Engineering Development of Coal-Fired High-Performance Power Systems

Progress Report
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INTRODUCTION

In Phase 1 of the project, a conceptual design of a coal-fired, high-performance power system was developed, and small-scale R&D was done in critical areas of the design. The current phase of the project includes development through the pilot plant stage and design of a prototype plant that would be built in Phase 3.

Foster Wheeler Development Corporation (FWDC) is leading a team of companies in this effort. These companies are:

- Foster Wheeler Energy Corporation (FWEC)

- AlliedSignal Aerospace Equipment Systems

- Bechtel Corporation

- University of Tennessee Space Institute (UTSI)

- Westinghouse Electric Corporation

The power-generating system being developed in this project will be an improvement over current coal-fired systems. Goals have been identified that relate to the efficiency, emissions, costs, and general operation of the system. These goals are:

- Total station efficiency of at least 47 percent on a HHV basis.

- Emissions:

NOx	<0.06 lb/MM Btu
SOx	<0.06 lb/MM Btu
Particulates	< 0.003 lb/MM Btu

- All solid wastes must be benign with regard to disposal.

- Over 95 percent of the total heat input is ultimately from coal, with initial systems capable of using coal for at least 65 percent of the heat input.

- Ten-percent lower cost of electricity (COE) relative to a modern coal-fired plant conforming to NSPS.

The base case arrangement of the HIPPS cycle is shown in Figure 1. It is a combined-cycle plant. This arrangement is referred to as the All Coal HIPPS because it does not require any other fuels for normal operation. A fluidized bed, air-blown pyrolyzer converts coal into fuel gas and char. The char is fired in a high-temperature advanced furnace (HITAF) which heats both air for a gas turbine and steam for a steam turbine. The air is heated up to 760°C (1400°F) in the HITAF, and the tube banks for heating the air are constructed of alloy tubes. The fuel gas from the pyrolyzer goes to a topping combustor where it is used to raise the air entering the gas turbine to 1288°C (2350°F). In addition to the HITAF, steam duty is achieved with a heat-recovery steam generator (HRSG) in the gas turbine exhaust stream and economizers in the HITAF flue gas exhaust stream.
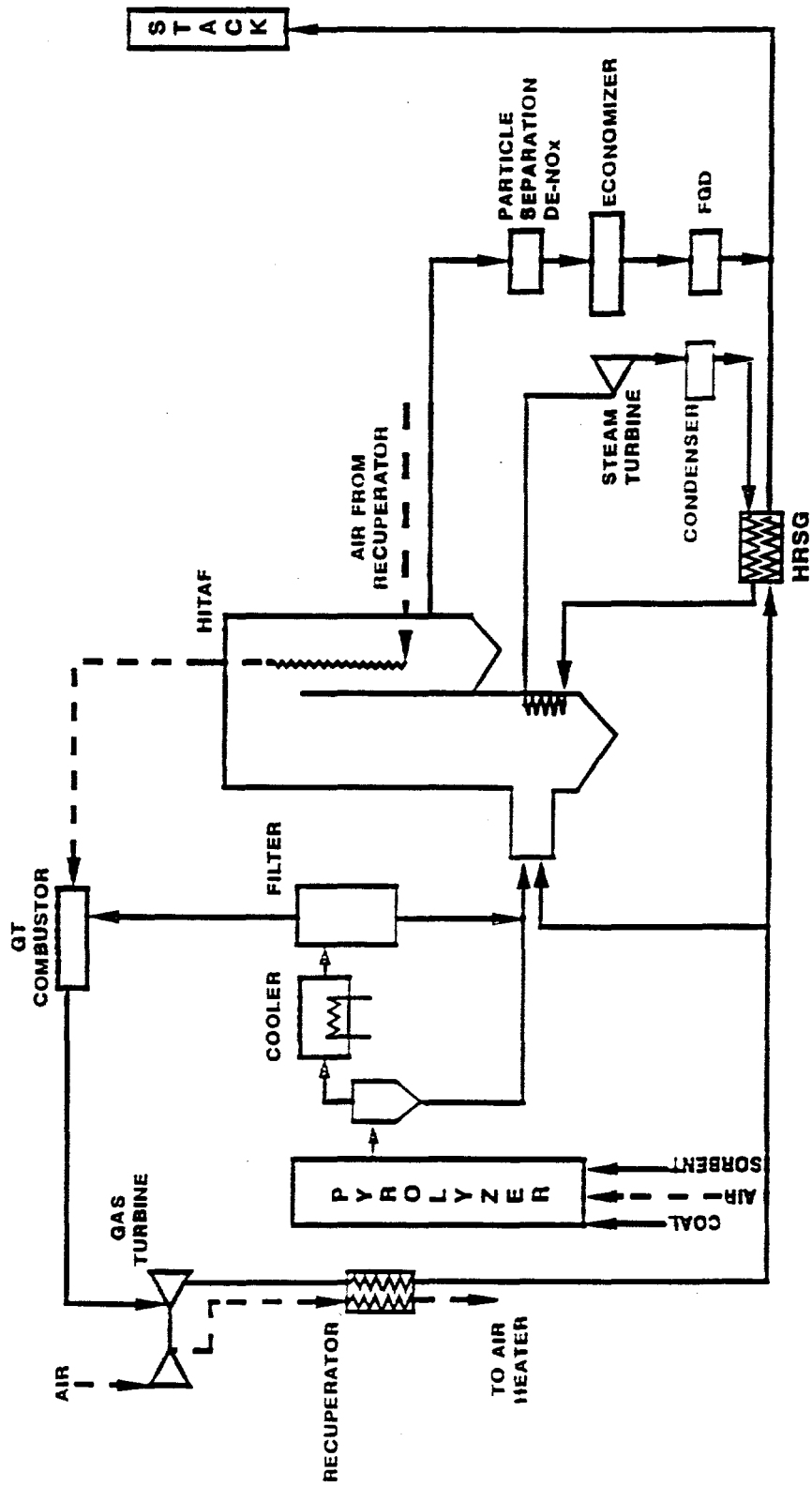
Figure 1   All Coal-Fired HIPPS

A schematic of an alternative HIPPS cycle is shown in Figure 2. This arrangement uses a ceramic air heater to heat the air to temperatures above what can be achieved with alloy tubes. This arrangement is referred to as the 35-percent natural gas HIPPS. A pyrolyzer is used as in the base-case HIPPS, but the fuel gas generated is fired upstream of the ceramic air heater instead of in the topping combustor. Gas turbine air is heated to 760°C (1400°F) in alloy tubes the same as in the All Coal HIPPS. This air then goes to the ceramic air heater where it is heated further before going to the topping combustor. The temperature of the air leaving the ceramic air heater will depend on technological developments in that component. An air exit temperature of 982°C (1800°F) will result in 35 percent of the heat input from natural gas.

TECHNICAL PROGRESS

Task 1 - Project Planning and Management

During this Quarter, Foster Wheeler Energy Corporation (FWEC) was substituted for TRW as the supplier of the char combustion system. FWEC has years of commercial experience in burning low-volatile fuels, and their arch-fired combustion system has always been an option for HIPPS. The proposal was modified to substitute FWEC for TRW. The R&D that was to be done by TRW will still be done, but it will be done by FWEC on their combustion system.

During this reporting period, Bechtel's activities continued to be focused on improving the performance and operability of the all-coal feed version of the commercial plant design. Specifically, the areas receiving attention included:

- Modifying operating parameters to either adjust for changes in equipment specifications or to improve overall plant performance. Areas modified during the current reporting period include:

    - Examining the effect of adjusting the air temperature leaving the primary (alloy) air heater

