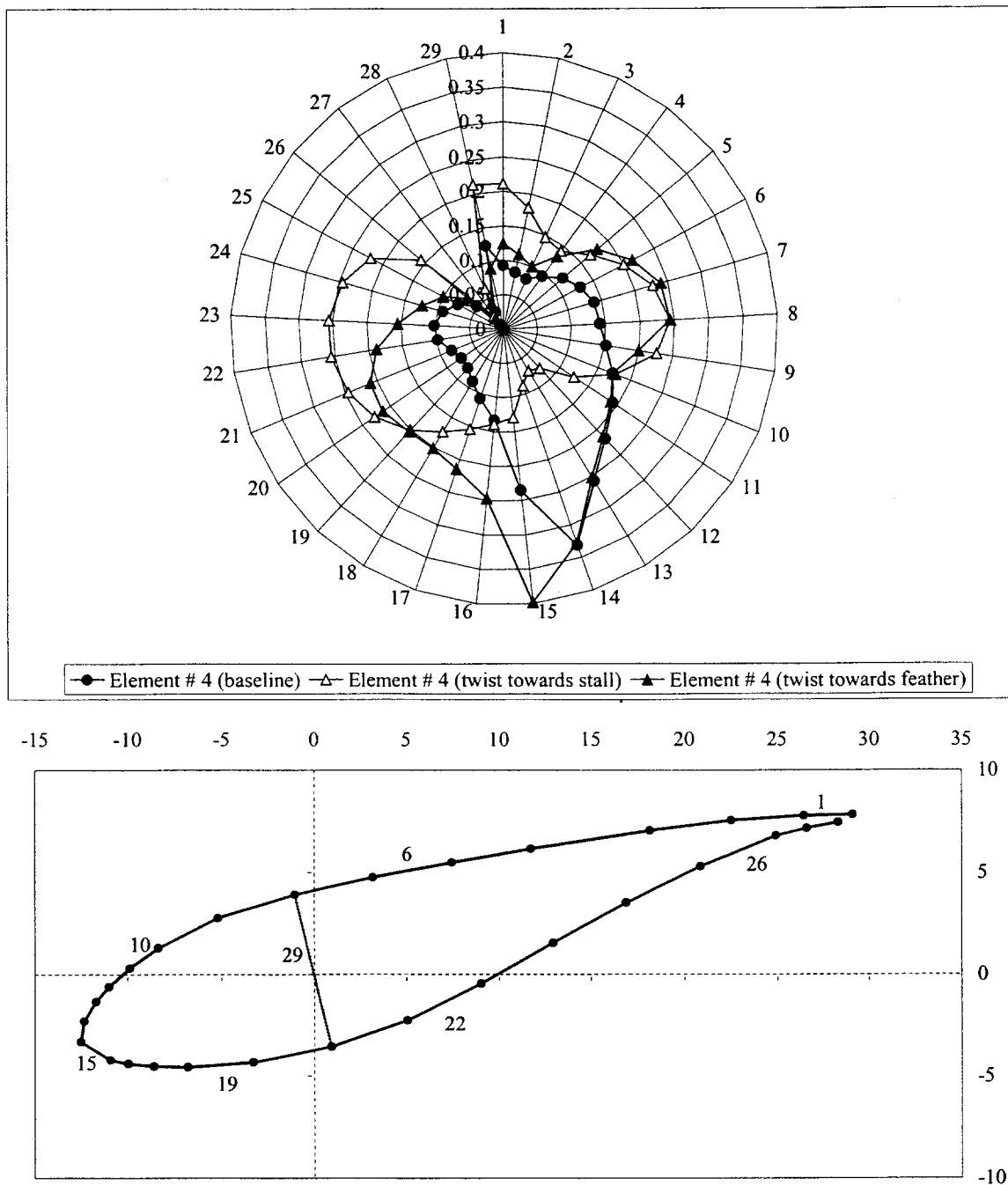


Figure 3.17 **Top:** Failure Index along the "Brick Elements" for Blade Element No. 4 (Station 72 - 96) for the Baseline Configuration and Configuration with 100% Unidirectional Glass Replacement (Wind Load is 70 m/s; C_d is 1.7).

Bottom: Brick Element Numbering for Blade Element No. 4.

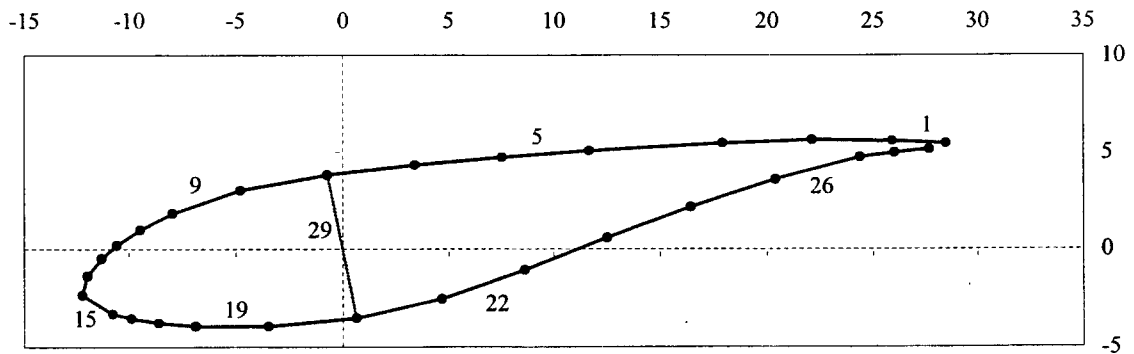
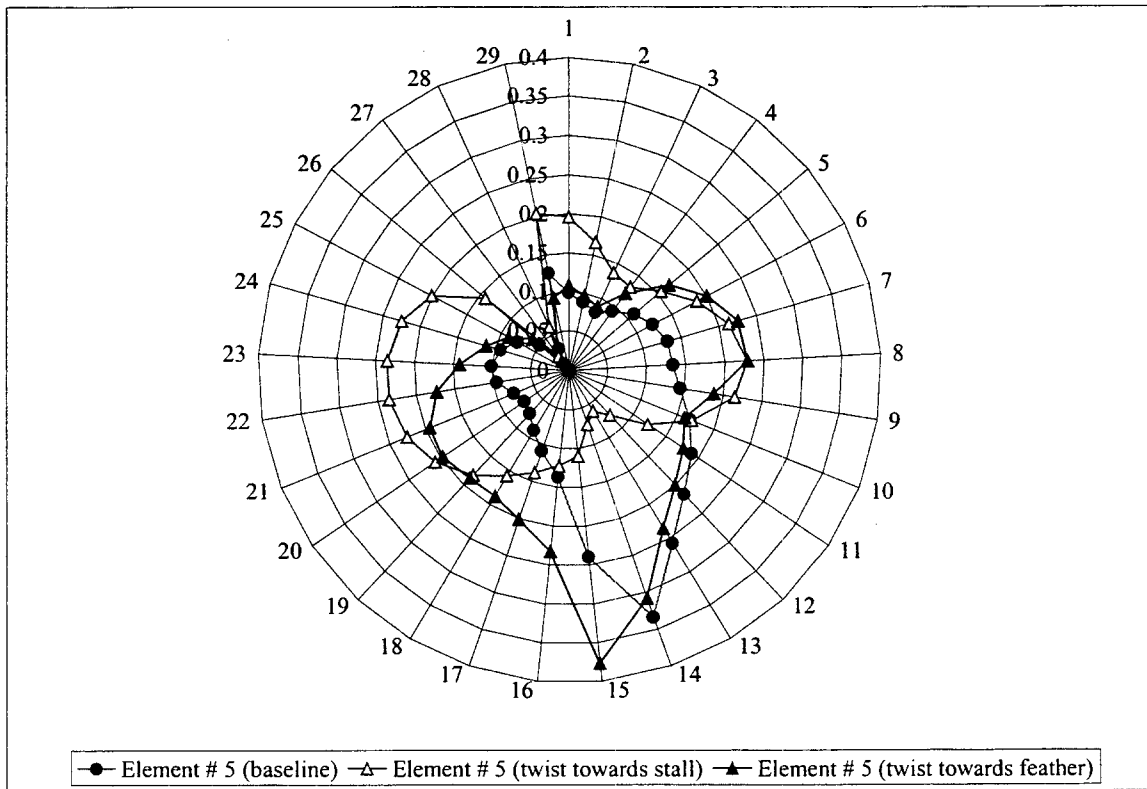


Figure 3.18 Top: Failure Index along the "Brick Elements" for Blade Element No. 5 (Station 96 - 120) for the Baseline Configuration and Configuration with 100% Unidirectional Glass Replacement (Wind Load is 70 m/s; C_d is 1.7).

Bottom: Brick Element Numbering for Blade Element No. 5.

