

# The Effect of Unheated Sections on Moisture Transport in the Emplacement Drift

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## **General Objectives**

- Prepare Thermal Hydrological Near-Field/In-Drift Model
  - > Develop a multi-scale, coupled seepage model that accounts for natural convection processes in the emplacement drifts
  - Use TOUGH2 code for rock mass and MULTIFLUX code for in-drift processes
  - Apply an efficient NTCF methodology for coupling in-rock and indrift model elements
  - > Study two application scales:
    - >> Mountain-scale/drift-scale model to provide temperature and relative humidity evolutions along representative drifts
    - Small-scale, high-resolution seepage model to evaluate evaporation impact on seepage predictions
- Study the Impact of Natural Convection on Seepage with



Less conservatism in seepage predictions

More realistic performance estimate of natural barrier



### **Goals of Current Study**

- Configure a thermal-hydrological natural-ventilation model for simulating temperature, humidity, and condensate distributions in the coupled domains of in-drift airspace and near-field rockmass. Rokmass model: TOUGH2, in-drift model: MULTIFLUX (MF)
- Obtain meaningful results from the model for a practical application in which the beneficial effects of unheated drift sections are analyzed.
- Study the sensitivity to the axial dispersion coefficient with the model.



## **Coupled Modeling with MULTIFLUX and TOUGH2**



### Mountain-Scale-to-Drift-Scale Model



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## **Mountain-Scale NTCF matrix model**

$$qh = qh^{c} + hh \cdot \left(T - T^{c}\right) + \left\langle T \right\rangle \cdot hm \cdot \left(P - P^{c}\right)$$

$$qm = qm^{c} + mh \cdot \left(T - T^{c}\right) + \left\langle T \right\rangle \cdot mm \cdot \left(P - P^{c}\right)$$

*F*, *P*: wall temperature and partial vapor pressure vectors *qh*, *qm*: heat and moisture fluxes from TOUGH2 *T<sup>c</sup>*, *P<sup>c</sup>*: central input boundary conditions *qh<sup>c</sup>*, *qm<sup>c</sup>*: central output fluxes from TOUGH2, for T=T<sup>c</sup> and P=P<sup>c</sup> *hh*, *hm*: dynamic admittance matrices for heat flow *mh*, *mm*: dynamic admittance matrices for moisture flow



## **NTCF Model Fit Against Input TOUGH2 Data**



#### Energy Balance Equation in the CFD Model of MF

The energy balance equation in the CFD model of MF for "x" directional flow in "dy" by "dz" cross-section is:

$$\rho c \frac{\partial T}{\partial t} + \rho c v_i \frac{\partial T}{\partial x} = \rho c a \frac{\partial^2 T}{\partial x^2} + \rho c a \frac{\partial^2 T}{\partial y^2} + \rho c a \frac{\partial^2 T}{\partial z^2} + \dot{q}_h$$

where,

- $v_i$  velocity
- $\rho$  density of moist air
- *c* specific heat of moist air
- $q_h$  latent heat source or sink for condensation or evaporation



## Moisture Transport Convection-Diffusion Equation in the CFD Model of MF

The simplified moisture transport convection-diffusion equation in the CFD model of MF for "x" directional flow in "dy" by "dz" cross-section is:

$$\frac{\partial \rho_{v}}{\partial t} + v_{i} \frac{\partial \rho_{v}}{\partial x} = D \frac{\partial^{2} \rho_{v}}{\partial x^{2}} + D \frac{\partial^{2} \rho_{v}}{\partial y^{2}} + D \frac{\partial^{2} \rho_{v}}{\partial z^{2}} + \dot{q}_{cm} + \dot{q}_{sm}$$

where,

 $v_i$  - velocity

D - molecular or eddy diffusivity for vapor

 $\dot{q}_{sm}$  - moisture source or sink due to condensation or evaporation  $\dot{q}_{cm}$  - vapor flux in superheated steam form



#### In-Drift Lumped-Parameter CFD Model Concept

#### In-Drift Heat Flow Processes (Simplified Diagram)



#### In-Drift Lumped-Parameter CFD Model Concept

#### In-Drift Moisture Flow Processes (Simplified Diagram)



#### Sectional view of the emplacement drift with unsealed and sealed-off unheated sections





#### Simplified Logic Flow Chart of Coupling In-Drift CFD and In-Rock NTCF Model Elements





### **Results: Temperature and Relative Humidity on the Drift Wall, Un-Sealed Unheated Drift Sections**





### **Results: Temperature and Relative Humidity on the Drift Wall with Sealed-off Unheated Sections**





#### Axial distribution of (a) drift wall temperature, (b) relative humidity, and (c) condensation rate using an axial moisture dispersion coefficient of 0.1 m<sup>2</sup>/s

solid line: unheated sections not sealed

dashed line: unheated sections sealed off

















![](_page_15_Figure_11.jpeg)

![](_page_15_Figure_12.jpeg)