    - Replacing the TRW slagging combustors with FW burners

    - Examining the effect of increasing the temperature to the gas turbine

    - Removing the supplemental coal feed to the furnace.
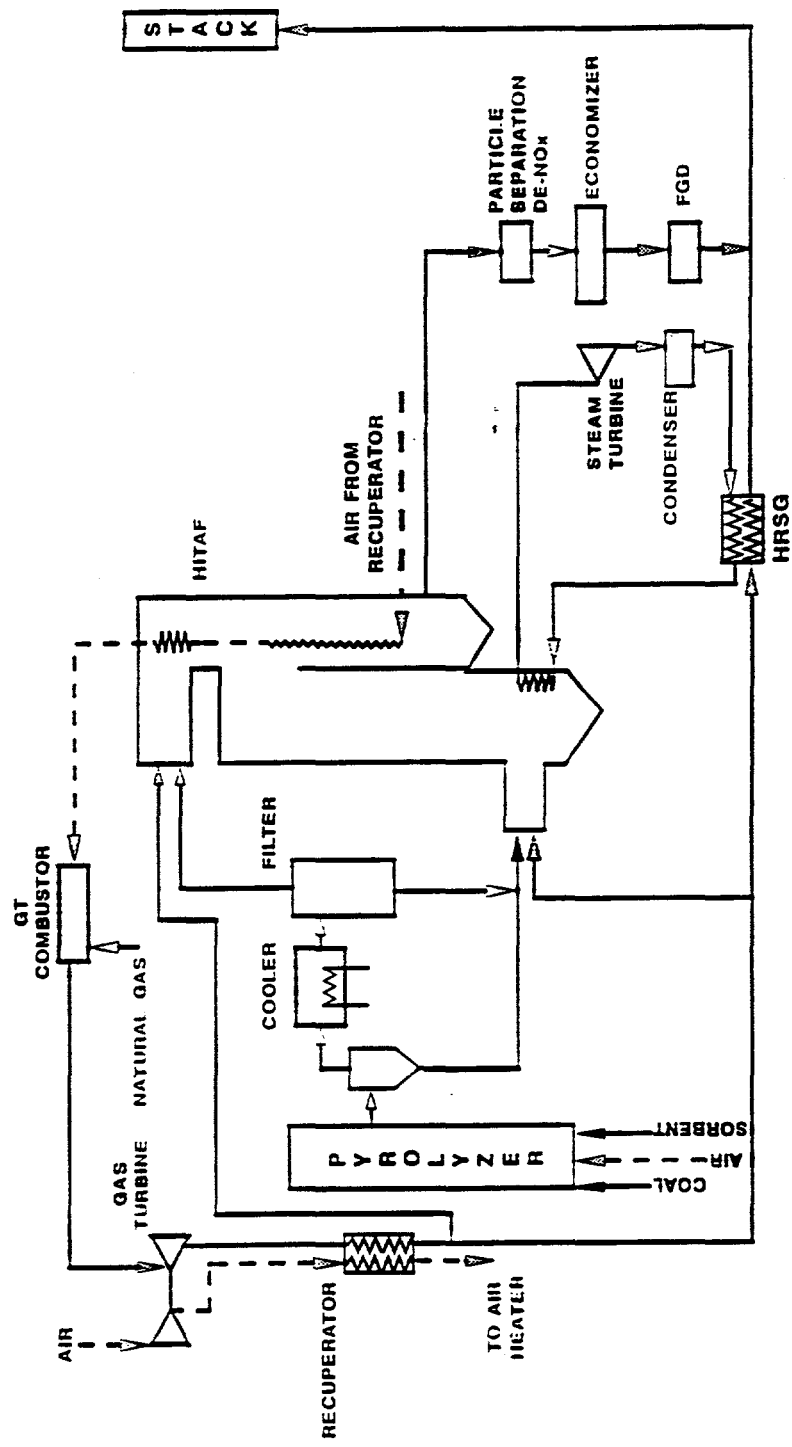
Figure 2 35-Percent Natural Gas HIPPS

- Looking at possible modifications in the steam bottoming cycle to improve operability including:

  - Adding low-pressure boilers

  - Relocating some superheaters/reheaters

  - Changing the pyrolysis fuel gas cooler from superheater to a high-pressure BFW heater

  - Changing the BFW flow configuration.

The impact of increasing the temperature limits surrounding the ceramic exchanger in the partial natural gas-fired version of the commercial design was also investigated.

Additional information on each of these topics follows.

Modifying the Commercial Design Operating Parameters

Changing air heater temperatures. The initial phases of the work in this area were reported last quarter. It included the examination of the impact of adding water, by various means, to the pyrolyzer system. It was found that injection of high levels of steam into the pyrolyzer appeared to offer the best means to improve the overall system performance. It was found that increasing the level of steam increased the production of fuel gas at the expense of char production. This increased the amount of energy in the coal going to the gas turbine and, in return, reduced the amount going to the bottoming cycle, thereby improving the overall system performance.

During this reporting period, this analysis was taken one step further. Logically, as the amount of char is reduced, the point is reached where there is insufficient high-level heat to sustain an acceplable level of efficiency for the steam system. For the 166 bar/580°C/580°C (2400 psig/1075°F/1075°F) steam system currently employed, nearly 60 percent of the heat would be considered high-level heat required for evaporation, superheating, or reheating. The other 40+ percent is used to heat boiler feed water (BFW). The effect of changing this ratio was confirmed during the last quarter when it was found that for very high pyrolysis steam loadings, the heat rate for the bottoming cycle increased dramatically when the temperatures of the exhaust gas leaving the economizers could not be maintained because of reduced BFW flows.

Although the vitiated air flow going to the clean-side evaporators and economizers is more than twice the gas passing through the furnace, the furnace is the key source of high-level heat since the vitiated air leaving the recuperator is only about 480°C (896°F). The effect of char on furnace operation has been studied, but there is another key user of high-level heat. It is the tube banks in the HITAF that heat the gas turbine air. Almost since project inception, the temperature of the air leaving this air heater has been maintained at 760°C (1400°F). This level was set based on material considerations. The question was posed: What would be the effect if this temperature were allowed to float?

As noted above, ideally the process should have about 50 percent more high-level heat available than low-level heat in order to match the steam cycle. If the temperature of the air leaving the air heater

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were allowed to float, then the amount of high-level heat available to the steam cycle could be tailored to provide the nominal 60/40 split required, i.e., for every level of char production, there should be an air heater exit temperature that will leave the proper amount of high-level heat for the bottoming cycle.

A series of GateCycle and spreadsheet runs confirmed this hypothesis, and the results are presented in Figures 3 and 4. Also shown is the impact of increasing the gas turbine inlet temperature. This topic will be discussed in more detail below. As can be seen from Figure 3, an outlet temperature of 760°C (1400°F) is actually only optimum for a steam injection level of about 0.43 kg/kg of coal feed. Fortunately, this is the level previously reported as the dry feed, high-steam case (DF-HS) which was the case chosen to be the proposed New Base Case for the commercial plant design.

The general trend shown in Figure 3 is understandable. As the steam injection level is increased, the char production is decreased. The reduction in char means less heat is available in the furnace. This must be offset by lowering the heat requirements of the air heater to maintain the proper ratio of high-/low-level heat for the bottoming cycle.

Figure 4 presents the estimated overall performance at the various pyrolysis steam levels. It can be seen that above about 0.55 kg/kg-coal the performance drops off rapidly and that there appears to be a maximum around 0.30-0.35 kg/kg-coal. The maximum is the result of offsetting factors.

As discussed above, at low steam rates, there is an increase in the amount of char and subsequently the amount of steam to the bottoming cycle. When the temperature of the air heater is allowed to float, this is offset by the increase in air heater temperature which means more energy from the char is being picked up by the air. This allows more energy to be used in the gas turbine before being picked up by the steam system.

At steam flows below about 0.30 kg/kg-coal, the fraction of energy going to steam begins to be dominant, and the overall performance suffers. The reverse is true at the high end of steam injection. The reduced production of char reduces the amount of steam , but to balance the high-/low-heat loads, the air heater temperature must be reduced. This means less energy from the char goes to the gas turbine and also that more fuel must be produced to make up for the lower temperature of the air reaching the GT combustor.

What this set of calculations indicates is that 1) there is an optimum point, from a performance standpoint, and 2) there is a reasonably broad range of operating conditions where the performance is nearly constant, as seen in Figure 4, between about 0.20 and 0.50 kg steam/kg-coal. This means that the decision on where to operate within this range depends on capital costs and the capabilities of new materials being considered for the air heater.