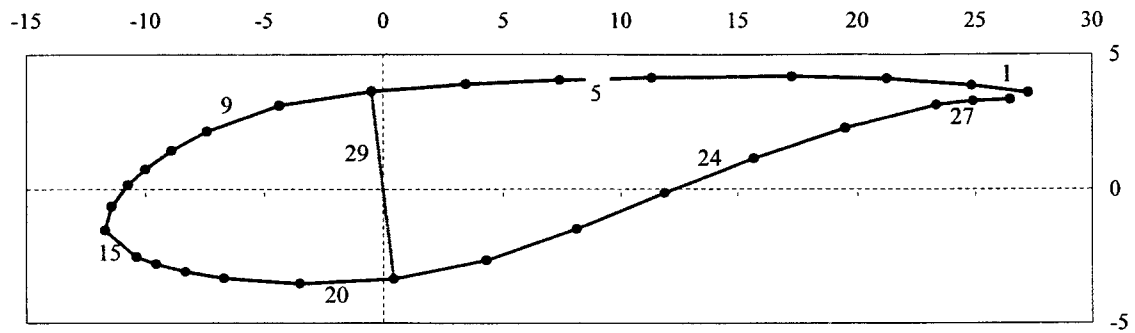
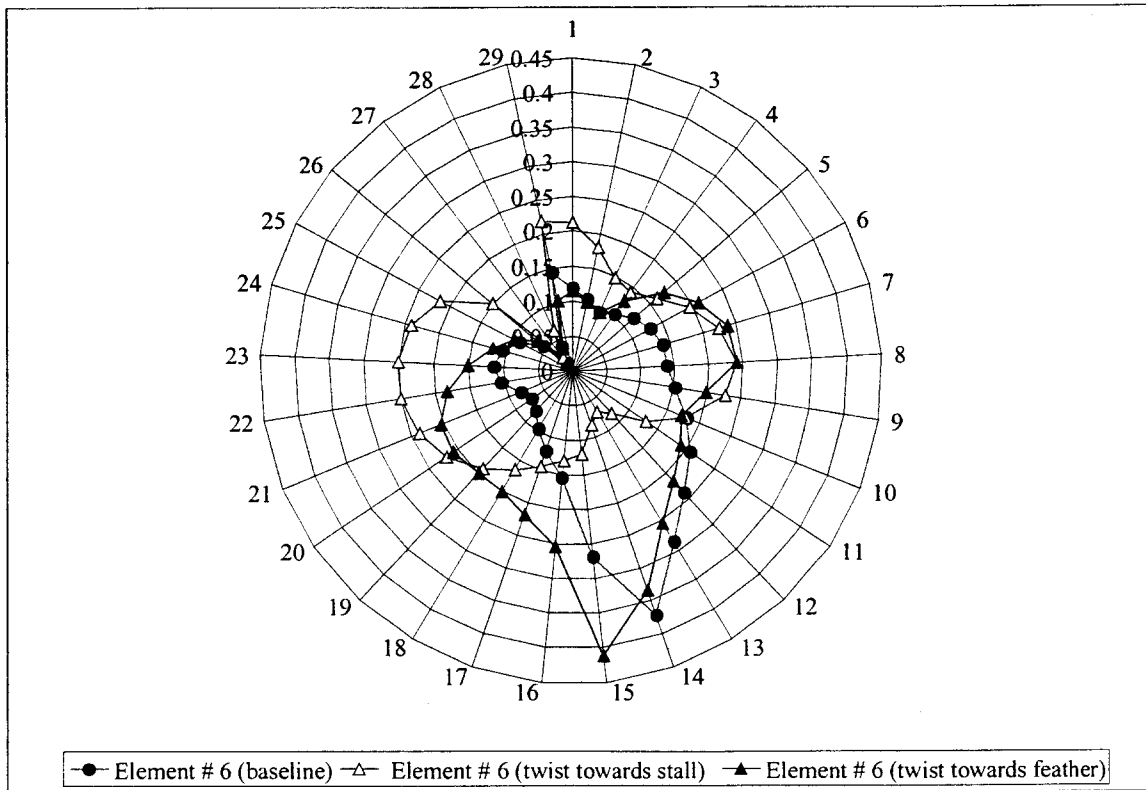


Figure 3.19 **Top:** Failure Index along the "Brick Elements" for Blade Element No. 6 (Station 120 - 144) for the Baseline Configuration and Configuration with 100% Unidirectional Glass Replacement (Wind Load is 70 m/s; C_d is 1.7).

Bottom: Brick Element Numbering for Blade Element No. 6.

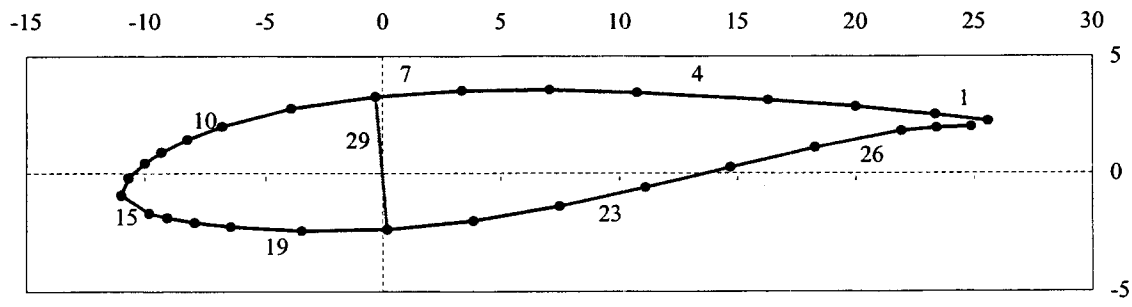
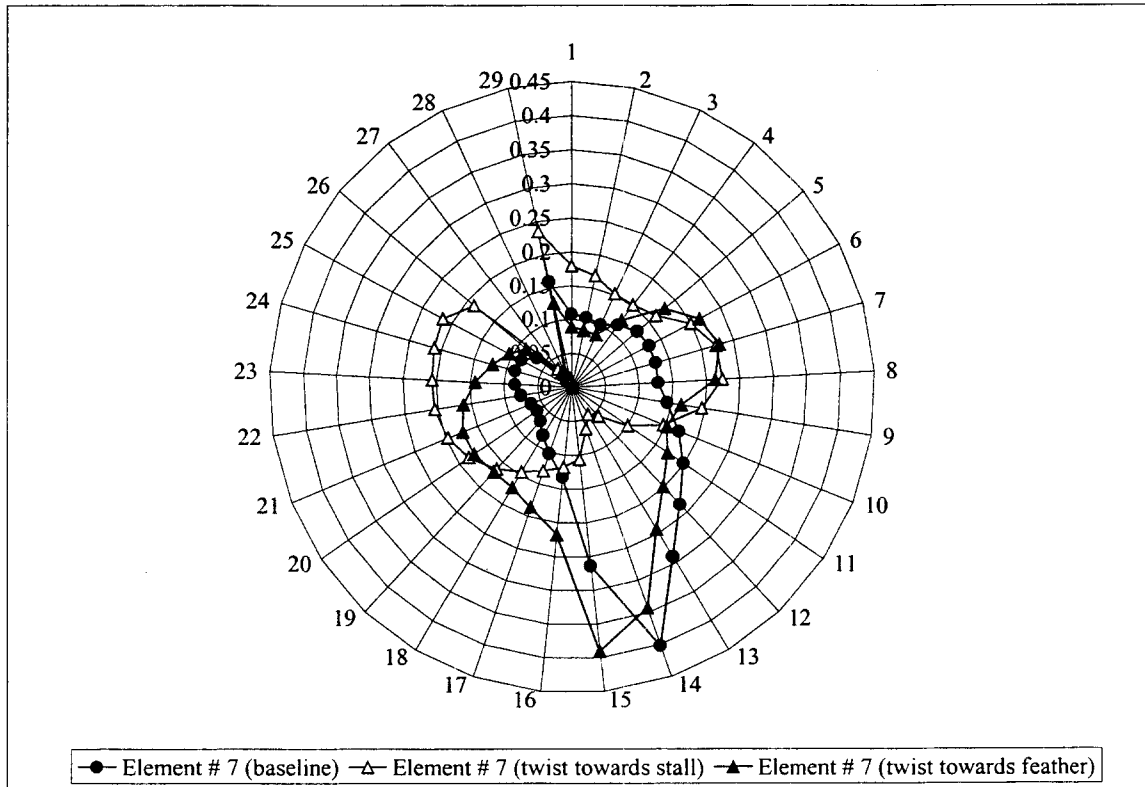


Figure 3.20 **Top:** Failure Index along the "Brick Elements" for Blade Element No. 7 (Station 144 - 168) for the Baseline Configuration and Configuration with 100% Unidirectional Glass Replacement (Wind Load is 70 m/s; C_d is 1.7).
Bottom: Brick Element Numbering for Blade Element No. 7.

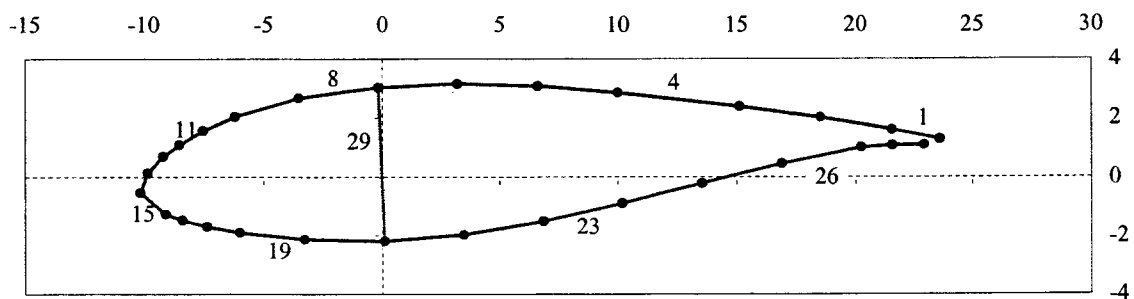
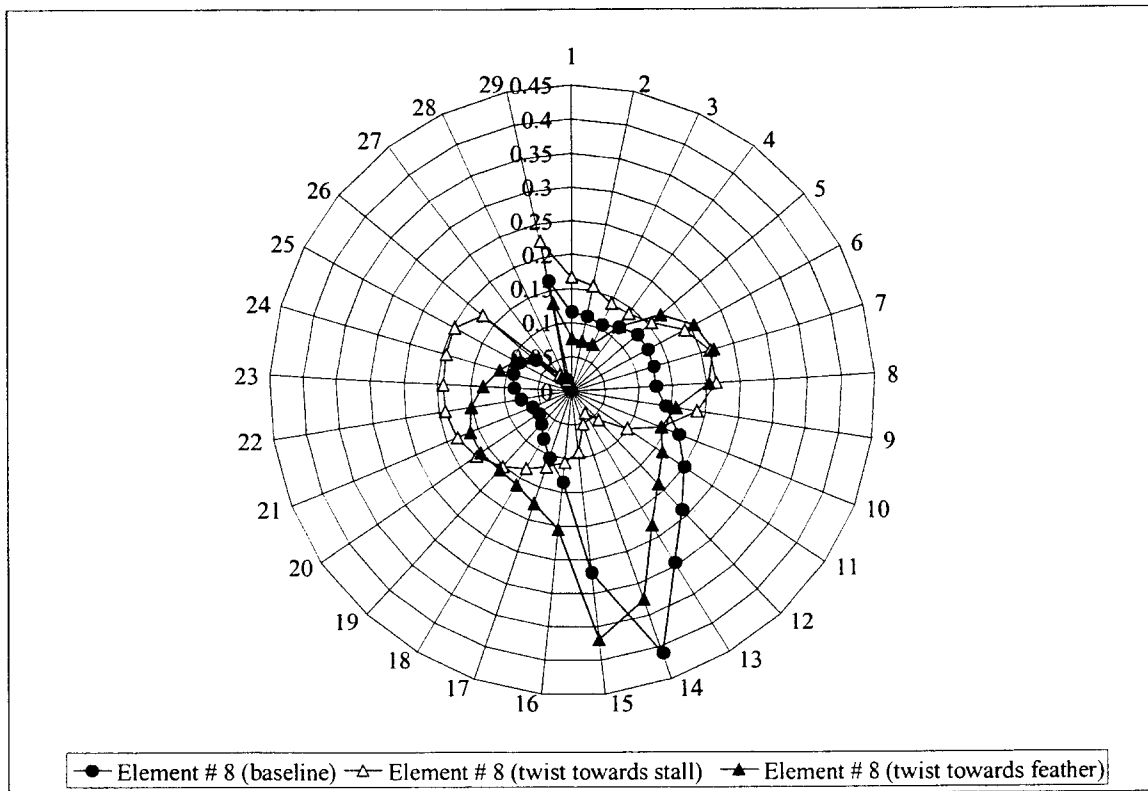


Figure 3.21 **Top:** Failure Index along the "Brick Elements" for Blade Element No. 8 (Station 168 - 192) for the Baseline Configuration and Configuration with 100% Unidirectional Glass Replacement (Wind Load is 70 m/s; C_d is 1.7).

