

**Composite Repair Methods for Steel Pipes**  
Research Activities Summary Report

by

Dr. Ozden O. Ochoa, Professor  
Department of Mechanical Engineering  
Texas A&M University

Chris Alexander  
Stress Engineering Services, Inc.

**Supplemental Project Report**  
**Prepared for the Minerals Management Service**  
**Under the MMS/OTRC Cooperative Research Agreement**  
**1435-01-04-CA-35515**  
**Task Order 39300**  
**MMS Project Number 558**

**June, 2007**

OTRC Library Number: 6/07CPSR

“The views and conclusions contained in this document are those of the authors and should not be interpreted as representing the opinions or policies of the U.S. Government. Mention of trade names or commercial products does not constitute their endorsement by the U. S. Government”.



*For more information contact:*

**Offshore Technology Research Center**  
Texas A&M University  
1200 Mariner Drive  
College Station, Texas 77845-3400  
(979) 845-6000

or

**Offshore Technology Research Center**  
The University of Texas at Austin  
1 University Station C3700  
Austin, Texas 78712-0318  
(512) 471-6989

*A National Science Foundation Graduated Engineering Research Center*

## TABLE OF CONTENTS

TABLE OF CONTENTS.....	i
LIST OF TABLES AND FIGURES.....	ii
JIP ON COMPOSITE REPAIR: STATE OF THE ART ASSESSMENT.....	1
STATE OF THE ART ASSESSMENT OVERVIEW .....	2
TECHNICAL DETAILS OF THE TEST PROGRAM.....	4
Pressure only test .....	4
Pressure-tension test.....	6
Pressure-tension-bending test .....	7
PRESENTATION OF RESULTS .....	10
Pressure-only Test.....	10
Pressure-tension Test .....	13
Pressure-tension-bending Test.....	15
GENERAL OBSERVATIONS ON THE JIP TEST RESULTS.....	18

## LIST OF TABLES AND FIGURES

Table 1 – Summary of test results relative to design conditions .....	19
Figure 1 – Schematic diagram showing pressure only test sample .....	5
Figure 2 – Location of strain gages on the pressure and pressure/tension samples.....	5
Figure 3 – Schematic diagram showing pressure-tension test sample.....	7
Figure 4 – Four point bending configuration for pressure-tension-bend testing .....	8
Figure 5 - Location of strain gages on the pressure-tension-bend samples .....	8
Figure 6 – Load frame used for pressure-tension-bend testing.....	9
Figure 6 – Test results from pressure-only testing.....	12
Figure 7 – Failure in unrepaired test sample.....	12
Figure 8 – Failure in burst sample using Product C.....	13
Figure 9 – Test results from pressure-tension testing .....	14
Figure 10 – Post-failure photos of Product D pressure-tension test .....	15
Figure 11 – Test results from pressure-tension-bending testing.....	17
Figure 12 – Photo showing Product C prior to bend testing.....	17

## **JIP ON COMPOSITE REPAIR: STATE OF THE ART ASSESSMENT**

MMS sponsored a research program starting in 2006 with the Offshore Technology Research Center (OTRC) to assess existing composite repair technology. The conventional method for performing a state of the art assessment involves surveying industry and manufacturers about their use of a particular technology. Although this is a valid preliminary approach, it has the potential for failing to capture any of the deficiencies in an existing technology and the requirements for improving an existing technology. For this reason, a Joint Industry Program (JIP) was formed that involved the evaluation of four different composite repair systems using a full-scale test program. The program incorporated 8.625-inch x 0.406-inch, Grade X46 pipe test samples that were fitted with simulated corrosion by machining. The program involved destructively testing three samples repaired by each respective composite repair system. The three tests included a burst test (increasing pressure to failure), a tension to failure test (pressure with increasing axial tension loads to failure), and a four-point bend test (pressure and tension held constant with increasing bending loads to achieve significant yielding in steel).

It should be recognized that the primary purpose of the JIP study was to identify and confirm the critical elements required for an effective composite repair. Having practically unlimited access to manufacturers with the ability to understand the overall mechanics of each repair, the author was provided with insights useful for developing an optimized repair system. Other benefits were also derived in the execution of the program, including the development of guidelines for industry and regulators and providing the manufacturers with the opportunity to assess their repair systems relative to loading conditions associated with offshore risers.

## STATE OF THE ART ASSESSMENT OVERVIEW

The primary focus of the work to date has been on the repair of onshore pipelines where hoop stress is dominant. However, offshore risers experience not only hoop stress due to internal pressure, but are subject to tensile and bending loads. To date there has been no single study directed at assessing composite repair technology subject to offshore riser loads. As a result, a four-team JIP was formed. Each repair system was evaluated considering a combination of pressure, tension, and bending loads.

Four companies participated in the JIP study. To maintain anonymity, each company was assigned a letter reference designation as noted below.

**Product A** – this system uses an E-glass fiber system in a water-activated matrix. The cloth is a balanced weave with orthogonal fibers aligned at 0 and 90 degrees relative to the axis of the pipe. During installation, the cloth was oriented either axially or circumferentially to achieve the desired level of reinforcement.

**Product B** – this system uses an E-glass fiber system in a water-activated matrix. The cloth is a balanced weave with orthogonal fibers aligned at 0 and 90 degrees relative to the axis of the pipe. This particular repair involved using an epoxy filler material in the corroded region, as opposed to placing composite material in this region of the repair. All of the other manufacturers chose to install fibers in the corroded region. During installation, the cloth was oriented either axially or circumferentially to achieve the desired level of reinforcement.