It should be noted that the data presented in Figures 3 and 4 are off the Bechtel spreadsheet simulation, and while they match the GateCycle runs reasonably well, the absolute values are slightly different. Nevertheless, the trends displayed are accurate and the level of change is also considered representative.
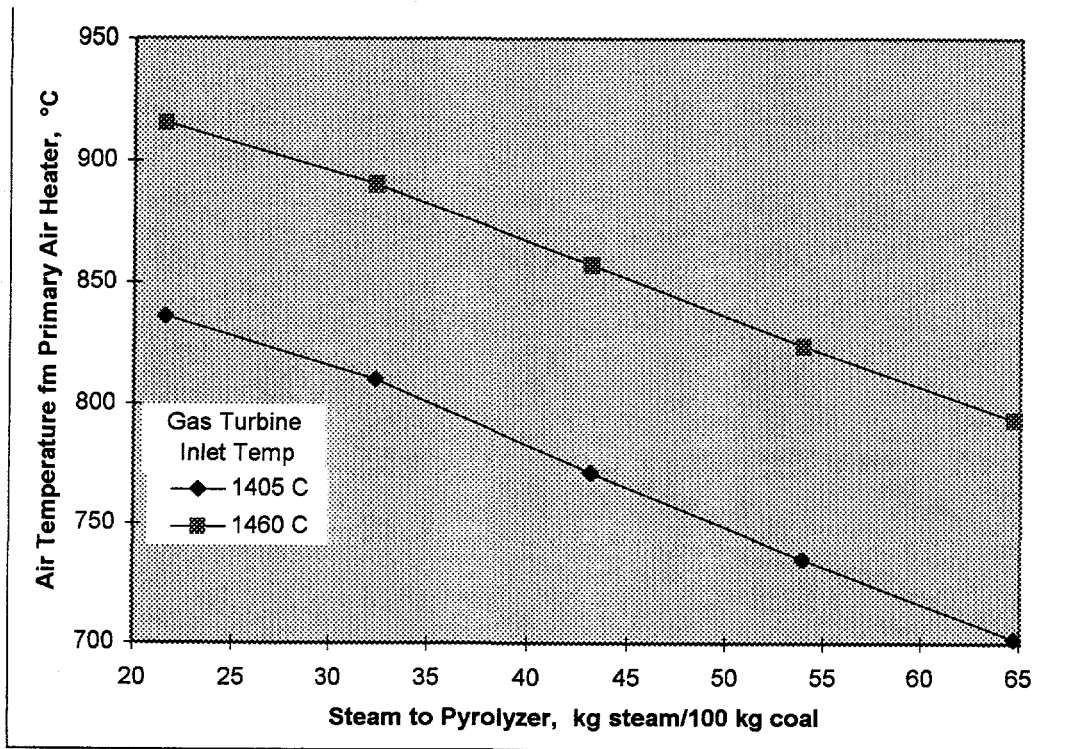
Figure 3
Relationship of Steam to Pyrolyzer and Air Heater Temperature
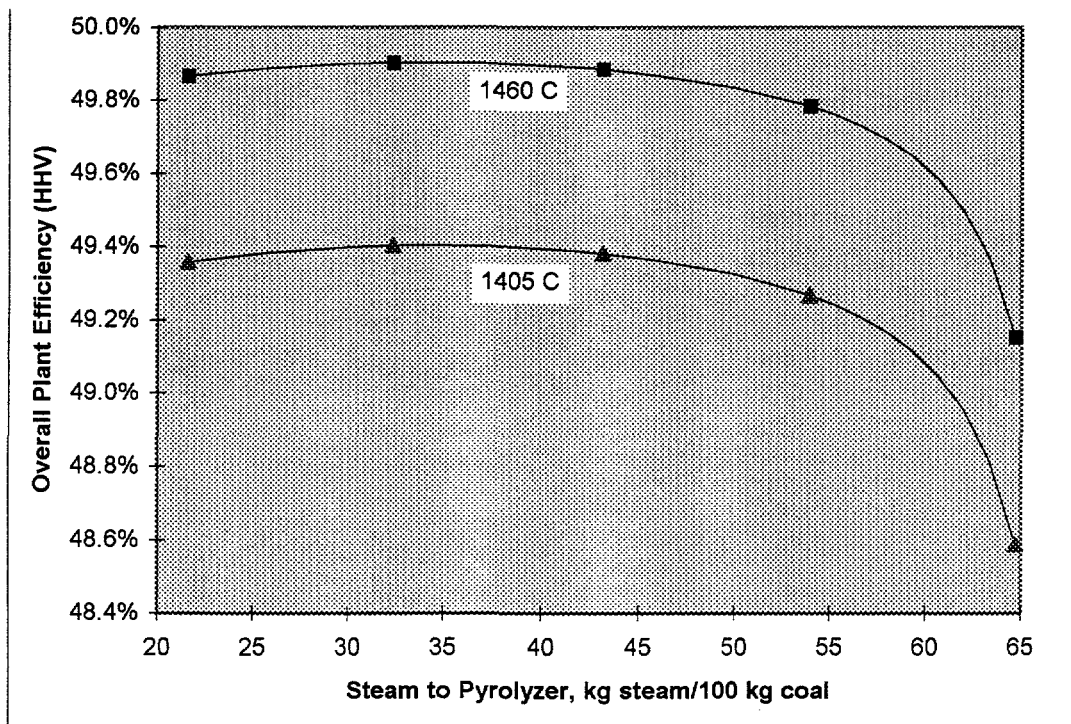
Figure 4
Relationship of GT Inlet Temperature and Steam Flow to Pyrolyzer on Plant Efficiency

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Replacing the TRW slagging combustors. Concurrent with the changes being made in the air heater temperature, the new runs also include the conversion from TRW slagging combustors to FW burners. This involved a couple of changes. First, the TRW combustors incorporated an evaporative cooler that provided a significant amount of the steam generation capacity within the bottoming cycle. This capacity was assumed to be transferred to the furnace as part of the water wall surface.

The second item is more significant, especially in its impact on the overall efficiency. TRW had specified a certain amount of coal to be fed to the preliminary combustor where the primary air was heated before contacting the char in the main combustor. The amount of coal specified provided about 20 percent of the energy to the slagging combustors. It should be noted that any energy supplied by the coal goes directly to the bottoming cycle since the air going to the gas turbine is already being heated by the char. The amount of coal needed for the FW burners will be determined in Phase 2, but it is estimated that it would be from zero to 10 percent. The current balance is at zero percent. This improved the overall efficiency about 1 percentage point. Removing the coal also reduced the amount of high-level heat which also impacted the calculations discussed above.

As part of Phase 2, FWEC will evaluate char as a fuel to determine whether some coal may be required to provide flame stability. Initial testing of the char as part of the Phase 1 program indicated that it was fairly reactive, and it is hoped that coal may only be needed during start-up and/or off-load operating conditions.

Increasing the inlet temperature to the gas turbine. The combustion temperatures used by state-of-the-art main frame and aeroderivative gas turbines continue to climb. Interest was expressed in what effect a high combustion temperature might have on the overall efficiency. The temperature currently used is 1405°C (2561°F) leaving the combustor. This was arbitrarily increased by 55°C (131°F) to 1460°C (2600°F). The results are presented in Figures 3 and 4.

Based on the results to date, it appears that increasing the temperature provides about a 0.5 percentage point increase in overall efficiency. Typically, such an increase in simple cycle and combined cycle gas turbine systems would provide a greater improvement in overall efficiency. However, such changes in inlet temperatures are usually accompanied by corresponding increases in the compression ratio to maximize the gross power production and reduce the heat losses from the turbine exhaust. The current examination did not attempt to adjust the pressure from the compressor since it impacts on so many pieces of equipment. Bechtel will work with Westinghouse to determine what operating conditions should be used if there is continued interest in evaluating the new G-frame machines in a HIPPS configuration.

New Base Balance

It appeared, from the GateCycle runs carried out in parallel with the spreadsheet analysis presented in Figures 3 and 4, that a near-maximum performance was available using the DF-HS steam loading reported in the previous quarter. These were modified to allow for the replacement of the TRW combustors and for an air heater outlet temperature adjusted to 750°C (1385°F). This is very close to the 760°C (1400°F) used previously.

Although some modifications may take place as a result of possible changes in the bottoming cycle (discussed below) and from future economic analyses, it was decided that the current balance was sufficiently close to allow Westinghouse to evaluate the conditions surrounding the gas turbine. Current runs still employ operating conditions suggested by General Electric during the Phase 1 program. Upon receipt of the Westinghouse data, the optimum point will be reconfirmed, and a brief economic analysis will be made to determine the optimum operating conditions for the material balance to be used during Phase 2. It will also serve to provide a basis for the revised capital cost estimate for the commercial plant. The process streams for the heat and material balance are shown in Figure 5. The performance and summary material balance are presented in Tables 1 and 2.

Possible Bottoming Cycle Modifications

The design of the commercial plant is reaching the point where it was felt that a review of some operability and constructability aspects was considered warranted. This work is still ongoing, but some of the items being considered include:

- High-temperature steam pipe is expensive. This is especially true for piping associated with the reheat system since the pressure is much lower and, therefore, the diameter of the pipe is greater. Thus some thought should be given to limiting the length of these lines by minimizing the transfer of reheat steam between various pieces of equipment. The current design employs three reheaters. These are being examined to see whether they can be relocated or combined.

- The fuel gas cooler, while not unique, is a special piece of equipment. British Coal includes an elaborate fuel gas cooler in their topping cycle design employing evaporators and superheaters. Nevertheless, this can be difficult to design and operate, and can be expensive. To simplify the design of this exchanger, its conversion from a second-stage superheater to a high-pressure BFW heater is being considered.

- To aid start-up and shutdown, the addition of two low-pressure boilers operating at about 45 psia and located between the high- and low-pressure economizers on both the clean and dirty flue gas streams is being evaluated. The low-pressure steam would be superheated to about 440°C (824°F) and sent to the LP steam turbine. The inclusion of a LP boiler could preclude the need for an auxiliary boiler during start-up.

- The flue gas leaving the clean-side economizers is at a much lower temperature than that for the dirty side, nominally 102°C (216°F) versus 129°C (264°F). A modification in the condensate routing leaving the steam turbine condenser takes advantage of that fact by routing all the condensate through the clean side first before splitting.