Bottom: Brick Element Numbering for Blade Element No. 8.

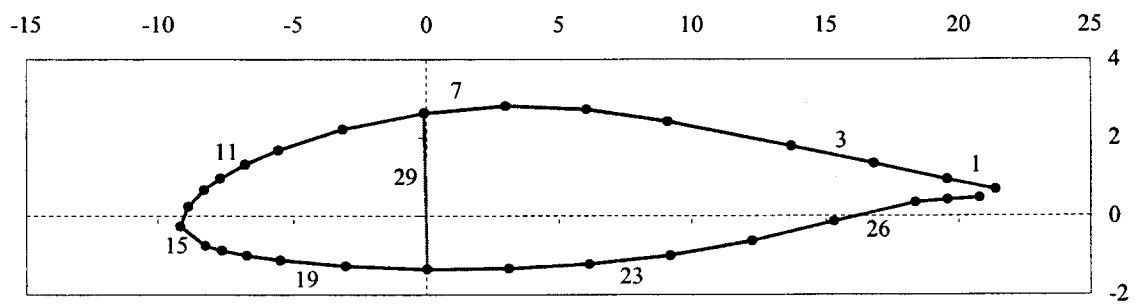
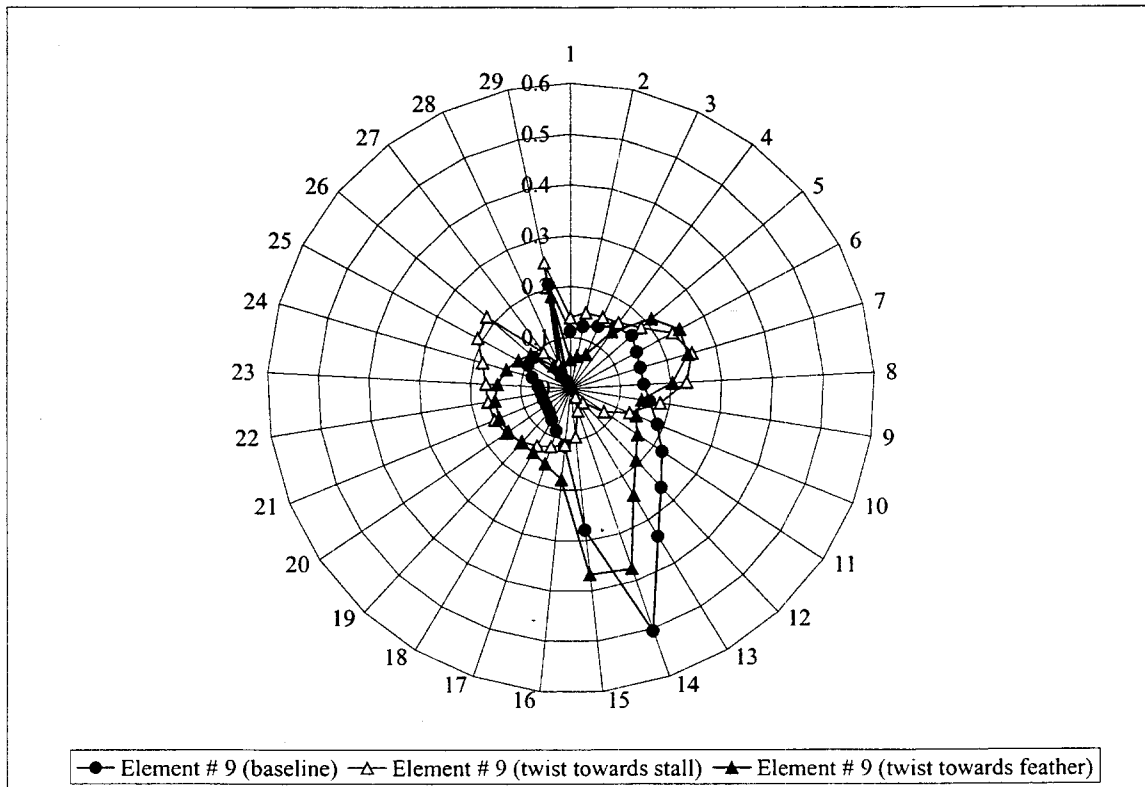


Figure 3.22 Top: Failure Index along the "Brick Elements" for Blade Element No. 9 (Station 192 - 216) for the Baseline Configuration and Configuration with 100% Unidirectional Glass Replacement (Wind Load is 70 m/s; C_d is 1.7).
Bottom: Brick Element Numbering for Blade Element No. 9.

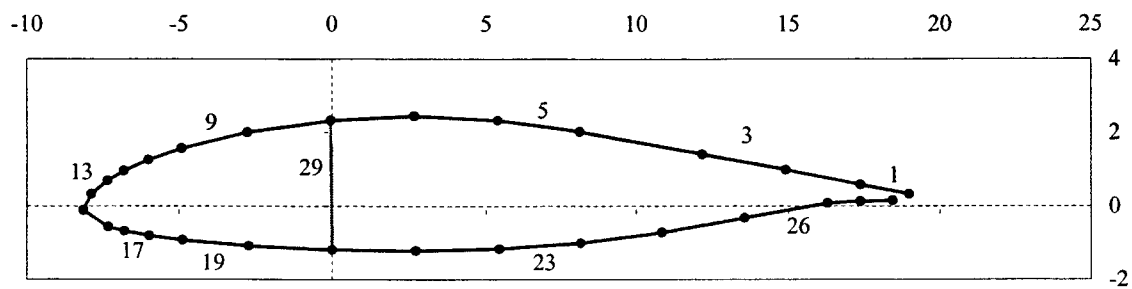
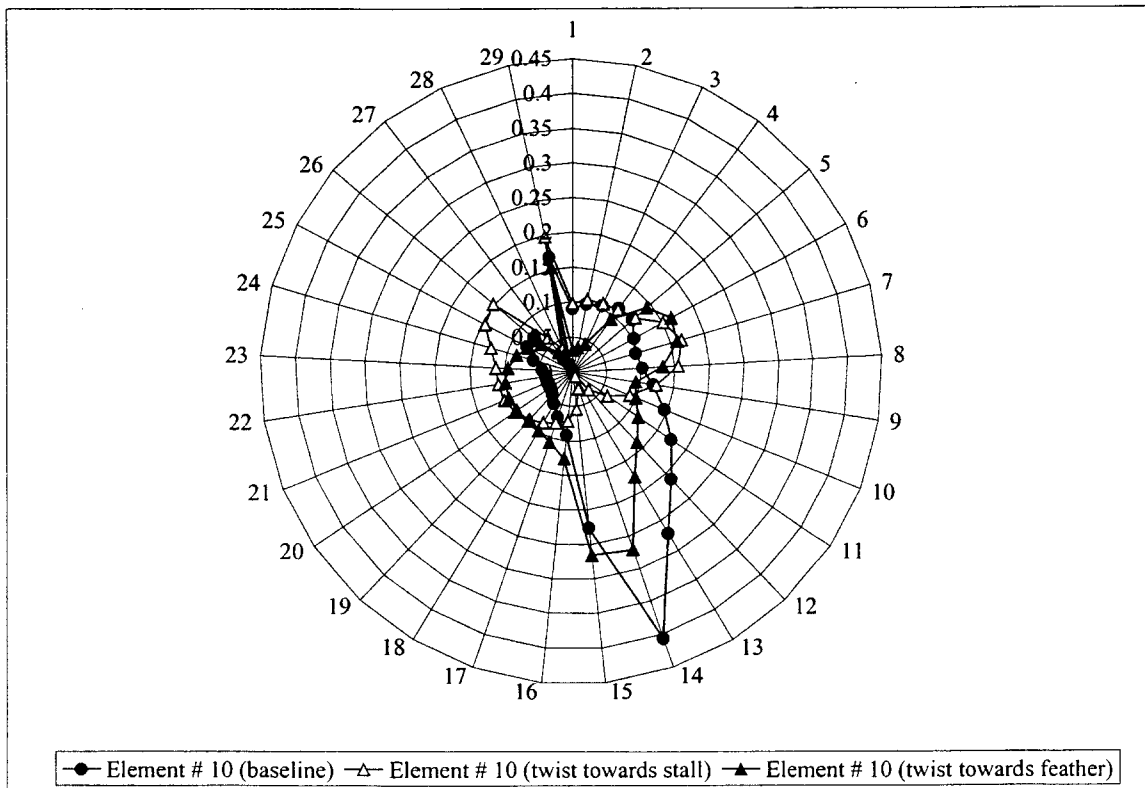


Figure 3.23 **Top:** Failure Index along the "Brick Elements" for Blade Element No. 10 (Station 216 - 240) for the Baseline Configuration and Configuration with 100% Unidirectional Glass Replacement (Wind Load is 70 m/s; C_d is 1.7).