**Product C** – this system uses a carbon fiber system in an epoxy matrix. The cloth is a stitched fabric with all uniaxial fibers. During installation, the fibers were aligned at 0 and 90 degrees relative to the axis of the pipe to achieve the desired level of reinforcement.

**Product D** – this system uses an E-glass fiber system in an epoxy matrix. The cloth has fibers that are oriented at , 0, 90, and +/- 45 degrees. Additionally, a layer of chopped strand fibers is sprayed on the underside of the cloth. During installation, the cloth was oriented either axially or circumferentially to achieve the desired level of reinforcement.

Because of the lack of available performance data on composite repairs subject to tension and bending loads, the need for integrating these load types was identified. Additionally, discussions with participating manufacturers focused on the need to ensure that their repair systems would be designed in a manner that could provide adequate reinforcement in terms of both bonding to the pipe and also providing sufficient bending rigidity to reinforce the corroded section of pipe. Fundamentally, bonding to the pipe involves shear strength of the adhesive (or resin used in fabricating the composite) as well as available shear area. In other words, even with a strong adhesive, shear failure is likely if an inadequate bond area is present. In terms of bending rigidity, the manufacturers were encouraged to integrate a sufficient percentage of fibers in the axial direction. This required additional consideration for all participants as their systems have preferential orientations directed at circumferential reinforcement. The problem of having insufficient fibers in the axial direction was resolved by rotating a certain percentage of the fabric during installation to align with the axis of the pipe.

As will be shown in the following sections, by and large the manufacturers were able to use their existing hoop-dominated repair systems with slight modifications to achieve acceptable reinforcement for the imposed riser loads. This is an important observation as the key to repairing damaged structures is to first identify the potential load conditions and then design a repair system that adequately reinforces against the anticipated loads. It is also important to note the role that quality installation plays in the success of a composite repair system.

## TECHNICAL DETAILS OF THE TEST PROGRAM

A specific test program was devised to evaluate the performance of the repair systems subject to internal pressure, tension, and bending loads. To provide greater clarity in assessing the performance of a particular load type (i.e. pressure, tension, or bending), three specific tests were developed to decouple interactions between the three possible load types. Details are provided in the sections that follow.

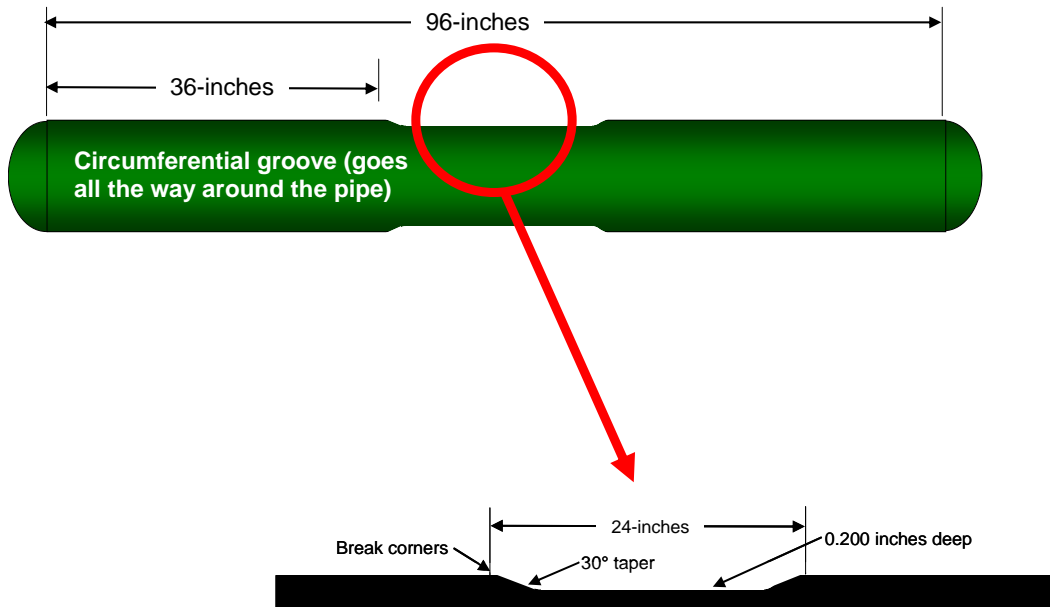
A final point concerns the geometry for each of the repairs. Recognizing the potential for significant variability in the repair system developed by each manufacturer, each manufacturer was told that the limit on axial length of the repair was 60 inches. This length ensures that an 18-inch length of repair extends on both sides of the 24-inch long corrosion section. Additionally, all manufacturers were told that each of their particular repairs on the three test samples had to be identical. This ensured that the only variation between tests was the type of loading, and not the repair itself. Prior to installation of the repair systems, each pipe was sandblasted to near white metal to ensure a quality adhesive bond between the steel and composite materials.