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LP
ECON

AIR
⟨8⟩

⟨30⟩

⟨31⟩

⟨9⟩

ID FAN

⟨14⟩

⟨32⟩

⟨15⟩

FGD
SCRUBBER

⟨38⟩
TO STACK

⟨18⟩

⟨37⟩

COAL PREP

COAL

⟨1⟩

⟨36⟩

SORBENT

⟨3⟩

LP
ECON

⟨4⟩

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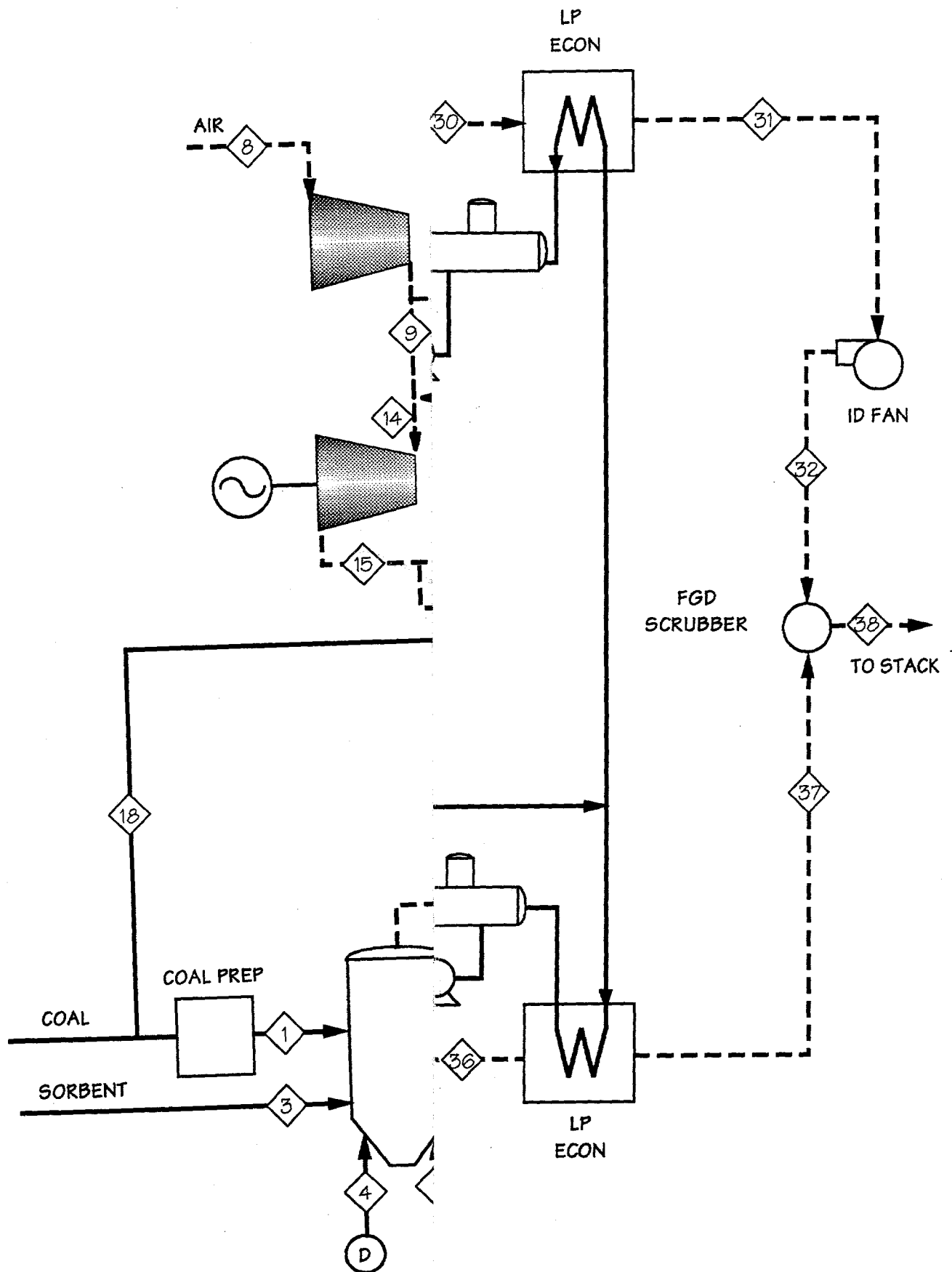
Table 1
Performance Summary

Coal Flow	kg/h	69,126
Limestone	kg/h	6,514
Steam	kg/h	28,757
Fuel Gas	kg/h	235,454
Char	kg/h	27,332
Air Heater Temperature	C	752
Gas Turbine Power	MWe	154.56
Steam Turbine Power	MWe	135.01
Gross Power	MWe	289.57
Auxiliary Power	MWe	13.01
Net Power	MWe	276.56
Heat Rate	kJ/kWh	7,236
Overall Efficiency (%)		49.73

Table 2
Major Process Flow Streams — All Coal Commercial Case
High Steam

Stream	Component	1 Coal to Pyrolyzer %wt	lb/hr	2 Air to Pyrolyzer %wt	lb/hr	3 Sorbent to Pyrolyzer %wt	lb/hr	4 Steam to Pyrolyzer %wt	lb/hr	5 Char fm Pyrolyzer %wt	lb/hr	6 Fuel Gas fm Pyrolyzer %wt	lb/hr	7 Fuel Gas fm Cooler %wt	lb/hr	8 Air to Compressor %wt	lb/hr
Stream	Carbon	71.92%	105,663							56.45%	34,013						
	Hydrogen	4.69%	6,890							0.36%	214						
	Oxygen	6.33%	9,297														
	Nitrogen	1.26%	1,848							0.99%	595						
	Sulfur	2.99%	4,398							0.93%	563						
Solids lb/h	Ash	10.32%	15,156			0.95%	136			25.39%	15,299						
	Moisture	2.50%	3,673			2.00%	287										
	CaCO3					95.50%	13,716										
	MgCO3					1.55%	223										
	CaO									2.43%	1,466						
	MgO									0.18%	106						
	CaS									13.28%	8,000						
	CaSO4																
Gases lb/h	CH4											1.21%	6,265	1.21%	6,222		
	C2H4																
	C2H6																
	C3H8																
	CO											21.04%	109,216	21.04%	108,454		
	H2											1.51%	7,843	1.51%	7,789		
	CO2			0.82%	2,895							15.39%	79,887	15.39%	79,330	0.82%	26,970
	H2O							100.00%	63,398			8.48%	44,034	8.48%	43,727		
	O2			23.07%	81,901											23.07%	763,060
	N2			76.11%	270,186							52.16%	270,756	52.16%	268,866	76.11%	2,517,270
	H2S											0.06%	297	0.06%	295		
	COS																
	SO2																
	C6H6+																
	Argon																
	NH3											0.15%	786	0.15%	781		
	NO2																
	Total Flow, lb/h	100.00%	146,925	100.00%	354,982	100.00%	14,362	100.00%	63,398	100.00%	60,257	100.00%	519,087	100.00%	515,462	100.00%	3,307,300
	M/H				12,365								22,160		22,006		115,203
	Mw				28.71								23.42		23.42		28.71
	Pressure, psia		258.20		263.90		258.20		232.52		238.22		238.22		228.22		14.60
	Temperature, °F		70.00		405.10		70.00		415.00		1,700.00		1,700.00		1,100.00		70.00
	Temperature, °C		21.11		207.28		21.11		212.78		926.67		926.67		593.33		21.11
	Sens. Heat, MM Btu/hr		0.39		33.17		0.04		10.24		25.71		293.03		177.45		8.86
	Lat. Heat, MM Btu/hr				3.04				66.57				46.24		45.91		28.32
	LHV, Btu/lb		12,445.05							8,931.56	8,962.95		1,964.71		1,967.29		
	HHV, Btu/lb		12,915.83							8,966.16	8,996.60		2,215.40		2,200.44		
	LHV, MM Btu/hr		1,828.49								540.08		1,021.20		1,014.07		
	HHV, MM Btu/hr		1,897.66								542.11		1,142.22		1,134.24		
Total Energy MM Btu/h	LHV		1,828.88		33.17		0.04		10.24		565.79		1,314.22		1,191.51		8.86
	HHV		1,898.05		36.21		0.04		76.81		567.81		1,435.25		1,311.69		37.17

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Table 2 (cont)
Major Process Flow Streams — All Coal Commercial Case
High Steam

Stream No.	9 Bypass Air		10 Air to Recuperator		11 Air to Furnace		12 Air to GT Combustor		13 FG fm GT Combustor		14 Total Gas to GT		15 Gas fm GT		16 Gas to Recuperator	
Stream	%wt	lb/hr	%wt	lb/hr	%wt	lb/hr	%wt	lb/hr	%wt	lb/hr	%wt	lb/hr	%wt	lb/hr	%wt	lb/hr
Carbon																
Hydrogen																
Oxygen																
Nitrogen																
Sulfur																
Solids lb/h																
Ash																
Moisture																
CaCO3																
MgCO3																
CaO																
MgO																
CaS																
CaSO4																
Gases lb/h																
CH4																
C2H4																
C2H6																
C3H8																
CO																
H2																
CO2			0.05%	1,039	0.05%	1,039	0.05%	1,039	9.88%	267,839	7.72%	267,839	7.72%	267,839	7.72%	217,456
H2O	0.82%	6,172	0.82%	17,949	0.82%	17,949	0.82%	17,949	5.41%	146,651	4.41%	152,823	4.41%	152,823	4.41%	124,076
O2	23.07%	174,627	22.95%	503,934	22.95%	503,934	22.95%	503,934	13.02%	353,071	15.22%	527,698	15.22%	527,698	15.22%	428,434
N2	76.11%	576,080	74.90%	1,644,466	74.90%	1,644,466	74.90%	1,644,466	70.59%	1,913,642	71.80%	2,489,722	71.80%	2,489,722	71.80%	2,021,384
H2S																
COS																
SO2										0.02%	554	0.02%	554	0.02%	554	0.02%	450
C6H6+																
Argon			1.28%	28,050	1.28%	28,050	1.28%	28,050	1.03%	28,050	0.81%	28,050	0.81%	28,050	0.81%	22,774
NH3																
NO2										0.04%	1,094	0.03%	1,094	0.03%	1,094	0.03%	888
Total Flow, lb/h	100.00%	756,880	100.00%	2,195,438	100.00%	2,195,438	100.00%	2,195,438	100.00%	2,710,901	100.00%	3,467,780	100.00%	3,467,780	100.00%	2,815,461
M/H		26,364		75,471		75,471		75,471		93,604		119,969		119,969		97,402
Mw		28.71		29.09		29.09		29.09		28.96		28.91		28.91		28.91
Pressure, psia		222.10		223.10		222.40		221.70		208.70		208.70		16.10		16.10
Temperature, °F		740.40		740.40		1,052.70		1,385.00		2,559.20		2,196.94		1,123.83		1,123.83
Temperature, °C		393.56		393.56		567.06		751.67		1,404.00		1,202.74		606.57		606.57
Sens. Heat, MM Btu/hr		128.56		368.12		546.61		743.58		1,935.09		2,063.66		966.83		784.96
Lat. Heat, MM Btu/hr		6.48		18.85		18.85		18.85		153.98		160.46		160.46		130.28
LHV, Btu/lb																
HHV, Btu/lb																
LHV, MM Btu/hr																
HHV, MM Btu/hr																
Total Energy, MM Btu/h LHV		128.56		368.12		546.61		743.58		1,935.09		2,063.66		966.83		784.96
HHV		135.04		386.97		565.46		762.43		2,089.08		2,224.12		1,127.29		915.24