Bottom: Brick Element Numbering for Blade Element No. 10.

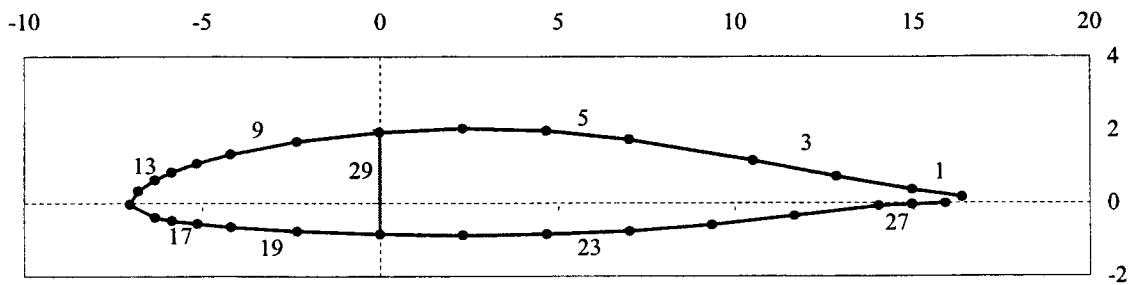
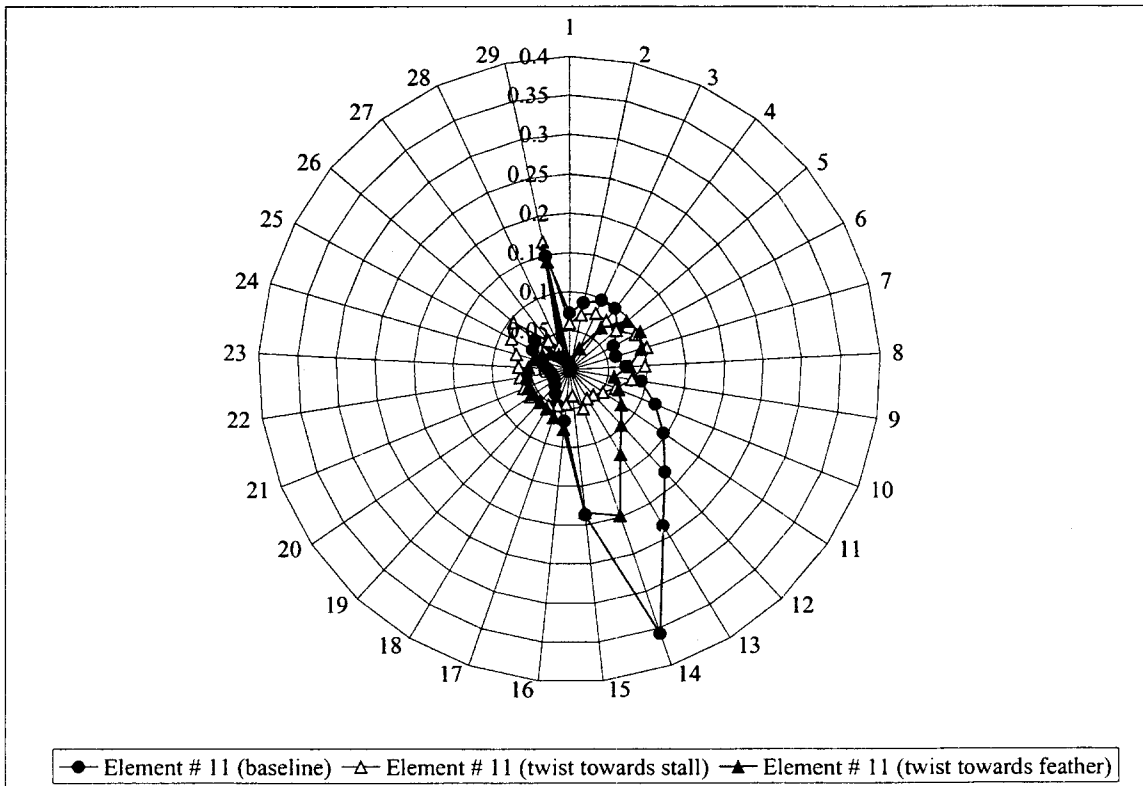


Figure 3.24 **Top:** Failure Index along the "Brick Elements" for Blade Element No. 11 (Station 240 - 264) for the Baseline Configuration and Configuration with 100% Unidirectional Glass Replacement (Wind Load is 70 m/s; C_d is 1.7).

Bottom: Brick Element Numbering for Blade Element No. 11.

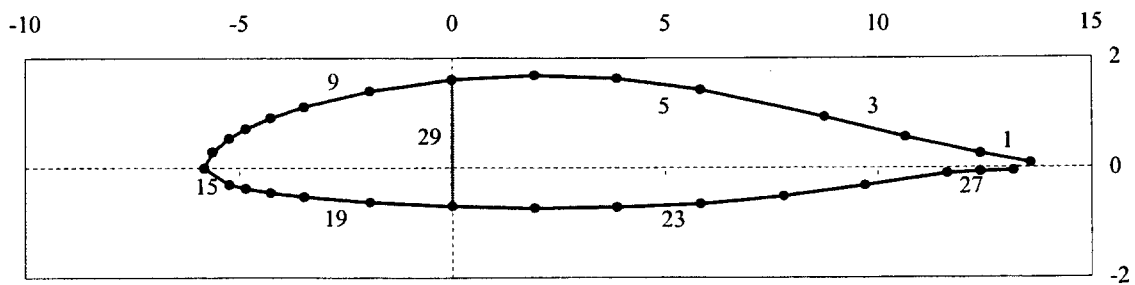
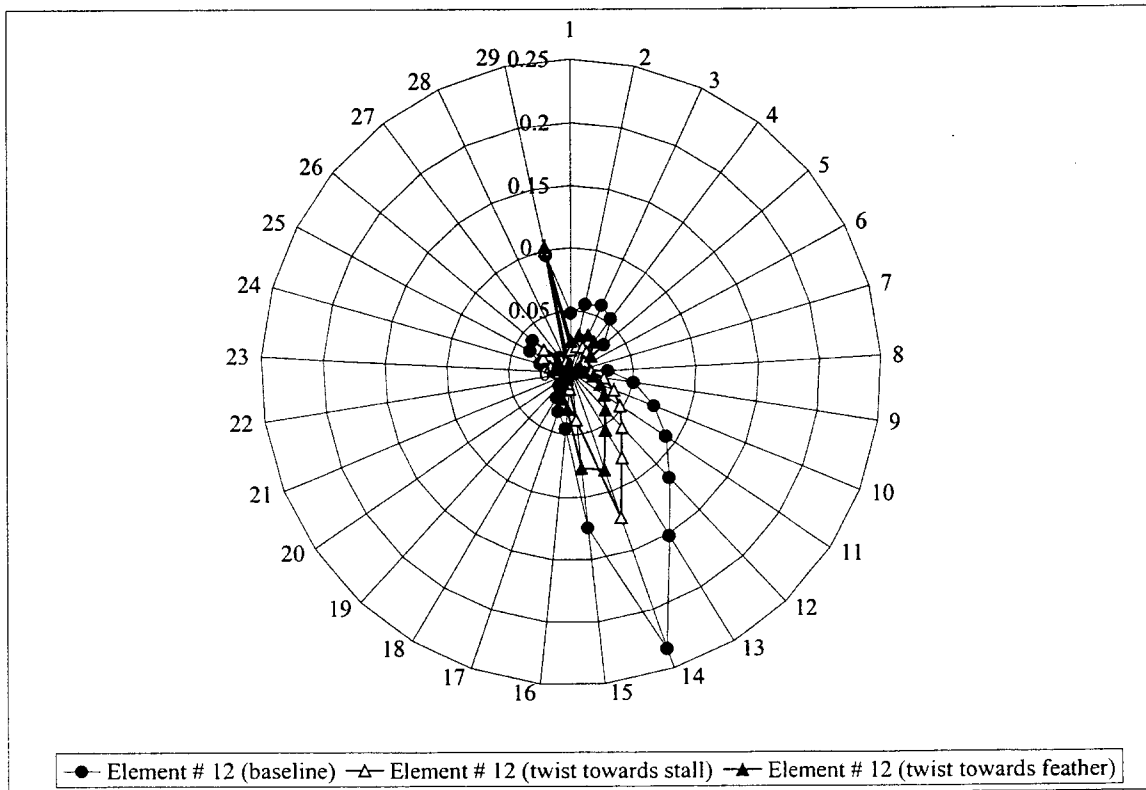


Figure 3.25 Top: Failure Index along the "Brick Elements" for Blade Element No. 12 (Station 264 - 288) for the Baseline Configuration and Configuration with 100% Unidirectional Glass Replacement (Wind Load is 70 m/s; C_d is 1.7).
Bottom: Brick Element Numbering for Blade Element No. 12.