### **Pressure only test**

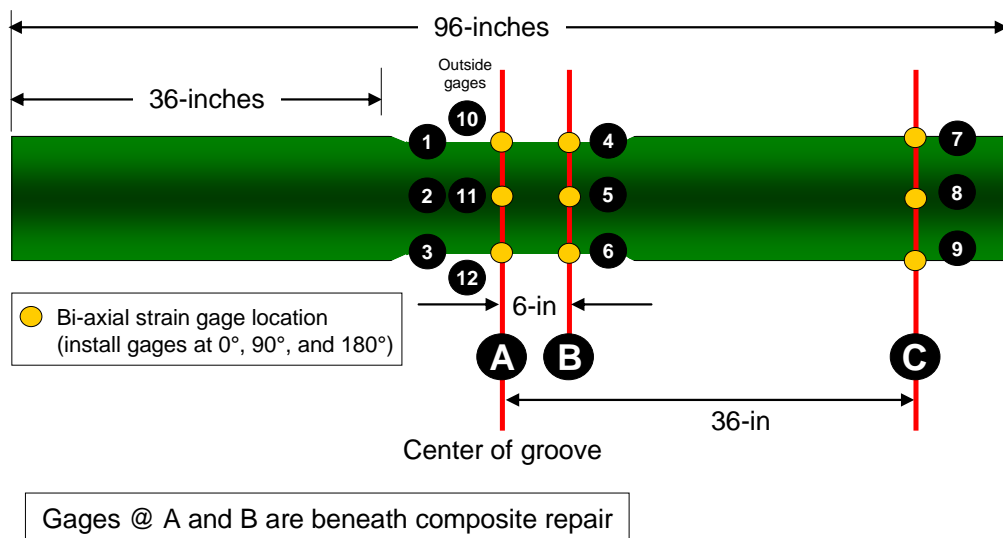
As the name implies, the purpose of this test type is to assess the performance of the composite repair in providing hoop strength. **Figure 1** is a schematic showing the unrepaired sample geometry. As noted, an axisymmetric groove is machined in the center of the 8-ft long sample to simulate corrosion. It is recognized that actual corrosion never possesses the uniformity of the simulated corrosion; however, to achieve uniform results this test geometry is appropriate. Prior to installation of the repair, strain gages were installed on the sample. **Figure 2** shows the location of the strain gages. Nine were placed on the steel pipe and three were placed on the outside surface of the repair once it had been installed. The gages that provide the greatest information relative to the performance of the repair are those located in the center of the corrosion groove beneath the repair (i.e. Gages 1 through 3). These gages indicate the level of reinforcement



provided by the composite material and at what point load is transferred from the steel to the composite material.



**Figure 1 – Schematic diagram showing pressure only test sample**



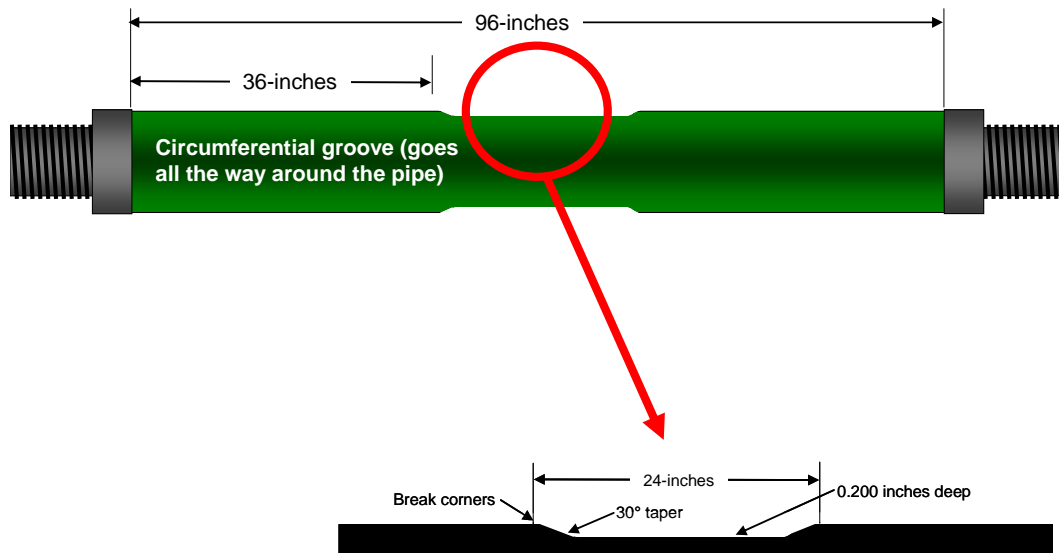
**Figure 2 – Location of strain gages on the pressure and pressure/tension samples**

### **Pressure-tension test**

The next series of tests involved a sample similar to the pressure only sample; however, the focus was on axial tension capacity. In this test, pressure was held constant (2,887 psi which is 72 percent of the 4,000 psi yield strength, consistent with the design basis from ASME B31.8), while axial tension was increased to the point of failure. **Figure 3** shows the schematic for this test, which is identical to the pressure only test except that instead of elliptical dome caps, 7-1/2 inch diameter STUB ACME threaded end caps were used to interface with the tension load frame. As with the pressure only sample, strain gages were installed on the tension-pressure sample at the same locations shown in **Figure 2**..

Prior to testing, details on the importance of having adequate repair length were provided to each of the manufacturers. If a sufficient reinforcing length is not available, during tension loading premature failure of the repair will ensue because of the inability of the repair to remain attached to the pipe. As a point of reference, consider that an axial length of 18 inches exists on each side of the repair. If an adhesive lap shear strength of 1,000 psi exists (a conservative estimate considering the performance of most epoxy adhesive systems), a tensile capacity of approximately 490 kips exists prior to failure of the adhesive bond between the steel pipe and composite material. For the nominal pipe wall, this results in an axial stress of 44.5 ksi.

Samples were taken to failure by increasing the axial tension in the sample to the point where failure in the corroded region occurred. The indication of failure was when pressure in the sample could no longer be maintained.