Table 2 (cont)

Major Process Flow Streams — All Coal Commercial Case
High Steam

Stream No.	17 Gas fm Recuperator %wt	17 lb/hr	18 Coal to Burners %wt	18 lb/hr	20 Prim Air to Burners %wt	20 lb/hr	21 Char to Burners %wt	21 lb/hr	22 Flue Gas fm Burners %wt	22 lb/hr	24 Secondary Air to Furn %wt	24 lb/hr	25 Total Flue Gas to Furn %wt	25 lb/hr	26 Flue Gas to SH#3 %wt	26 lb/hr
Stream																
Carbon			71.92%				53.25%	34,013								
Hydrogen			4.69%				0.34%	214								
Oxygen			6.33%													
Nitrogen			1.26%				0.93%	595								
Sulfur			2.99%				0.88%	563								
Solids lb/h																
Ash			10.32%				23.95%	15,299	2.14%	15,299			1.81%	15,299	1.81%	15,299
Moisture			2.50%													
CaCO3																
MgCO3																
CaO							2.29%	1,466	1.07%	7,685			0.91%	7,685	0.91%	7,685
MgO							0.17%	106	0.01%	106			0.01%	106	0.01%	106
CaS							12.52%	8,000								
CaSO4																
Gases lb/h																
CH4							0.07%	44								
C2H4																
C2H6																
C3H8																
CO							1.19%	762								
H2							0.09%	55								
CO2	7.72%	217,456			7.72%	50,383	0.87%	558	24.70%	176,886	7.72%	10,077	22.08%	186,963	22.08%	186,963
H2O	4.41%	124,076			4.41%	28,747	0.48%	307	4.41%	31,565	4.41%	5,749	4.41%	37,315	4.41%	37,315
O2	15.22%	428,434			15.22%	99,265	2.96%	1,890			15.22%	19,853	2.34%	19,853	2.34%	19,853
N2	71.80%	2,021,384			71.80%	468,338	0.00%	2	65.74%	470,822	71.80%	93,668	66.67%	564,490	66.67%	564,490
H2S																
COS																
SO2	0.02%	450			0.02%	104			1.16%	8,337	0.02%	21	0.99%	8,358	0.99%	8,358
C6H6+																
Argon	0.81%	22,774			0.81%	5,277			0.74%	5,277	0.81%	1,055	0.75%	6,332	0.75%	6,332
NH3							0.01%	5								
NO2	0.03%	888			0.03%	206			0.03%	221	0.03%	41	0.03%	262	0.03%	262
Total Flow, lb/h	100.00%	2,815,461	100.00%		100.00%	652,319	100.00%	63,880	100.00%	716,199	100.00%	130,464	100.00%	846,662	100.00%	846,662
M/H		97,402				22,567				22,713		4,513		27,227		27,227
Mw		28.91				28.91				31.53		28.91		31.10		31.10
Pressure, psia		15.40		25.00		16.10		25.00		14.50		15.40		14.50		14.50
Temperature, °F		894.23		60.00		1,123.83		600.00		3,254.28		894.23				
Temperature, °C		479.02		15.56		606.57		315.56		1,790.16		479.02				
Sens. Heat, MM Btu/hr		606.47				181.87		8.97		677.27		28.10		705.37		625.15
Lat. Heat, MM Btu/hr		130.28				30.18		0.32		33.14		6.04		39.18		39.18
LHV, Btu/lb				12,481.12				8,536.44								
HHV, Btu/lb				12,915.83				8,578.75								
LHV, MM Btu/hr								545.30								
HHV, MM Btu/hr								548.01								
Total Energy, LHV		606.47				181.87		554.27		677.27		28.10		705.37		625.15
MM Btu/h, HHV		736.75				212.05		557.30		710.41		34.14		744.55		664.33

Table 2 (cont)
Major Process Flow Streams — All Coal Commercial Case
High Steam

Stream No.	27		28		29		30		31		32		33		34	
Stream	Flue Gas fm SH#3		D Flue Gas to RH		Flue Gas to D. HP-ECO		Flue Gas to D LP-ECO		D. FG to ID Fan		D. FG to SO2 Scrub		Flue Gas to C SH		Flue Gas to C Evap	
	%wt	lb/hr	%wt	lb/hr	wt%	lb/hr	%wt	lb/hr	%wt	lb/hr	%wt	lb/hr	wt%	lb/hr	wt%	lb/hr
Solids lb/h																
Carbon																
Hydrogen																
Oxygen																
Nitrogen																
Sulfur																
Ash	1.81%	15,299	1.81%	15,130	1.81%	15,299	1.81%	15,299								
Moisture																
CaCO3																
MgCO3																
CaO	0.91%	7,685	0.91%	7,600	0.91%	7,685	0.91%	7,685								
MgO	0.01%	106	0.01%	105	0.01%	106	0.01%	106								
CaS																
CaSO4																
Gases lb/h																
CH4																
C2H4																
C2H6																
C3H8																
CO																
H2																
CO2	22.08%	186,963	22.08%	184,897	22.08%	186,963	22.08%	186,963	22.70%	186,963	22.70%	186,963	7.73%	207,379	7.73%	207,379
H2O	4.41%	37,315	4.41%	36,902	4.41%	37,315	4.41%	37,315	4.53%	37,315	4.53%	37,315	4.41%	118,326	4.41%	118,326
O2	2.34%	19,853	2.34%	19,634	2.34%	19,853	2.34%	19,853	2.41%	19,853	2.41%	19,853	15.22%	408,581	15.22%	408,581
N2	66.67%	564,490	66.67%	558,252	66.67%	564,490	66.67%	564,490	68.54%	564,490	68.54%	564,490	71.82%	1,927,717	71.82%	1,927,717
H2S																
COS																
SO2	0.99%	8,358	0.99%	8,265	0.99%	8,358	0.99%	8,358	1.01%	8,358	1.01%	8,358	0.02%	429	0.02%	429
C6H6+																
Argon	0.75%	6,332	0.75%	6,262	0.75%	6,332	0.75%	6,332	0.77%	6,332	0.77%	6,332	0.81%	21,719	0.81%	21,719
NH3																
NO2	0.03%	262	0.03%	259	0.03%	262	0.03%	262	0.03%	262	0.03%	262	0.03%	847	0.00%	80
Total Flow, lb/h	100.00%	846,662	100.00%	837,306	100.00%	846,662	100.00%	846,662	100.00%	823,572	100.00%	823,572	100.03%	2,684,998	100.00%	2,684,231
MJH		27,227		26,926		27,227		27,227		27,227		27,227		92,888		92,871
Mw		31.10		31.10		31.10		31.10		30.25		30.25		28.91		28.90
Pressure, psia		14.40		14.20		13.40		13.20		13.10		15.10		0.26		15.40
Temperature, °F		2,103.91		1,300.95		680.20		352.10		266.40		280.10		894.23		867.60
Temperature, °C		1,151.06		704.97		360.11		177.83		130.22		137.83		479.02		464.22
Sens. Heat, MM Btu/hr		471.63		269.45		129.96		59.78		42.03		44.85		578.37		558.80
Lat. Heat, MM Btu/hr		39.18		38.75		39.18		39.18		39.18		39.18		124.24		124.24
LHV, Btu/lb																
HHV, Btu/lb																
LHV, MM Btu/hr																
HHV, MM Btu/hr																
Total Energy, LHV		471.63		269.45		129.96		59.78		42.03		44.85		578.37		558.80
MM Btu/h, HHV		510.81		308.20		169.14		98.96		81.21		84.03		702.61		683.04

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Table 2 (cont)
Major Process Flow Streams — All Coal Commercial Case
High Steam