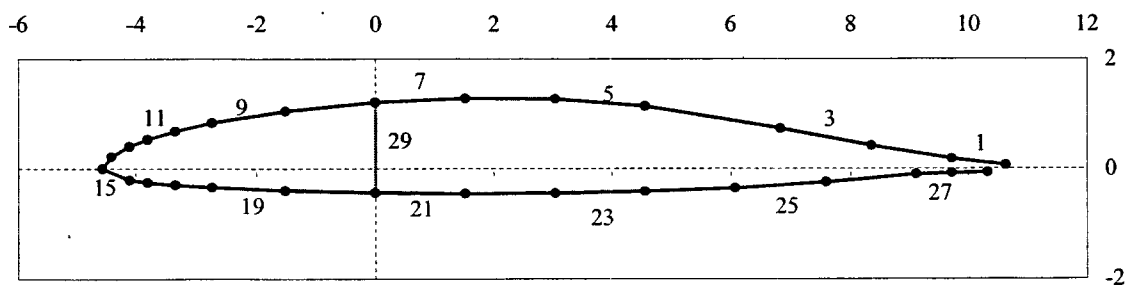
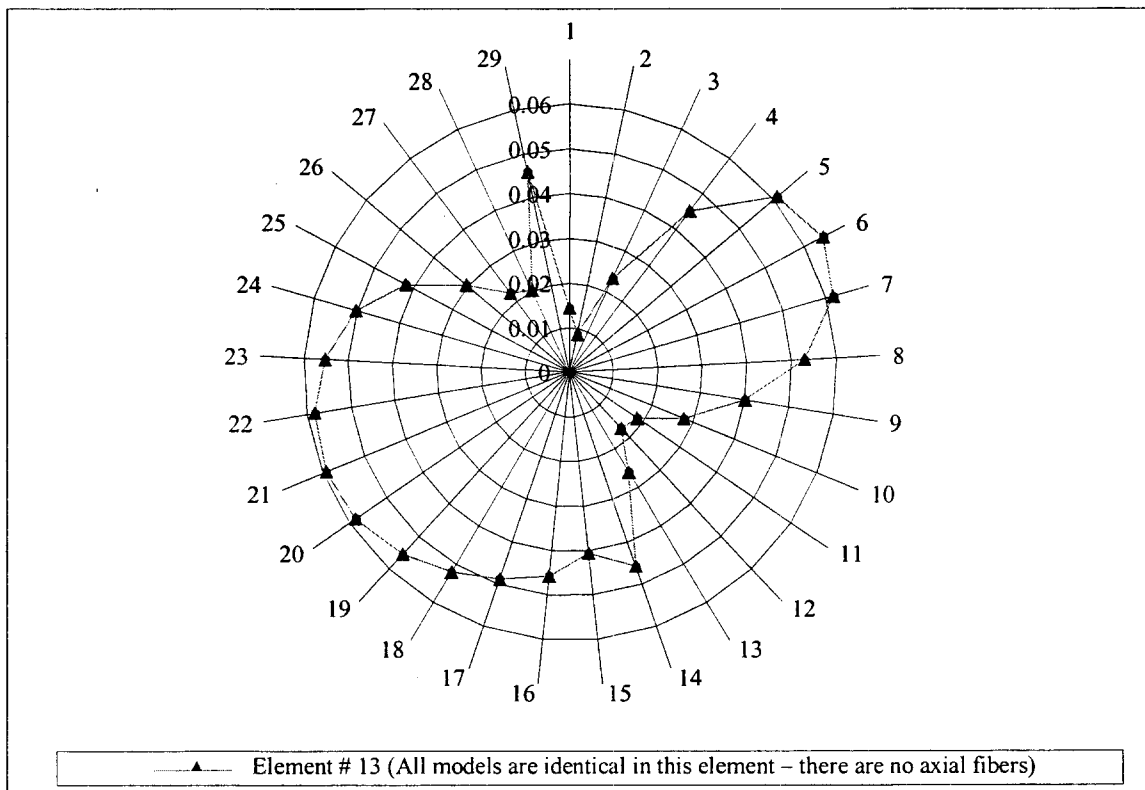


Figure 3.26 **Top:** Failure Index along the "Brick Elements" for Blade Element No. 13 (Station 288 - 312) for the Baseline Configuration, which is identical to the twist-coupled configurations at this element. (Wind Load is 70 m/s; C_d is 1.7).
Bottom: Brick Element Numbering for Blade Element No. 13.

Chapter 4

Cost Estimates for the SERI-8 Blade

Cost has been a significant factor in composite designs and their manufacture. Cost reduction can be achieved by improving the design, manufacturing processes, equipment, and materials systems.

Glass fibers are the traditional basis material used in wind turbine blade design and manufacturing. As discussed in previous chapters, tremendous improvements in blade performance (such as reduction in weight, skin thickness, higher bend-twist coupling) can be achieved, if carbon fibers are introduced. However, carbon fibers generally cost at least ten times more than glass fibers. The perception that high material cost will lead to an increase in total cost keeps blade manufacturers from using carbon fibers. However, there is often a net cost savings in spite of the high carbon fiber material cost because the labor cost is reduced in laminating when fewer fibers are required.

In evaluating manufacturing costs for wind turbine blades, we looked at recurring costs, those associated with all elements in manufacturing a part. These costs are linked closely to design and manufacturing processes and provide criteria for evaluating cost effectiveness of these parameters. Recurring costs include factory fabrication labor, support labor functions for engineering, quality control, tooling, manufacturing engineering, and graphic services; production and support material; indirect charges such as labor and material overhead and general and administrative costs [8], illustrated in Figure 4.1. This particular recurring cost structure was developed by an aerospace manufacturer, Northrop Corporation, and so not all fabrication functions are applicable to blade manufacturing. We focus here on material cost and factory fabrication labor cost.

Factory fabrication labor is the direct effort required to transform raw material into the final composite part, and has four major steps: lay-up, core operation, part consolidation, and finishing.

In a comparison between glass composite and hybrid (carbon and glass fibers) composite, we looked at three cases: (a) material only for a general composite part, (b)

material and labor costs for a general composite part, and (c) the estimate for a SERI-8 blade.

4.1 Cost Estimate I (Materials Only)

The criterion for comparing costs between a hybrid and glass composite is to maintain the same axial stiffness (it is easy to derive some ratio parameters using this criterion). We assume that glass fibers cost \$1.2/lb., carbon fibers cost \$13/lb, and the labor rate is assumed to be \$65/hour [9]. The material properties for glass and carbon prepreg materials are given in Table 2.1 and 3.1 respectively. Two important ratios, weight ratio and cost ratio, can be derived.

The weight ratio is defined as the weight of the hybrid composite divided by that of an all-glass composite. The weight ratio is given by

$$\text{Weight_Ratio} = \frac{1 + \text{Vol}_{c/g} \cdot S_c / S_g}{1 + \text{Vol}_{c/g} \cdot E_c / E_g} \quad (4.1)$$

The parameters in Equation (4.1) are defined as follows: S, specific gravity; E, axial modulus; subscript 'g', glass; subscript 'c', carbon; and $\text{Vol}_{c/g}$, carbon/glass volume ratio.

The cost ratio is the ratio of the cost of the hybrid composite to the all-glass composite to provide the same axial stiffness. This is given by

$$\text{Cost_Ratio_I} = \frac{1 + \text{Vol}_{c/g} \cdot S_c / S_g \cdot \$c / \$g}{1 + \text{Vol}_{c/g} \cdot E_c / E_g} \quad (4.2)$$

The additional parameter, \$, is cost per unit weight.

The two variables in the cost and weight ratios are the volume ratio, $\text{Vol}_{c/g}$, and material cost ratio. If we assume that the material cost ratio is about 10 to 1 (\$13/lb for carbon and \$1.2/lb for glass), then we can tabulate both the cost ratio and weight ratio as a function of volume ratio of a hybrid composite (see Table 4.1). Thus, an all-carbon fiber composite costs 2.4 times more than an all-glass fiber composite to provide the same axial stiffness at a weight savings of 78%. When a 50/50 glass/carbon hybrid

composite is used, however, the calculated cost ratio is about 2.1 and the weight savings is 62%.

4.2 Cost Estimate II (Material and Labor Costs)

In Cost Estimate I, we assume no or negligible labor costs. This assumption ultimately limits the use of carbon fibers in any composite part design and fabrication if cost is the only design parameter. In fact, we can't produce any composite part without incurring labor costs, which are the labor hours required to produce a composite part and labor rate (\$/hour, \$65/hour is assumed here). The hours depend on the types of manufacturing processes involved. Processes such as pultrusion, filament winding and automatic tape layup reduce the recurring labor hours significantly, although capital investment is huge. Thus, one major strategy to reduce the cost of advanced composites is to automate the labor-intensive activities. In this discussion, we focus on the hand lay-up process, which dominates in blade manufacturing.

The cost ratio, which includes both material and labor costs, is the ratio of the cost of the hybrid composite to the all-glass composite with the same axial stiffness. It is given by

$$\text{Cost_Ratio_II} = \frac{L_h + M_h}{L_g + M_g} = \frac{\frac{L_h}{L_g} \cdot \frac{L_g}{M_g} + \frac{M_h}{M_g}}{\frac{L_g}{M_g} + 1} \quad (4.3)$$

The parameters in Equation (4.3) are defined as follows: L, labor cost; M, material cost; and subscript 'h', hybrid composite. The ratio, M_h/M_g , is similar to the Cost_Ratio_I as defined in Equation (4.2). L_g/M_g is the ratio of labor cost to the material cost of the all-glass composite. L_h/L_g is the ratio of the labor cost of the hybrid composite to the all-glass composite to provide the same axial stiffness. In the hand lay-up process, the labor cost is a function of labor rate, lay-up hours per layer, and total number of layers of plies. We further assume that the labor rate and the lay-up hours per layer are the same regardless of the types of materials used, and that the thickness of the ply layers is equal for glass and carbon fibers (see Table 2.1). The labor cost ratio is then given by

$$\text{Labor_Cost_Ratio} = \frac{L_h}{L_g} = \frac{1 + \text{Vol.}_{c/g}}{1 + \text{Vol.}_{c/g} \cdot \frac{E_c}{E_g}} \quad (4.4)$$

If we substitute Equation (4.4) into (4.3), the two variables in the Cost_Ratio_II are the volume ratio, $\text{Vol.}_{c/g}$, and the labor/material cost ratio. We assume that the material cost ratio is about 10 to 1 (\$13/lb for carbon and \$1.2/lb for glass); we tabulate a cost matrix in terms of volume ratio of the hybrid composite and labor/material cost ratio of the all-glass composite (see Table 4.2). Thus, an all-carbon fiber composite still costs 2.4 times more than an all-glass fiber composite *when material cost is the only element* to provide the same axial stiffness at a weight savings of 78 %. However, the all-carbon fiber composite will cost four times less than the all-glass fiber composite *when labor cost is the only element*. A cost savings (shown as italic print in Table 4.2) is achievable for a hybrid composite with any carbon/glass volume ratio if the labor cost is about 70% and the material cost is about 30% of the total (material and labor).

4.3 Cost Estimate of SERI-8 Enhanced Blade Models

Sections 4.1 and 4.2 described two simple methods to compare the cost ratio of a hybrid composite to an all-glass composite to maintain the same axial stiffness. The cost ratios are functions of parameters such as labor/material in the all-glass composite, the material in the hybrid composite and the all-glass composite, etc. The cost ratios do not reflect absolute production costs, which are driven by design, production volume, and how the process is run.