![](_page_15_Picture_13.jpeg)

#### Axial distribution of (a) drift wall temperature, (b) relative humidity, and (c) condensation rate using an axial moisture dispersion coefficient of 0.004 m<sup>2</sup>/s

solid line: unheated sections not sealed

dashed line: unheated sections sealed off

![](_page_16_Figure_3.jpeg)

![](_page_16_Figure_4.jpeg)

![](_page_16_Figure_5.jpeg)

![](_page_16_Figure_6.jpeg)

500

1000

![](_page_16_Figure_7.jpeg)

150

100

Year 75

![](_page_16_Figure_8.jpeg)

![](_page_16_Figure_9.jpeg)

![](_page_16_Figure_10.jpeg)

![](_page_16_Picture_11.jpeg)

#### Future Model: Proposed, Variable Dispersion Coefficient in the CFD Model Element

$$D = D_0 \left\{ 1 + 76Ra_{L_C,\Delta T_r}^{0.05} \left[ 1 + 0.05 \left( \frac{L_C}{L} \right) Ra_{L_C,\Delta T_L}^{0.5} \right] \right\}$$

$$L_c$$
 - gap-width characteristic length

- $Ra_{Lc,\Delta Tr}$  calculated with the gap-width characteristic length,  $L_c$ , and temperature difference in radial direction
- *L* axially connected length
- $Ra_{Lc,\Delta Tr}$  calculated with the same characteristic length but with a temperature difference over the axially connected distance, L

![](_page_17_Picture_6.jpeg)

#### Fit of Dispersion Coefficient Values (m<sup>2</sup>/s) between Proposed and Published Results by Webb and Itamura

|                        |                      | Without Temperature Tilt |          | With Temperature Tilt |          |
|------------------------|----------------------|--------------------------|----------|-----------------------|----------|
|                        |                      | 1000 Yrs                 | 3000 Yrs | 1000 Yrs              | 3000 Yrs |
| Under<br>Drip Shield   | Published<br>results | 0.006                    | 0.007    | 0.007                 | 0.009    |
|                        | proposed             | 0.004                    | 0.004    | 0.009                 | 0.008    |
| Outside<br>Drip Shield | published<br>results | 0.004                    | 0.004    | 0.1                   | 0.1      |
|                        | proposed             | 0.004                    | 0.004    | 0.1                   | 0.081    |

![](_page_18_Picture_2.jpeg)

## **Observed Natural Convection Effects**

Phase I: Boiling Conditions

![](_page_19_Picture_2.jpeg)

More drying of rock mass and later rewetting (delay of seepage)

![](_page_19_Picture_4.jpeg)

![](_page_19_Picture_5.jpeg)

NW05-028

![](_page_19_Picture_7.jpeg)

## Conclusions

- A fully-coupled, in-drift and near-field, in-rock model is configured and applied for the solution of a complex thermo-hydrologic-airflow problem at YM.
- As a coupled thermal-hydrologic model exercise, the beneficial effect of elongated, unheated emplacement drift sections at both ends was studied and comparatively evaluated.
- No condensation was found around the WPs, and an improvement to the results for a drift arrangement without the long unheated sections was achieved in the high axial dispersion coefficient case.
- Lower condensation rates and fewer condensation locations in the emplacement drift are predicted with the present model than those obtained using an approximate, and basically uncoupled condensation model.

![](_page_20_Picture_5.jpeg)

## **Conclusions (Continued)**

- The current result illustrates the benefit of maintaining unheated, lowtemperature sections in the drift airspace in order to lower the relative humidity in the active emplacement drift section.
- Significant sensitivity to the axial dispersion coefficient in the emplacement drift is found. This fact underlines the importance of a fastrunning, efficient modeling method, since input data variations will likely be needed in future studies and design exercises.
- The range of the values for axial dispersion coefficient arches over three orders of magnitude from molecular diffusion to turbulent dispersion in an emplacement drift.
- In order to reduce uncertainties, it will be important to use locationspecific, temperature-field dependent coefficients in the lumped-parameter CFD model instead of overall constants for the entire drift in future studies using a dispersion coefficient model example proposed in the current paper.

![](_page_21_Picture_5.jpeg)

# **Support Material**

![](_page_22_Picture_1.jpeg)

## Background

- Seepage of water into emplacement drifts is important for performance assessment (PA)
- Seepage predictions in PA are currently conducted using a conservative assumption of 100% relative humidity (RH) in emplacement drifts (no evaporation)
- Natural convection processes in emplacement drifts will create evaporative potential over long drift sections

![](_page_23_Picture_4.jpeg)

![](_page_23_Figure_5.jpeg)

## Logic Flow Chart of the Coupled Simulation

![](_page_24_Figure_1.jpeg)

![](_page_24_Picture_2.jpeg)

#### **Embedded High-Resolution Seepage Model**

![](_page_25_Figure_1.jpeg)