Stream No.	35 FG to C. HP Econ wt%	lb/hr	36 FG to C. LP Econ wt%	lb/hr	37 C. FG to SO2 Scrub wt%	lb/hr	38 FG to Stack wt%	lb/hr
Stream								
Carbon								
Hydrogen								
Oxygen								
Nitrogen								
Sulfur								
Solids Ash								
lb/h Moisture								
CaCO3								
MgCO3								
CaO								
MgO								
CaS								
CaSO4								
Gases CH4								
lb/h C2H4								
C2H6								
C3H8								
CO								
H2								
CO2	7.73%	207,379	7.73%	207,379	7.73%	207,379	11.93%	394,342
H2O	4.41%	118,326	4.41%	118,326	4.41%	118,326	5.63%	199,687
O2	15.22%	408,581	15.22%	408,581	15.22%	408,581	11.38%	428,434
N2	71.82%	1,927,717	71.82%	1,927,717	71.82%	1,927,717	70.25%	2,492,207
H2S								
COS								
SO2	0.02%	429	0.02%	429	0.02%	429	0.02%	513
C6H6+								
Argon	0.81%	21,719	0.81%	21,719	0.81%	21,719	0.78%	28,050
NH3								
NO2	0.00%	80	0.00%	80	0.00%	80	0.00%	106
Total Flow, lb/h	100.00%	2,684,231	100.00%	2,684,231	100.00%	2,684,231	100.00%	3,548,008
MlH		92,871		92,871		92,871		122,319
Mw		28.90		28.90		28.90		29.01
Pressure, psia		15.30		15.10		15.00		14.70
Temperature, °F		692.60		329.40		217.60		118.40
Temperature, °C		367.00		165.22		103.11		48.00
Sens. Heat, MM Btu/hr		432.93		180.98		105.64		52.48
Lat. Heat, MM Btu/hr		124.24		124.24		124.24		209.67
LHV, Btu/lb								
HHV, Btu/lb								
LHV, MM Btu/hr								
HHV, MM Btu/hr								
Total Energy, LHV MM Btu/h		432.93		180.98		105.64		52.48
HHV		557.17		305.23		229.88		262.15

Heat to Stm

SH #1	41.01	SH #2	123.55
FG Comb	107.91		
SH #3	153.52	Dirty HP Econ	70.19
Dirty HRSG	87.93	Dirty LP Econ	17.74
Clean HRSG	453.15	Clean Evap	125.87
Precomb		Clean HP Econ	251.94
Char Comb	58.87	Clean LP Econ	75.34
Reheater	68.99	Eum. Evap	42.99
Total	**1,137.93**	**Total Coal**	**146,925**
Steam Turbine	135.14 MWe		1,898.05
GT	154.56	Heat Rate	8,420 / 8,571
Total Gross	**289.70**	Recuperator	178.49
Net Power	276.70	Prim Air Htr	196.97
Heat Rate	6,859.61	ID Fan	0.83 MWe
	49.76%	Booster Comp	1.34 MWe
		Misc	10.83 MWe

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- Implementation of some of these alternatives could adversely impact the overall performance of the system. A cost and performance benefit analysis will be carried out before final implementation is made.

A temperature increase for the ceramic air heater version of HIPPS was discussed among FWDC, AlliedSignal, and Westinghouse. It was indicated by Westinghouse that through the use of new materials they may be able to use a higher air temperature going to the combustor, perhaps as high as 1205°C (2200°F). The commercial design using the external ceramic exchanger was based on temperatures of 760°C (1400°F) leaving the alloy air heater and 982°C (1800°F) leaving the ceramic air heater (HTHE) going to the gas turbine combustor. Bechtel was asked to determine what impact raising the HTHE outlet temperature to 1205°C (2200°F) would have on overall plant performance.

It was obvious that the higher temperatures would reduce the natural gas requirements, but other changes were less predictable. Several spreadsheet cases were developed to uncover any trends. These were supported by two GateCycle runs for confirmation. The results from the spreadsheet runs are presented in Table 3.

On the surface it would appear that increasing the outlet temperature should improve the overall performance since it would put more energy from the fuel gas into the air, and hence, into the gas turbine. Generally, the greater the amount of energy from the coal that can be put into the gas turbine, the better. However, the larger demand of energy by the HTHE requires additional fuel gas from the pyrolyzer. Unfortunately, as the fuel gas increases, so does the char production. Since the amount of energy in the char used to heat the air in the primary heater is fixed, the energy from any additional char to the furnace goes directly to the bottoming cycle. The result is that what is gained in the HTHE is more than lost in the extra char to the furnace.

Based on the results obtained last quarter with steam feed to the pyrolyzer, a natural extension would be to increase the steam to the pyrolyzer for the natural gas configuration as well. The results from this analysis were also unexpected. Comparing Cases 1 through 3 and 4 through 5, it can be seen that increasing the steam to the pyrolyzer had a negative impact on overall plant performance. This contradicts the results obtained for the configuration used in the "all coal" design.

The reason for the drop appears to be the cumulative effect of several factors. First, unlike the "all coal" case where increasing the water content resulted in an increase in power from the gas turbine because of the increase in mass flow, in the ceramic air heater system the gas turbine power remains constant. Since the primary air heater temperature was not allowed to change as the amount of char to the furnace was reduced, the size of the bottoming cycle was also reduced. Generally, this would be good. However, without changing the air heater temperature to help balance the high- and low-level heat sources, the bottoming cycle gradually becomes less efficient. While this analysis was very brief, it appears from the results in Table 3 that the benefits of increasing the ratio of gas turbine to steam turbine power is more than offset by the loss in efficiency of the bottoming cycle. This would have to be confirmed in a more detailed analysis. Whether there is a slight drop in efficiency, as shown in Table 3, or a slight rise (if the algorithm used for the steam system was too severe), the

Table 3
Summary of Indirect Heating Results

		Ref Case	Case 1	Case 2	Case 3	Case 4	Case 5	Case 6	Case 7
HTHE Inlet Temp	C	760	760	760	760	760	760	760	871
HTHE Outlet Temp	C	927	927	927	927	1,204	1,204	1,204	1,204
Coal to Pyrolyozer	kg/h	49,550	54,818	45,008	43,481	117,258	95,348	91,646	73,874
Coal to Char Burner	kg/h	4,955	5,482	4,501	4,348	11,726	9,535	9,165	7,387
Natural Gas	kg/h	15,009	15,009	15,009	15,009	6,372	6,372	6,372	6,372
Total Energy In	kJ/h x 10^6	2,498	2,650	2,380	2,354	4,234	3,625	3,554	2,889
Steam to Pyrolyzer	kg/kg coal	16.18	0.00	43.15	64.73	0.00	43.15	64.73	43.15
Char to Combustor	kg/h	23,693	33,543	19,382	15,997	71,748	41,059	33,718	31,812
Gas Turbine	MWe	130.6	130.6	130.6	130.6	123.4	123.4	123.4	123.4
Steam Turbine	MWe	213.7	241.4	195.0	177.7	386.0	299.1	269.9	249.6
Gross Power	MWe	344.3	372.0	325.6	308.3	509.4	422.5	393.3	373.1
Total Auxiliary Power	MWe	9.1	9.6	8.5	8.2	19.0	13.4	12.8	10.9
Net Power	MWe	335.2	362.4	317.1	300.1	490.4	409.1	380.6	362.2
Heat Rate	kJ/kWh								
Efficiency	%	48.4%	49.3%	48.0%	45.9%	41.7%	40.7%	38.6%	45.2%
HTHE									
Duty	kJ/h x 10^6	224.5	224.5	224.5	224.5	609.0	609.0	609.0	459.9
Gas Flow	kg/h	461,989	377,342	450,180	484,704	742,508	883,690	948,831	684,666

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results indicate that injecting steam into the pyrolyzer does not have a significant beneficial impact on system performance.

As an aside, an additional run (Case 7) was made also assuming that the outlet temperature from the HTHE was increased to 1205°C (2200°F). However, for Case 7 the temperature from the primary air heater was increased from 760°C (1400°F) to 870°C (1600°F). This has two benefits: 1) it reduces the demands on the HTHE, thereby reducing the amount of fuel required, and 2) it provides a means to get more heat from the char into the gas turbine. As seen in Table 3, this change provides a significant increase over Case 5, but it still falls short of the performance estimated for Case 2 (the other two cases using the same steam injection ratio).

Based on Table 3, the bottom line is that there does not appear to be any significant benefit from increasing the temperature from the HTHE.

Task 2 - Engineering Research and Development

Subtask 2.2 - Restrictive Pipe Discharge (RPD) System Cold Model

The cold RPD tests have been completed. The results of these tests were reported in the January-March 1996 Technical Progress Report [1]. Three particle size distributions (PSD's) were tested. The coarsest and the medium-size distributions performed well, but the flow with the finest PSD was somewhat erratic. The repercussions of these results are discussed further under Subtask 3.1.

Task 3 - Subsystem Test Unit Design

Subtask 3.1 - Pyrolyzer/Char Transport Test Design

General Status

A Design Review Meeting was held for the pyrolyzer/char transport test (PCTT). FWDC personnel and representatives from other Foster Wheeler units were present. The test plan and equipment design were presented. For the most part, the PCTT test plan and design were accepted with minor design changes. There were, however, two more significant changes to the plan. These changes involved the addition of another pyrolyzer arrangement and a change in the char depressurization system.