To estimate the absolute cost of the SERI-8 blade, we utilized some functions of the cost model developed for advanced composites manufacturing for the aerospace industry by Northrop Corporation [8] for the Air Force. The cost structure for the model is shown in Figure 4.1; however, not all the elements are applicable to a blade manufacturing process. We used two elements, direct material cost and direct labor cost for factory fabrication, to evaluate the multiple design options for the SERI-8 enhanced models. As mentioned earlier, the factory fabrication process has four steps but because the SERI-8 models do not include core as one of the design variables, core operation is not evaluated here. Part consolidation and finishing costs, which are related to the size of

the composite part, are applicable to all the SERI-8 models, since all models have the same external dimensions.

The estimated factory labor hours to ensure various SERI-8 enhanced models have the same flapping stiffness for all models are shown in Figure 4.2. The estimated factory labor hour for the baseline all-glass model is about 400 hours, and the estimated factory labor hour for the configuration with 100% unidirectional fiber glass replacement to carbon fibers (about 65% carbon content, see Table 4.3) is about 140 hours. The cost ratio reference to the baseline model is about 1 to 3; this ratio assumed that labor is the only element in total cost and assumed a fixed labor rate. The cost ratio (1/3) is close to the estimated value of 1/4, which is for an all-carbon composite using Equation 4.4.

We found out from TPI, a blade manufacturer, that their estimate of labor hour would be about 40 hours. In order to reflect TPI's estimated hours for fabricating a blade, we scaled the estimated lay-up hours by a factor of 10 for the baseline and all enhanced models. The scaling results in reduction of factory labor for the all-glass SERI-8 blade from 400 hours to 45 hours (there is no scaling applied to the hours for consolidation and finish operations), which is close to TPI's estimate. Although the labor scaling does not affect the cost ratio, it affects the ratio of labor cost to material cost. Before the adjustment, the labor for the all-glass SERI-8 blade (baseline model) was the major cost as compared to the material cost. Thus, we see a huge cost reduction for all SERI-8 enhanced models, as shown in Figure 4.4. The adjustment effectively reduces the labor cost by a factor of 10 and makes material a significant cost parameter. Consequently, not all enhanced models reduce fabrication cost. The enhanced models that cost less than the all-glass model have less than 15% volume of carbon (see Table 4.3 and Figure 4.5) in the blade design.

In private correspondence with James Tangler, the originator of the SERI blade series, we obtained some previously unpublished blade-cost information. The information shown in Table 4.4, is for fabricating a SERI 9.7 meter blade, which is very similar to the SERI-8 blade. Tangler indicated that the cost ratio of labor to material for an all-glass blade should be about 50/50 (for SERI-9.7 meter blade, the ratio is about 40/60 as shown in Table 4.4). Therefore, there is no cost advantage for carbon fibers in a blade design with this labor to material cost ratio, as indicated in Table 4.2. Tangler also stated that the

labor hours were 84 hours. He does not have an exact labor rate but his reasonable guess would be about \$10 an hour which results in about 84 hours of labor. We used labor rate of \$65/hour [9] in the cost study. If a labor rate of \$65/hour were used for the estimate, the cost ratio of labor to material for the SERI 9.7 meter blade would be 80/20; and we would have different deduction on cost advantage of fabrication of a hybrid blade.

In summary, labor and materials are two main elements in total cost. The proportion of labor to material in the total cost affects the volume of carbon fibers allowed in a wind turbine blade design if cost is the only design variable.

Table 4.1 The Ratios of Carbon-Enhanced/Baseline Blade Weight and Cost as a Function of Volume Ratio of Glass and Carbon when only Material Costs are Considered

Glass_Vol./Carbon_Vol.	Weight_Ratio	Weight_Saving	Cost_Ratio
0/100	0.22	77.8%	2.41
10/90	0.24	75.7%	2.37
20/80	0.27	73.2%	2.32
30/70	0.30	70.3%	2.27
40/60	0.33	66.7%	2.21
50/50	0.38	62.2%	2.13
60/40	0.43	56.6%	2.02
70/30	0.51	49.1%	1.89
80/20	0.61	38.9%	1.70
90/10	0.76	23.9%	1.43
100/0	1.00	0.0%	1.00

Table 4.2 Estimated Total Cost Ratios for a Matrix of Labor/Material Cost Splits and Glass/Carbon Fiber Content

		Glass_Vol. / Carbon_Vol.										
		100/0	90/10	80/20	70/30	60/40	50/50	40/60	30/70	20/80	10/90	0/100
Labor_Cost / Material_Cost of the All-Glass Composite	0/100	1.00	1.43	1.70	1.89	2.02	2.13	2.21	2.27	2.32	2.37	2.41
	10/90	1.00	1.37	1.60	1.75	1.87	1.95	2.02	2.08	2.12	2.16	2.19
	20/80	1.00	1.30	1.49	1.62	1.71	1.78	1.84	1.88	1.92	1.95	1.98
	30/70	1.00	1.23	1.38	1.48	1.55	1.61	1.65	1.69	1.72	1.74	1.76
	40/60	1.00	1.17	1.27	1.34	1.40	1.44	1.47	1.49	1.51	1.53	1.54
	50/50	1.00	1.10	1.16	1.21	1.24	1.26	1.28	1.30	1.31	1.32	1.33
	60/40	1.00	1.03	1.06	1.07	1.08	1.09	1.10	1.10	1.11	1.11	1.11
	70/30	1.00	0.97	0.95	0.94	0.93	0.92	0.91	0.91	0.90	0.90	0.90
	80/20	1.00	0.90	0.84	0.80	0.77	0.75	0.73	0.71	0.70	0.69	0.68
	90/10	1.00	0.84	0.73	0.66	0.61	0.57	0.54	0.52	0.50	0.48	0.47
	100/0	1.00	0.77	0.63	0.53	0.45	0.40	0.36	0.32	0.29	0.27	0.25

Table 4.3 Averaged Volume of Carbon Fibers in Various SERI-8 Enhanced Models

SERI-8 Enhanced Models	Averaged Carbon Volume	Remarks
20% Unidirectional Glass Fiber Replacement	5%	Reference to Table 3.2
40% Unidirectional Glass Fiber Replacement	14%	Reference to Table 3.3
60% Unidirectional Glass Fiber Replacement	25%	Reference to Table 3.4
80% Unidirectional Glass Fiber Replacement	40%	Reference to Table 3.5
100% Unidirectional Glass Fiber Replacement	65%	Reference to Table 3.6

Table 4.4 Breakdown of Cost Elements of a SERI 9.7-meter Blade

Description	Cost
Labor	\$840
Raw Materials (fibers and resin)	\$1200
Flanges and Root Units	\$800
Spoiler Tip Mechanism	\$1000
Overhead	\$875
Total	\$4715

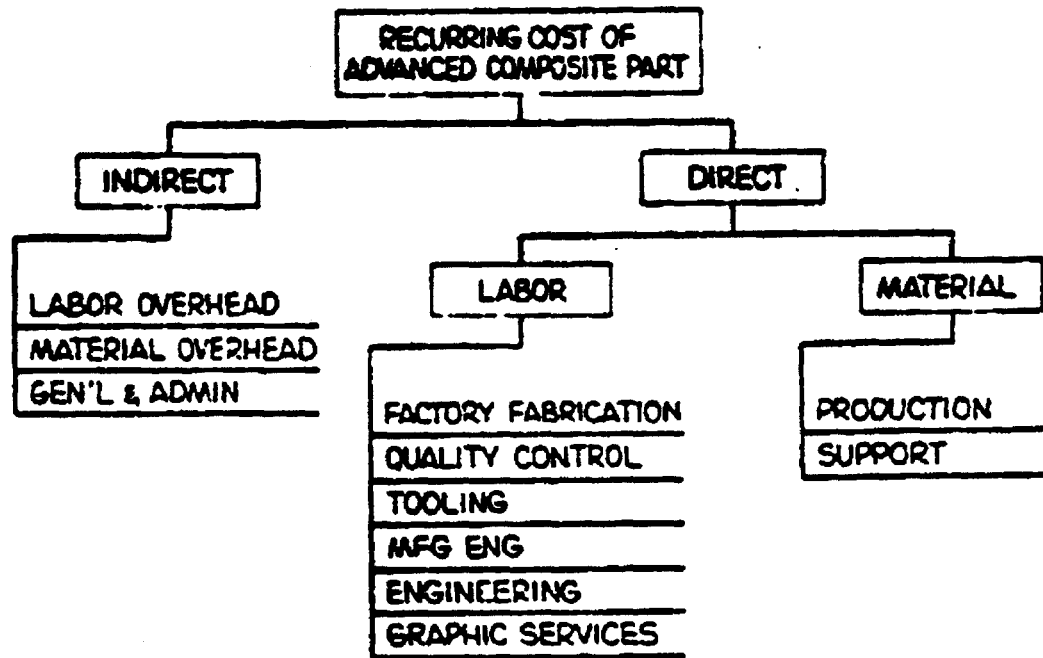


Figure 4.1 Recurring Cost of Advanced Composite Part [8]