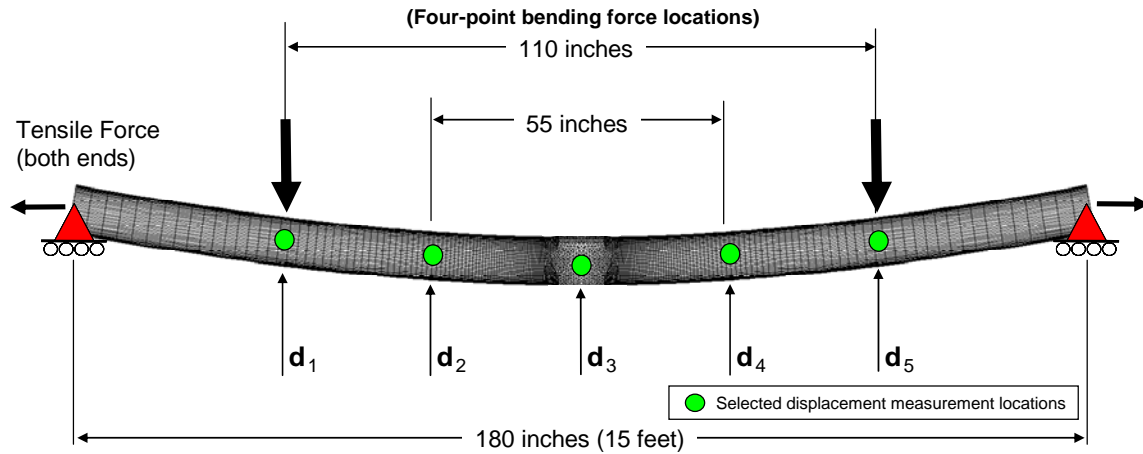


**Figure 3 – Schematic diagram showing pressure-tension test sample**

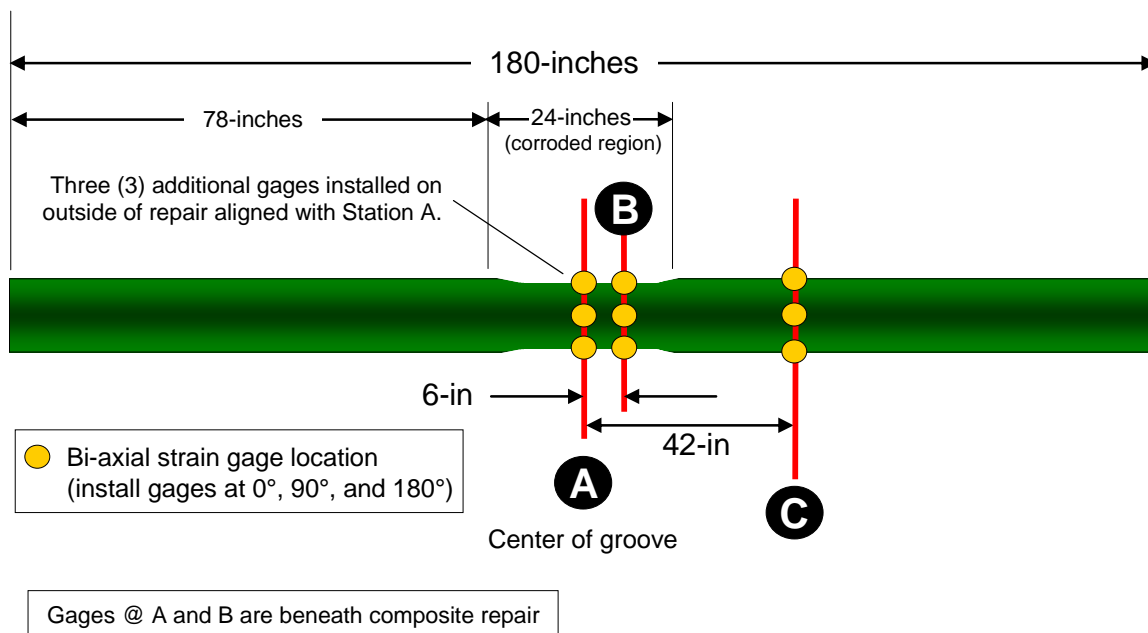
### **Pressure-tension-bending test**

This test combined all three load types: internal pressure, tension, and bending. The variable load of interest in this round of testing was bending. During testing, internal pressure and tension were held constant at 2,887 psi and 145 kips, respectively. Bending loads were applied using a four-point bend configuration as shown in **Figure 4**. Holding pressure and tension constant, the bending load was increased by incrementally increasing the force applied by the two hydraulic rams. Due to safety concerns, testing was terminated once significant plastic flow in the reinforced corrosion area occurred. This also corresponded to the point where load was transferred from the steel to the composite material as observed by the strain gages positioned beneath the reinforcement.

**Figure 5** shows the location of the strain gages placed on the pressure-tension-bend samples. As with the other two tests, nine strain gages were installed on the pipe and three were installed on the outside surface of the composite repair after curing had taken place. **Figure 6** shows the load frame used for the bend tests. This load frame has an axial tension capacity of 1 million lbs and can apply bending loads up to 750 kip-feet.



**Figure 4 – Four point bending configuration for pressure-tension-bend testing**



**Figure 5 - Location of strain gages on the pressure-tension-bend samples**



**Figure 6 – Load frame used for pressure-tension-bend testing**

## PRESENTATION OF RESULTS

Over a five week period, tests were performed on one set of unrepaired samples and four different composite repair systems. Results are presented for the four repair systems and the unrepaired sample in the sections that follow. Considering all phases of testing, data were recorded by a total of 159 strain gages. However, presentation of results is limited to gages located beneath the repairs in order to demonstrate the level of reinforcement provided by each of the repair systems. It should be noted that results for Product B are not included. The manufacturer of this repair requested that their results not be included after sub-standard performance resulted due to uncured adhesives.