Some participants in the Design Review Meeting believed that it may be possible to use a bubbling bed arrangement for the pyrolyzer. Operationally this type of system would be simpler than a circulating bed because there would be no solids recirculation system. There are, however, some process concerns related to the HIPPS requirements. Because of the fine char size requirement, the bubbling bed will operate with a relatively short char residence time. This situation could cause uneven bed temperatures and fluctuations in the fuel gas conditions. Because of the possible benefits of bubbling bed operation, it was decided to test this arrangement in addition to the circulating bed tests.

The pilot plant currently has a bubbling bed pyrolyzer, so tests can be run with a shorter lead time than the circulating bed tests. The plan is to make a few modifications to the existing plant and then test the bubbling bed pyrolyzer while we are waiting for delivery of the long-lead items for the circulating bed pyrolyzer. Modifications done prior to the bubbling bed tests will mainly be to the coal-feed system and the instrumentation. By using existing equipment, it is possible to do these tests without any new pressure vessels. Testing will then be done while we are waiting for the delivery of the vessels required for the circulating bed tests.

The other change to the PCTT plan is the design of the char depressurization system. It was originally planned to use a restrictive pipe discharge (RPD) system. Tests were run at the Institute of Gas Technology (IGT) to investigate the operation of this system with the char. Three PSD's were tested representing a range of possible char PSD's. The medium and coarse size distributions performed well in these tests, but the fine distribution (90 percent through 200 mesh) had erratic char flow. Although the required char PSD for the Foster Wheeler arch-fired combustion system will be determined in subsequent tests, these systems generally use relatively fine PSD's for low-volatile coal. It was therefore decided to proceed with a lock hopper system. If the combustion system tests indicate that a coarser PSD is possible, the use of an RPD can be reevaluated and added as a later improvement.

Test Matrix

The PCTT test matrix has been revised to include the bubbling bed tests. Also, the level of steam injection into the pyrolyzer has been included as a variable. Bechtel is finding that plant efficiency can be maximized with an optimal level of steam injection into the pyrolyzer. The steam injection can be used to shift the ratio of fuel gas to char and thereby optimize the topping and bottoming cycles.

The revised test matrix is shown in Table 4. To explore all possible combinations of the variables would result in an unreasonable number of test runs. The number of test runs has been reduced because of two factors. One factor is that a computer model will be developed concurrently with the pyrolyzer testing. The data from each test run will be used to benchmark and modify the computer model. This approach will require less test data than a completely empirical method. Also, some of the variables will become "go–no go" decisions, and once the decision is made, one variable is removed from the test matrix. For example, initial test runs will have set points with coarse sorbent and other set points with fine sorbent. With coarse sorbent, the goal is to remove as much spent sorbent from the bottom of the bed as possible. The fine sorbent approach is to let the sorbent go with the char and have a bed of sand. The most feasible approach will be chosen based on initial testing, and the other approach will be dropped from the test matrix.

Data Acquisition

The HIPPS pilot plant will be controlled by the same Fisher Provox DCS used for the Second-Generation PFB test program. The original DCS system included a VAX/VMS-based data historian package which will be modified for the HIPPS program. This upgraded data archiving package will

Table 4
Pyrolyzer Test Matrix

Run No.	Bed Arrangement	Coal Size	Sorbent Size	Sand	Steam Flow (kg/h)	Riser Velocity (m/s)	Cyclone Design	Purpose
1	Bubbling	C1	S1	No	24.4	0.95	--	• Obtain design data for bubbling bed operation with initial coal size.
		C1	S1	No	97.5	0.95	--	
		C1	S2	Yes	24.4	0.95	--	
		C1	S2	Yes	97.5	0.95	--	
2	Bubbling	C2	S1	No	24.4	0.95	--	• Obtain design data for bubbling bed operation with second coal size.
		C2	S1	No	97.5	0.95	--	
		C2	S2	Yes	24.4	0.95	--	
		C2	S2	Yes	97.5	0.95	--	
3	Circulating	C1	S1	No	24.4	1.52	CY1	• Determine whether to design for sorbent carryover or sorbent separation.
		C1	S1	No	24.4	2.74	CY1	• Check initial coal size with circulating bed.
		C1	S2	Yes	24.4	1.52	CY1	
		C1	S2	Yes	24.4	2.74	CY1	
4	Circulating	C3	TBD	TBD	24.4	1.52	CY1	• Obtain design data for circulating bed with initial cyclone design.
		C3	TBD	TBD	24.4	2.74	CY1	
		C3	TBD	TBD	24.4	TBD	CY1	
		C3	TBD	TBD	97.5	1.52	CY1	
		C3	TBD	TBD	97.5	2.74	CY1	
		C3	TBD	TBD	97.5	TBD	CY1	
5	Circulating	TBD	TBD	TBD	TBD	1.52	CY2	• Obtain design data for circulating bed with second cyclone.
		TBD	TBD	TBD	TBD	2.13	CY2	
		TBD	TBD	TBD	TBD	2.74	CY2	
		TBD	TBD	TBD	TBD	3.65	CY2	
6	Circulating	TBD	TBD	TBD	TBD	1.52	TBD	• To obtain data as required.
		TBD	TBD	TBD	TBD	2.13	TBD	
		TBD	TBD	TBD	TBD	2.74	TBD	
		TBD	TBD	TBD	TBD	3.65	TBD	

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be supported by a Unix-based HP Workstation acting as a main file server. The database will be accessible through a local-area network, and all of the client PC's will be outfitted with X emulation software. This data-archiving system will facilitate open data exchange between the Unix system and the PC's.

In an effort to address the scale-up issues of the pyrolyzer, a high-speed data-acquisition system will be installed to monitor differential pressure fluctuations along the riser. The data gathered from these high-speed transmitters (100 Hz) will be used to study the underlying hydrodynamic behavior of the pyrolyzer and to validate the modeling efforts being performed at M.I.T. The tests to be performed at M.I.T. utilize small-scale "cold" (70°F., 14.7 psia) experimental models constructed of plexiglass to simulate "hot" (1700°F., 200 psia) operating performance. A series of non-dimensional scaling parameters is matched between the cold model and the hot pilot plant to ensure hydrodynamic similitude.

An on-line mass spectrometer will be used during the Phase 2 HIPPS program to monitor the transient behavior of the pyrolyzer gas quality. The gas heating value and ammonia content are of particular concern to the gas turbine manufacturers, as these values affect power output and NOx emissions, respectively. The mass spectrometer affords an accurate measurement of each individual fuel gas component with a total sample response time of less than 5 seconds. The transient behavior of each fuel gas component will be monitored to determine the optimum air-staging requirements for the gas turbine.

Pilot Plant Design

The general circulating bed arrangement of the PCTT pilot plant is shown in Figures 6 and 7. The arrangement is basically the same as previously reported [2] except a lock hopper system replaces the RPD system. Bids on this equipment will be evaluated in July and a purchase order will be issued as soon as approval is received. Key features of the pilot plant design are the ability to vary the riser and cyclone inlet velocity and the ability to replace or modify the cyclone easily.

One change from previous operation of this pilot plant is the use of pulverized coal and sorbent. The current feed system uses separate trains consisting of a bucket elevator, silo, and Z-belt conveyor to transport the feedstocks up to the lock hoppers. Two aspects of these systems were in question: the ability to reliably feed the finer material and the safety requirements with the fine coal. Tests were run with the sorbent train feeding pulverized limestone. The material was pneumatically fed from a tank truck directly to the silo, bypassing the bucket elevator. The material was then transported up to the lock hoppers with the Z-belt conveyor and through the lock-hopper system. The system performed well, and it will be used essentially as is for the sorbent feeding.

In analyzing the coal feed system, it was determined that the changes that would be required for safe operation of the Z-belt conveyor would be fairly extensive. It was decided that it would be better to replace the Z-belt conveyor with a nitrogen-blown, dense-phase transport system which would be both safer and more reliable. Bid packages have been prepared for this equipment, and the purchase order will be placed in July.

Subtask 3.2 Char Combustion System Test Design

The FWEC arch-fired combustion system is now the base case system for char combustion in HIPPS. The concept is illustrated in Figure 8. The lower furnace has two arches where the burners are located. The burners fire down into the furnace, and combustion air is added in stages through the front and rear walls. This type of arrangement causes a long flame path where the air supply is gradually added to the fuel to avoid quenching. This approach also results in essentially a staged combustion which tends to minimize NOx. In addition to the air staging, the geometry of the furnace promotes recirculation of hot gases from the upper furnace back into the flame. This situation is also a stabilizing influence on the flame and the flame temperature.