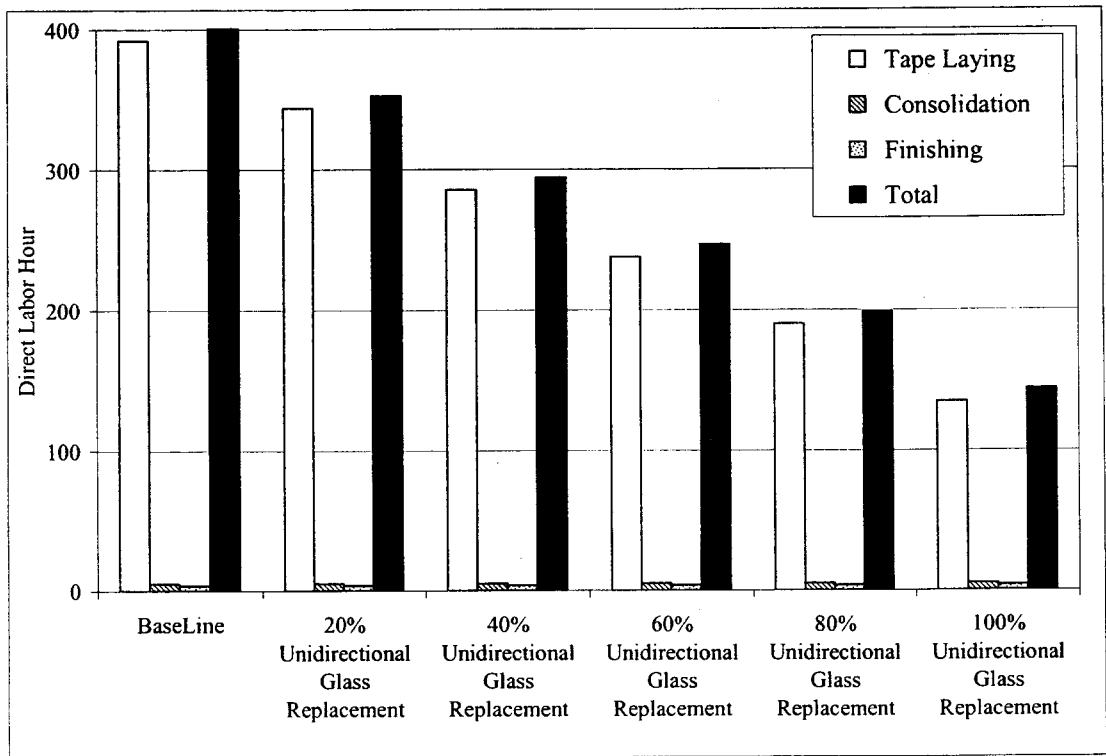


Figure 4.2 Factory Labor Hours for Various SERI-8 Enhanced Model Configurations Estimated by Northrop's Model [8]

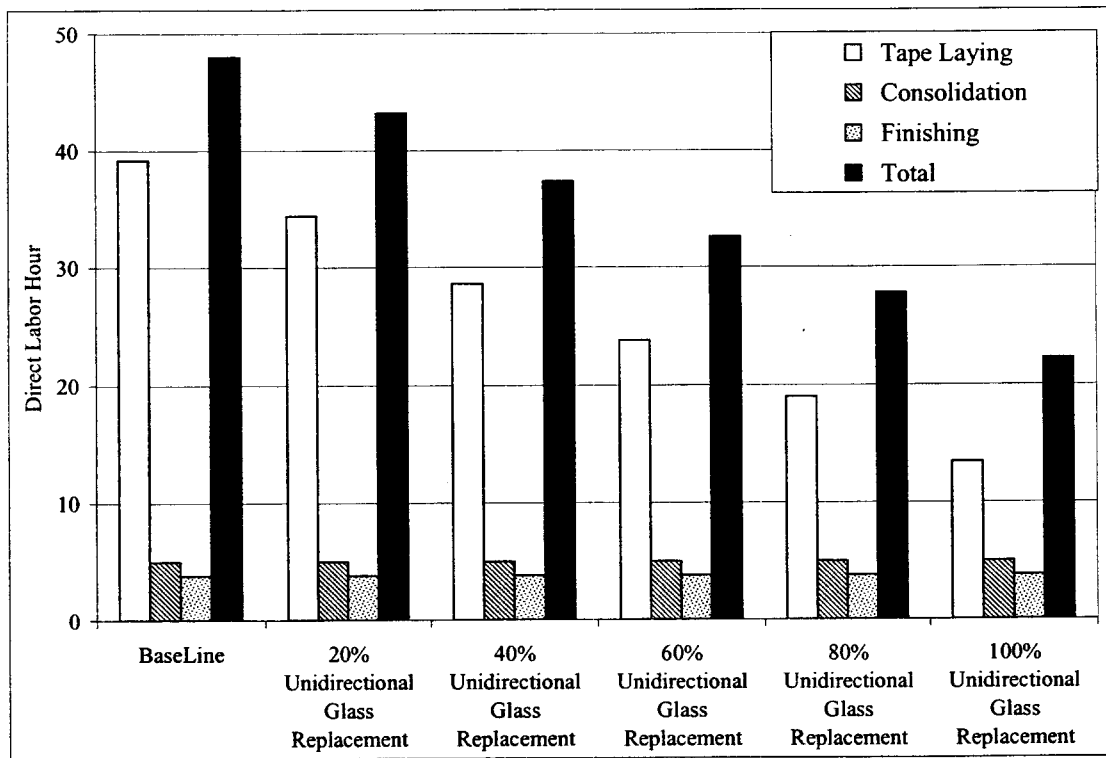


Figure 4.3 Scaled Factory Labor Hours for Various SERI-8 Enhanced Model Configurations after Applying Scale Factor of 10 to Layup Hours to Reflect TPI's Blade Fabrication Hours for All-Glass Blade

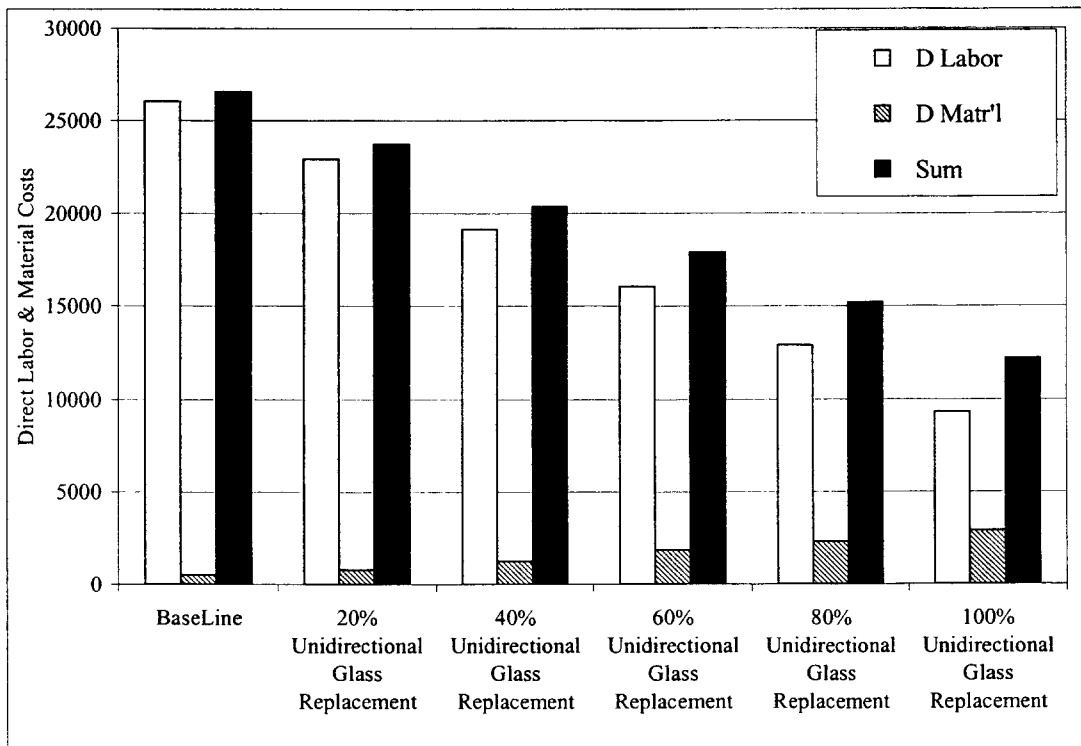


Figure 4.4 Factory Labor, Material and Total Costs for Various SERI-8 Enhanced Model Configurations Based Only on the Northrop Model

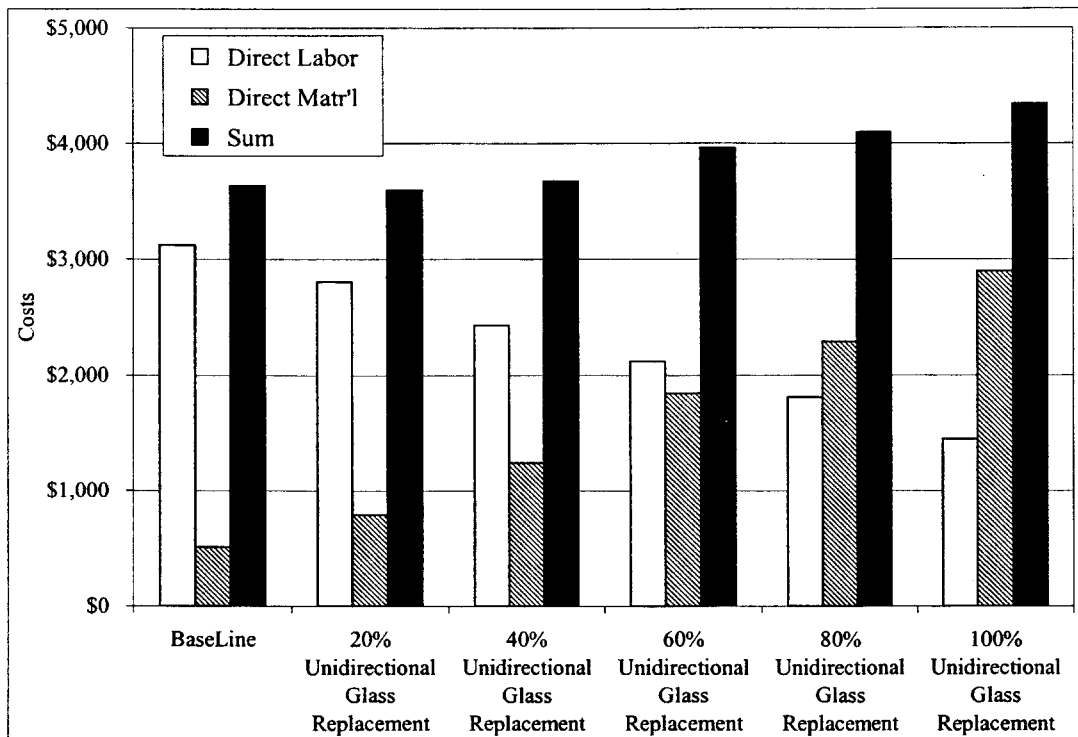


Figure 4.5 Factory Labor, Material and Total Costs for Various SERI-8 Enhanced Model Configurations after Applying Scale Factor of 10 to Layup Hours to Reflect TPI's Blade Fabrication Hours for All-Glass Blade

Chapter 5

Conclusions

We generated a baseline model of the SERI-8 blade validated it against an available database and test results. The validation is satisfactory. Enhanced models that incrementally replaced unidirectional glass fibers with carbon fibers were designed to study the cost and structural performance if carbon fibers were used in wind turbine blade design. The criterion used for all the enhanced models, which have options with and without bend-twist-coupled design, was to have the same flapping stiffness as the baseline model. The study indicates that increased use of carbon fibers in blade design improves blade performance because of reducing weight and skin thickness. A factor of safety of 2 is still achieved for an enhanced model with all unidirectional glass fibers replaced by 20° off-axis carbon fibers subjected to severe wind speed of 70 meters/second.