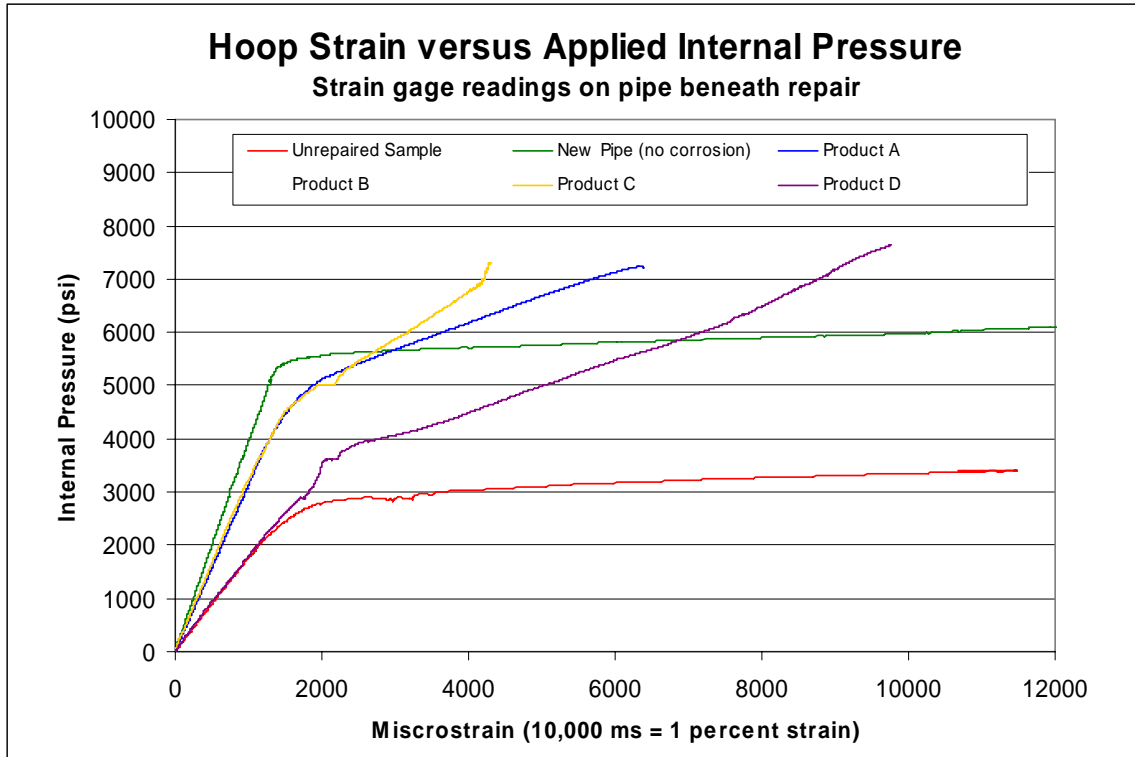
### Pressure-only Test

Results for the pressure-only test are provided in **Figure 6**. This phase of testing represents the initial benchmark of the test. To a certain extent, it presents the most basic test as it only addresses the performance of the repair in reinforcing hoop strength.

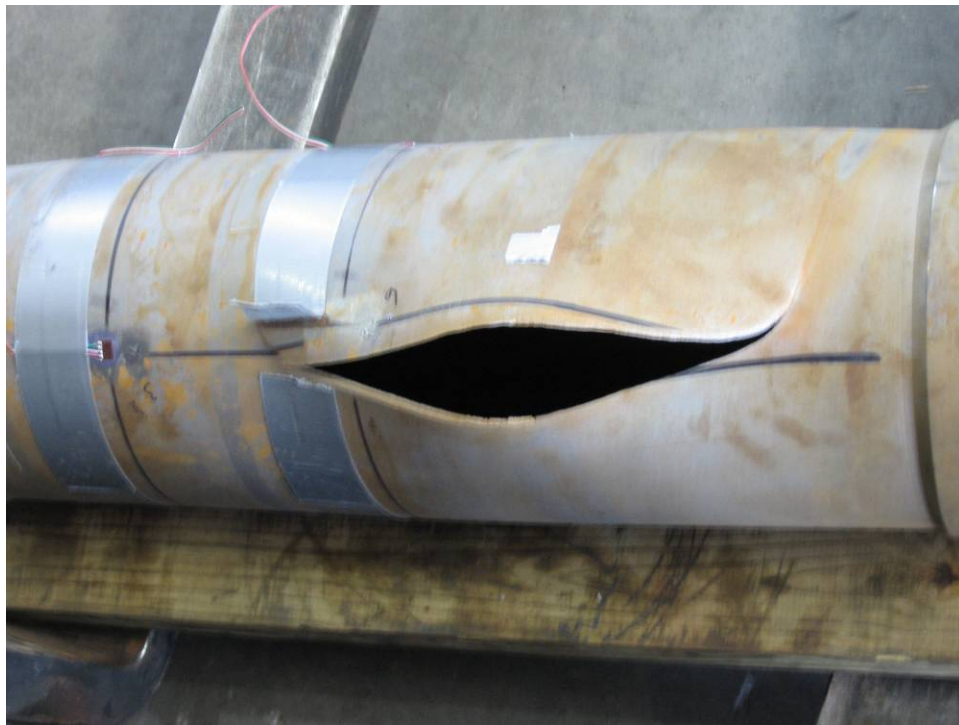
In reviewing the test data in **Figure 6**, there are several noteworthy points:

- In limit state design, one must address the limit state, or the maximum capacity a structure can withstand. Although fundamentally this involves failure, more practically it involves assessing the load at which unbounded displacements (or strains) occur. In pressure vessel design, this condition is known as the lower bound collapse load. The strain gage results presented in **Figure 6** show the pressure at which unbounded displacements occur, typically near 2000 microstrain (or 0.2 percent strain).
- The post-yield slope in the stress-strain curves observed for each of the repair systems is the result of reinforcement being provided to the corroded region of the steel pipe. This occurs once plasticity initiates in the steel and load is transferred to the reinforcing composite material. This bi-linear stress-strain curve is typical for structures reinforced using composite materials subject to tensile loading.

- The unrepaired sample failed at a pressure of 3,694 psi. The failure pressures for the four repaired samples are listed below. Failures in the test samples involving Products A, C, and D occurred in the steel away from the repaired region. **Figure 7** shows the failure in the unrepaired sample, while **Figure 8** shows the failure in the Product C repaired sample outside of the repaired region in the base pipe.
  - Unrepaired – 3,694 psi
  - Product A – 6,921 psi
  - Product B – data not reported
  - Product C – 7,502 psi
  - Product D – 7,641 psi
  
- The strain gage results provide measurements of the strains in the pipe during pressurization. The measurements of greatest significance are those that demonstrate behavior once yielding initiates in the steel and the point at which load is transferred from the steel into the composite material. This latter observation is the best indicator for determining how much reinforcement is provided by the composite material. Product C provides the greatest continuous reinforcement, while Product A provides similar results up to 2,500 microstrain. Although Product D did not provide the same level of strain reduction beneath the repair as the other two systems, this system generated the largest burst pressure of all the test samples.



**Figure 6 – Test results from pressure-only testing**



**Figure 7 – Failure in unrepaired test sample**





**Figure 8 – Failure in burst sample using Product C**

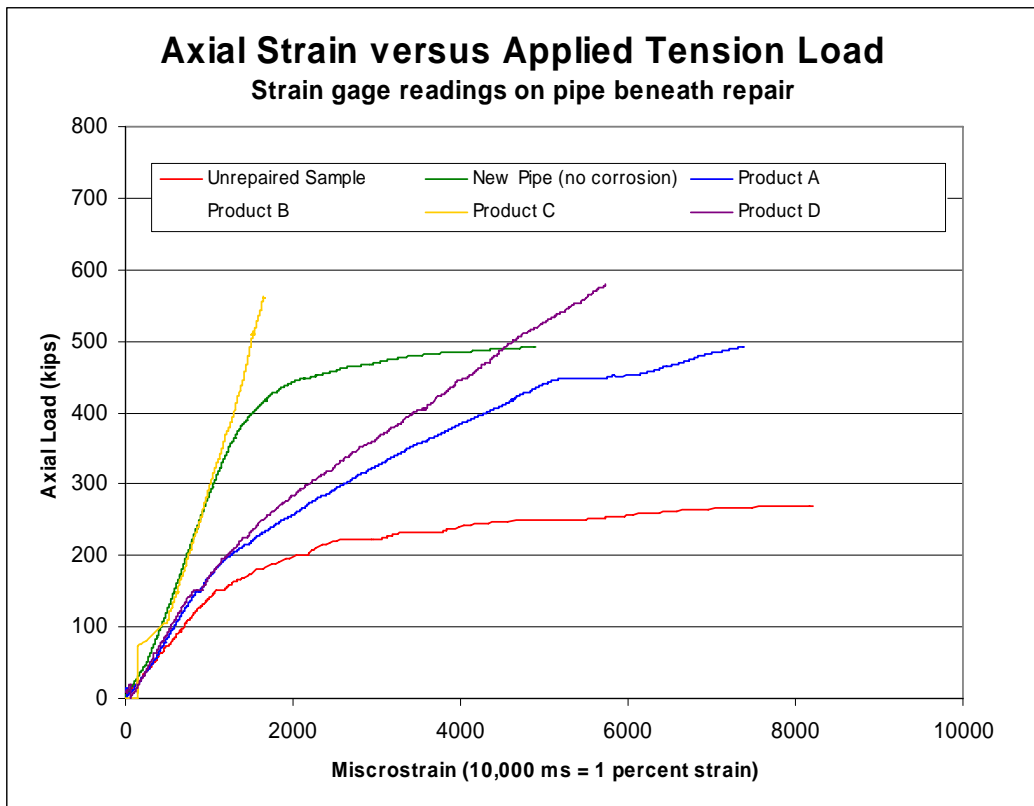
### **Pressure-tension Test**

Results for the pressure-tension test are provided in **Figure 9**. This phase of testing fundamentally assessed the lap shear strength of the adhesive that bonded the composite reinforcement to the steel pipe. This failure condition was anticipated prior to testing and was the basis for the minimum repair length of 60 inches. Several noteworthy observations are made in reviewing the test data presented in **Figure 9**:

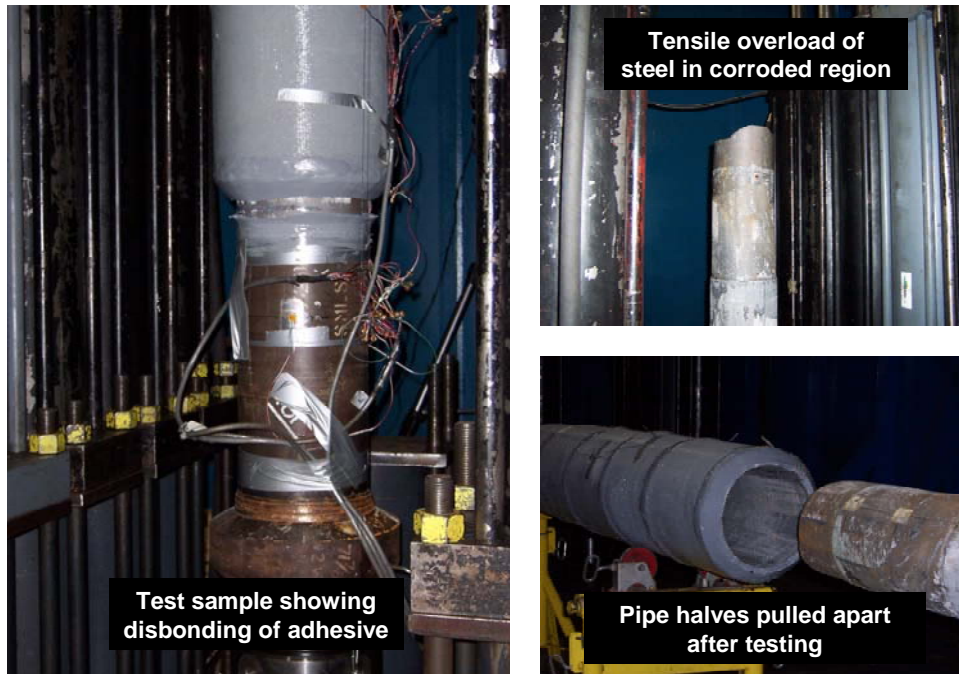
- Product C shows the greatest axial rigidity of all the repair systems. The basis for this observation is that Product C was fabricated using carbon fibers, with a large percentage of fibers being oriented axially. Products A and D show similar levels of reinforcement up to 200 kips, while after this point Product D shows greater reinforcement.
- The following tension failure data were recorded.

- Unrepaired sample – 317 kips
- Product A – 492 kips
- Product B – data not reported
- Product C – 562 kips
- Product D – 579 kips

**Figure 10** provides several photos showing the post-failure surface of the pressure-tension sample for Product D. As shown, the inner steel in the corroded region failed due to tensile overload. From a mechanics standpoint, the adhesive at the interface between the composite and steel transfers load into the composite material. At some point during loading, the strength in this bond is exceeded and the composite is no longer able to carry the tensile load. As shown in **Figure 10** (lower right hand side photo), the composite material remains intact so that the repair system mimics a pipe-in-pipe repair where the strength of the repair is dependent on the interface bond between the two pipes.



**Figure 9 – Test results from pressure-tension testing**



**Figure 10 – Post-failure photos of Product D pressure-tension test**

### **Pressure-tension-bending Test**

Prior to starting the testing phase of work, this particular test was recognized as the most likely challenge of the three test configurations. It not only combined constant pressure (2,887 psi) and constant axial tension (145 kips), it integrated bending loads that would induce significant axial strains in both the corroded steel and composite material. Unlike the pressure-tension tests where the primary focus was on the interfacial adhesive bond, this phase of testing integrated the needs for adequate bond strength, but the repair was also required to have sufficient stiffness in the composite to reinforce the corroded steel.

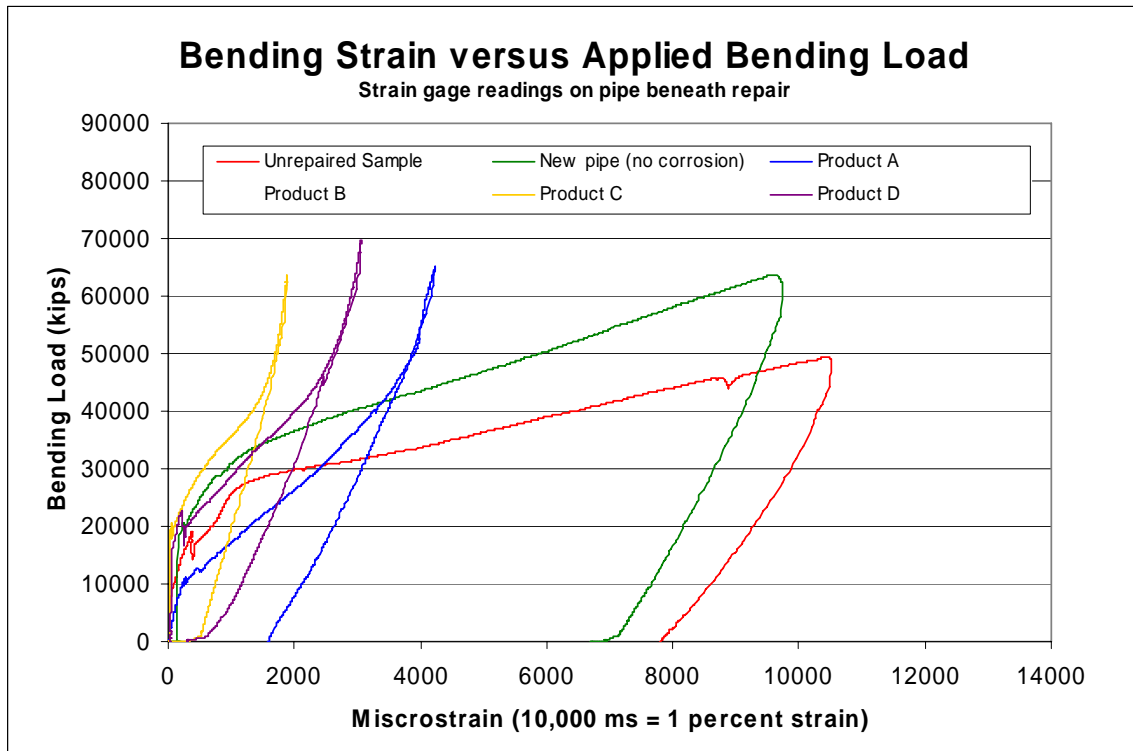
Results for the pressure-tension-bending test are provided in **Figure 11**. There are several noteworthy observations in reviewing the plotted data:

- Unlike the other tests, there is a unique pattern observed for the level of reinforcement provided by each of the respective repair systems. As expected, the carbon in Product C provides the greatest level of reinforcement. For comparison

purposes, consider the strain in the steel at a bending load of 40 kips (bending moment of 116.7 ft-lbs) for each of the repair systems:

- Product A – 4,130 microstrain
  - Product B – data not reported
  - Product C – 2,150 microstrain
  - Product D – 3,022 microstrain
- 
- In assessing the relative performance of the composite systems, the objective of the repair is to reduce the strain in the corroded steel during bend testing, as well as provide reinforcement in the circumferential and axial directions against internal pressure and axial tension loads, respectively. As noted in **Figure 11**, at some point the strain gage results appear to stop changing with increasing load (plotted lines go vertical). It is at this point that gross plastic deformation occurs outside of the reinforced region and that deflection is occurring primarily in areas outside the composite reinforcement. The sooner this transformation takes place, the more effective the repair is in reinforcing the corroded region.
  
  - Another option for assessing the relative performance of the composite repair systems is to determine the applied bending moment at a specified strain value. If the strain limit is 0.20 percent, the following bending forces and moments are extracted. This method is a better assessment of the relative performance of the repair systems. It should be noted that the unreinforced sample did not include internal pressure as failure would have occurred at a low bending load.
    - Unreinforced sample – 30 kips (87.5 kip-feet)
    - Product A – 26 kips (75.8 kip-feet)
    - Product B – data not reported
    - Product C – 70 kips (204.2 kip-feet)
    - Product D – 40 kips (116.7 kip-feet)

**Figure 12** is a photograph of the Product C repair in the load frame prior to bend testing.



**Figure 11 – Test results from pressure-tension-bending testing**



**Figure 12 – Photo showing Product C prior to bend testing**

## GENERAL OBSERVATIONS ON THE JIP TEST RESULTS

In assessing the overall performance of the repair system, it is clear that all of the reported data show clear benefit over the unrepaired configuration. **Table 1** is presented that shows the test results relative to the design performance criteria. As noted, the composite repair systems exceed the design loads by a relative large margin. Specifically, the following average design margins were calculated for all of the repair systems. These were calculated by dividing the failure load by the specified design load:

- Pressure testing – average design margin of 2.56
- Tension testing – average design margin of 3.75
- Bend testing – average design margin of 2.59

In conventional limit state design methods, design margins ranging from 1.5 to 2.0 are typical. Provided below are three resources for design codes that use limit state design methods:

- ASME Section III, Nuclear code
  - Limit Analysis (NB-3228.1)
  - Plastic Analysis (NB-3228.3)
  - Design margin of  $2/3$  against lower bound collapse load (alternative margin value is the inverse of this value: design margin of 1.5)
- ASME Section VIII, Division 2, Alternative Rules
  - Analysis (4-136.3 Limit Analysis)
  - Testing: Appendix 6 (specifically 6-153)
  - Design margin of  $2/3$  against lower bound collapse load (alternative margin value is the inverse of this value: design margin of 1.5)
- ASME Section VIII, Division 3, Construction of High Pressure Vessels
  - Design margin of 1.732 against collapse load (KD-240)

As seen with the above values, on average the composite reinforcement systems possess an adequate safety margin for their intended service conditions.

**Table 1 – Summary of test results relative to design conditions**

Loading Conditions	Design Load	Failure Loads				
		Unrepaired	Product A	Product B	Product C	Product D
Internal pressure	2,887 psi	3,694 psi	6,921 psi	N/A	7,592 psi	7,641 psi
Tension Load	145 kips	317 kips	492 kips	N/A	562 kips	579 kips
Bending Force (Moment)	17.5 kips (51 kip-feet)	30 kips (87.5 kip-feet)	26 kips (75.8 kip-feet)	N/A	69.9 kips (204.2 kip-feet)	40 kips (116.7 kip-feet)

In using composite materials to reinforce damaged and corroded risers, it is critical to integrate a design methodology that assesses the strain in the reinforced steel. This is especially important in offshore design as risers in the splash zone are subjected to combined loads including internal pressure, axial tension, and bending loads, as compared to onshore repairs that primarily involve restoration of hoop strength.

As demonstrated in this effort, use of strain based design methods is the ideal approach for assessing the interaction of load transfer between the reinforced steel and the reinforcing composite material. Industry should be cautious of any design methodology that does not capture the mechanics associated with the load transfer between the steel and composite materials during the process of loading. The two keys are to first determine strain limits based on acceptable design margins, and then assess strain levels in both the steel and composite reinforcement using either analysis methods, or the preferred approach involving full-scale testing with strain gages.