In this study, a hand lay-up process was chosen to evaluate a cost comparison between the hybrid blade and the all-glass blade. If material is the sole element used in total cost, it is expensive to produce an all-carbon blade. However, in the other extreme where labor is the main element, there is tremendous savings in using carbon fibers in blade design. Northrop's model for the SERI-8 blade, after applying a scale factor of 10 to the lay-up hours to be close to the labor hours estimated by TPI, indicates that the cost of some enhanced models is comparable to that of the all-glass model. For the specific design and manufacturing assumptions in this study, cost savings is achievable if the carbon volume is less than 15% (Figure 4.5). In general, the cost advantage or disadvantage of carbon fiber replacement will depend on the cost ratio of labor to materials as shown in Table 4.2. It was assumed that labor costs are proportional to the number of layers of plies and that glass and carbon laminate thickness are equal (Table 2.1). Generic conclusions can therefore not be drawn because of this dependence on the

specifics of the design and manufacturing process. However, sufficient data are provided here to estimate the potential for carbon fibers use in a specific blade manufacturing application.

References

1. J. Tangler, B. Smith, D. Jager, T. Olsen, "SERI Thin Airfoil Blade Atmospheric Performance Test: Final Results," *Windpower '90 Proceedings*, Washington D.C., September 1990, pp. 118-125.
2. Bob Keller & Brian Smith, *SERI 7.9 meter Thin Airfoil Blade Pitch Data Sheet*, Solar Energy Research Institute, Golden, Colorado, April 1990.
3. Kevin Jackson, *SERI 8 Meter Advanced Technology Wind Turbine Blade*, HL-9-18129-1, Dynamic Design, Davis, California, 1989.
4. Scott M. Klingenstein, "Analytical Prediction and Experimental Verification of Blade Loads Experienced by Two, Three-Bladed, Fixed Pitch, Horizontal Axis Wind Turbines," Master Thesis, University of North Dakota, 1989.
5. S. Sinh, *Composite Blade Model and Software*, XAF-4-14076-01, Stanford University, September 1997.
6. Phil Mosher, Data Sheets from TPI on glass/epoxy, Tillison Pearson Incorporated, Warren, Rhode Island, January 20, 1999.
7. C.H. Ong & S.W. Tsai, *Design, Manufacture and Testing of A Bend-Twist D-spar*, SAND99-1324, Sandia National Laboratories, Albuquerque, New Mexico, June 1999.
8. Donald J. LeBlanc, *Advanced Composites Cost Estimating Manual (ACCEM)*, AFFDL-TR-76-87, Northrop Corporation, Hawthorne, California, August 1976.
9. Timothy G. Gutowski, *Advanced Composites Manufacturing*, New York, John Wiley & Sons, Inc., 1997.

DISTRIBUTION

T. Almeida
TPI Inc.
225 Alexander Road
Portsmouth, RI 02871

H. Ashley
Dept. of Aeronautics and
Astronautics Mechanical Engr.
Stanford University
Stanford, CA 94305

B. Bell
Zond Energy Systems Inc.
13681 Chantico Road
P.O. Box 1970
Tehachapi, CA 93561

C. P. Butterfield
NREL
1617 Cole Boulevard
Golden, CO 80401

G. Bywaters
Northern Power Technology Co.
Box 999
Waitsfield, VT 05673

J. Cadogan
U.S. Department of Energy
Office of Geothermal & Wind
Technology
EE-12
1000 Independence Avenue SW
Washington, DC 20585

D. Cairns
Montana State University
Mechanical & Industrial Engineering Dept.
Bozeman, MT 59717

S. Calvert
U.S. Department of Energy
Office of Geothermal & Wind
Technology
EE-12
1000 Independence Avenue SW
Washington, DC 20585

J. Chapman
OEM Development Corp.
840 Summer St.
Boston, MA 02127-1533

C. Christenson
Zond Energy Systems Inc.
13681 Chantico Road
PO Box 19070
Tehachapi, CA 93561

R. N. Clark
USDA
Agricultural Research Service
P.O. Drawer 10
Bushland, TX 79012

J. Cohen
Princeton Economic Research, Inc.
1700 Rockville Pike
Suite 550
Rockville, MD 20852

C. Coleman
Northern Power Technology Co.
Box 999
Waitsfield, VT 05673

K. J. Deering
The Wind Turbine Company
515 116th Avenue NE
No. 263
Bellevue, WA 98004

A. J. Eggers, Jr.
RANN, Inc.
744 San Antonio Road, Ste. 26
Palo Alto, CA 94303

P. R. Goldman
Director
Office of Geothermal and
Wind Technology
EE-12
U.S. Department of Energy
1000 Independence Avenue
Washington, DC 20585

G. Gregorek
Aeronautical & Astronautical Dept.
Ohio State University
2300 West Case Road
Columbus, OH 43220

C. Hansen
Windward Engineering
4661 Holly Lane
Salt Lake City, UT 84117

C. Hiel
W. Brandt Goldsworthy & Assoc.
23930-40 Madison Street
Torrance, CA 90505

S. Hock
Wind Energy Program
NREL
1617 Cole Boulevard
Golden, CO 80401

K. Jackson
Dynamic Design
123 C Street
Davis, CA 95616

G. James
University of Houston
Dept. of Mechanical Engineering
4800 Calhoun
Houston, TX 77204-4792

M. Kramer
Foam Matrix, Inc.
PO Box 6394
Malibu, CA 90264

A. Lucero
Librarian
National Atomic Museum
Albuquerque, NM 87185

R. Lynette
Springtyme Co.
212 Jamestown Beach Lane
Sequim, WA 98382

D. Malcolm
Global Energy Concepts, LLC
5729 Lakeview Drive NE
Suite 100
Kirkland, WA 98033

J. F. Mandell
Montana State University
302 Cableigh Hall
Bozeman, MT 59717

T. McCoy
Global Energy Concepts, LLC
5729 Lakeview Drive NE
Suite 100
Kirkland, WA 98033

R. N. Meroney
Dept. of Civil Engineering
Colorado State University
Fort Collins, CO 80521

P. Migliore
NREL
1617 Cole Boulevard
Golden, CO 80401

A. Mikhail
Zond Energy Systems, Inc.
13681 Chantico Road
PO Box 1970
Tehachapi, CA 93561

E. Moroz
Zond Energy Systems, Inc.
13681 Chantico Road
PO Box 1970
Tehachapi, CA 93561

W. Musial
NREL
1617 Cole Boulevard
Golden, CO 80401

NWTC Library (5)
NREL
1617 Cole Boulevard
Golden, CO 80401

V. Nelson
Department of Physics
West Texas State University
WT P.O. Box 60248
Canyon, TX 79016

Cheng-Huat Ong
Stanford University
Dept. of Aeronautics & Astronautics
Stanford CA 94305-4035

L. Pratsch
U.S. Department of Energy
Office of Geothermal & Wind Technology
EE-12
1000 Independence Avenue SW
Washington, DC 20585

R. G. Rajagopalan
Aerospace Engineering Department
Iowa State University
404 Town Engineering Bldg.
Ames, IA 50011

Michael Robinson
NREL
1617 Cole Boulevard
Golden, CO 80401

D. Sanchez
U.S. Dept. of Energy
Albuquerque Operations Office
P.O. Box 5400
Albuquerque, NM 87185

L. Schienbein
CWT Technologies, Inc.
4006 S. Morain Loop
Kennewick, WA 99337

R. Sherwin
Atlantic Orient
PO Box 1097
Norwich, VT 05055

Brian Smith
NREL
1617 Cole Boulevard
Golden, CO 80401

K. Starcher
AEI
West Texas State University
WT P.O. Box 60248
Canyon, TX 79016

F. S. Stoddard
Dynamic Design
P.O. Box 1373
Amherst, MA 01004

A. Swift
University of Texas at El Paso
Department of Mech. & Ind. Engineering
El Paso, TX 79968

R. W. Thresher
NREL
1617 Cole Boulevard
Golden, CO 80401

S. Tsai (5)
Stanford University
Dept. of Aeronautics & Astronautics
Stanford University
Stanford, CA 94305-4035

W. A. Vachon
W. A. Vachon & Associates
P.O. Box 149
Manchester, MA 01944

B. Vick
USDA, Agricultural Research Service
P.O. Drawer 10
Bushland, TX 79012

L. Wendell
2728 Enterprise Dr.
Richland, WA 99352

R. E. Wilson
Mechanical Engineering Dept.
Oregon State University
Corvallis, OR 97331

S. R. Winterstein
Civil Engineering Department
Stanford University
Stanford, CA 94305

M. Zuteck
MDZ Consulting
931 Grove Street
Kemah, TX 77565

M.S. 0555 B. Hansche, 9122
M.S. 0557 T. J. Baca, 9125
M.S. 0557 T. G. Carne, 9124
M.S. 0708 H. M. Dodd, 6214 (25)
M.S. 0708 T. D. Ashwill, 6214
M.S. 0708 D. E. Berg, 6214
M.S. 0708 P. L. Jones 6214
M.S. 0708 D. L. Laird, 6214
M.S. 0708 M. A. Rumsey, 6214
M.S. 0708 H. J. Sutherland, 6214
M.S. 0708 P. S. Veers, 6214
M.S. 0708 J. R. Zayas, 6214
M.S. 0847 D. W. Lobitz, 9125
M.S. 0847 D. R. Martinez, 9124
M.S. 0612 Review & Approval Desk, 4912
M.S. 0899 Technical Library, 4916 (2)
M.S. 9018 Central Technical Files, 8940-2