One difference between the HIPPS firing system and current arch-fired systems will be the burner itself. In conventional arch-fired systems, primary air is removed from the fuel stream by having a cyclone directly upstream of the point where the burner injects into the furnace. This helps to accomplish the air-staging effect. In the HIPPS system, the char will be transported by an inert gas, probably fuel gas. A part of the HIPPS char burner development will be to determine how best to add air at the burners.
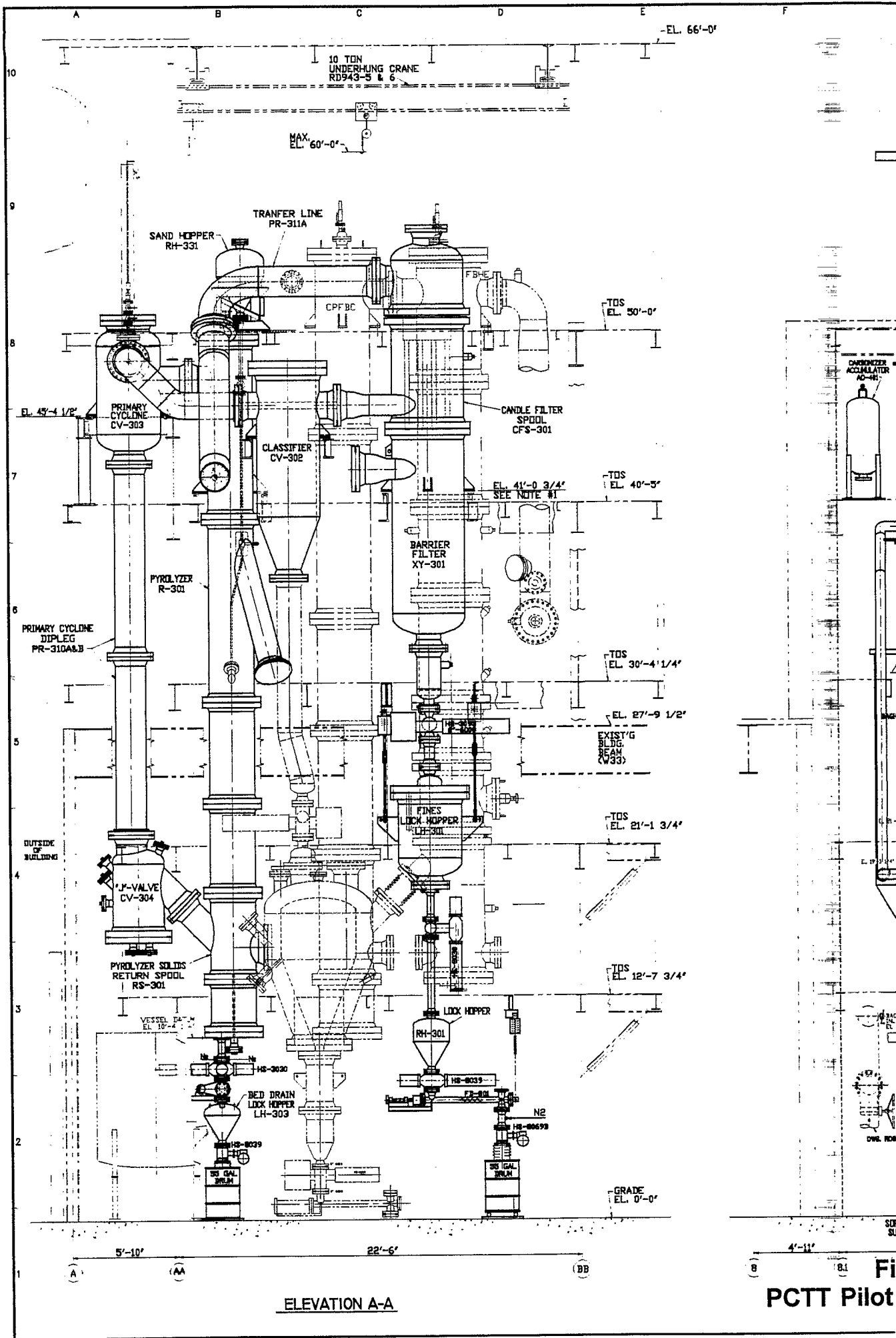
Combustion tests will be run at the Foster Wheeler Combustion and Environmental Test Facility (CETF) in Dansville, New York. This facility is used to test burners. It consists of a furnace and convection pass that was designed to simulate conditions in larger scale boilers. Under HIPPS conditions, the facility will be capable of approximately 30MM Btu/h heat input to the burner. The furnace was originally built for arch firing, but it was later converted to horizontal firing. It will be converted back to arch firing with a design that will facilitate changing back and forth between the two configurations.

The CETF has fans, a pulverizer, and other auxiliary equipment required for a conventional combustion system. However, some additional equipment will be required to simulate the HIPPS conditions. A gas-fired duct heater and recirculation of flue gas will be used to obtain hot, reduced-oxygen-level combustion air.

The char combustion system tests will be run concurrently with the PCTT. Most of the tests will be run with commercially produced char, but char generated in the PCTT will also be used to the extent that it is available.
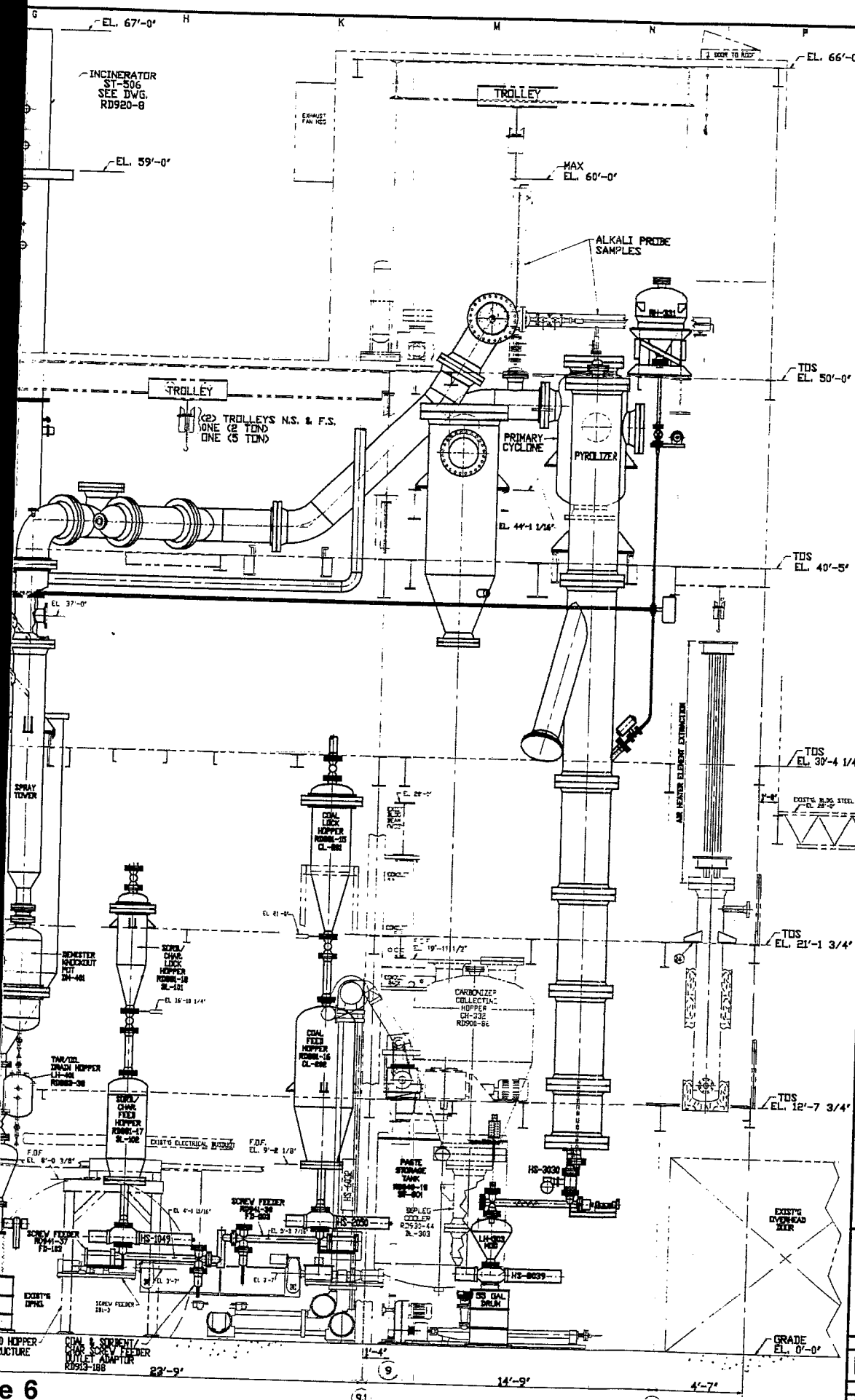
Subtask 3.3 - Integrated System Test Design (IST)

UTSI continued preliminary work to prepare for the IST. To help coordinate mechanical interfaces for the IST, packages of drawings of the coal-fired flow facility's (CFFF) mechanical systems were prepared for FWDC. Design details for the furnace, superheater and air heater, downstream pollution control equipment, and major support systems were provided. A Drafting Supervisor was assigned, initially to prepare drawing packages of the CFFF systems for FWDC. Using AutoCAD Release 12, the selected common software interface for drafting and mechanical design drawings, these packages

ELEVATION A-A

Fig
PCTT Pilot

EL. 67'-0"

INCINERATOR
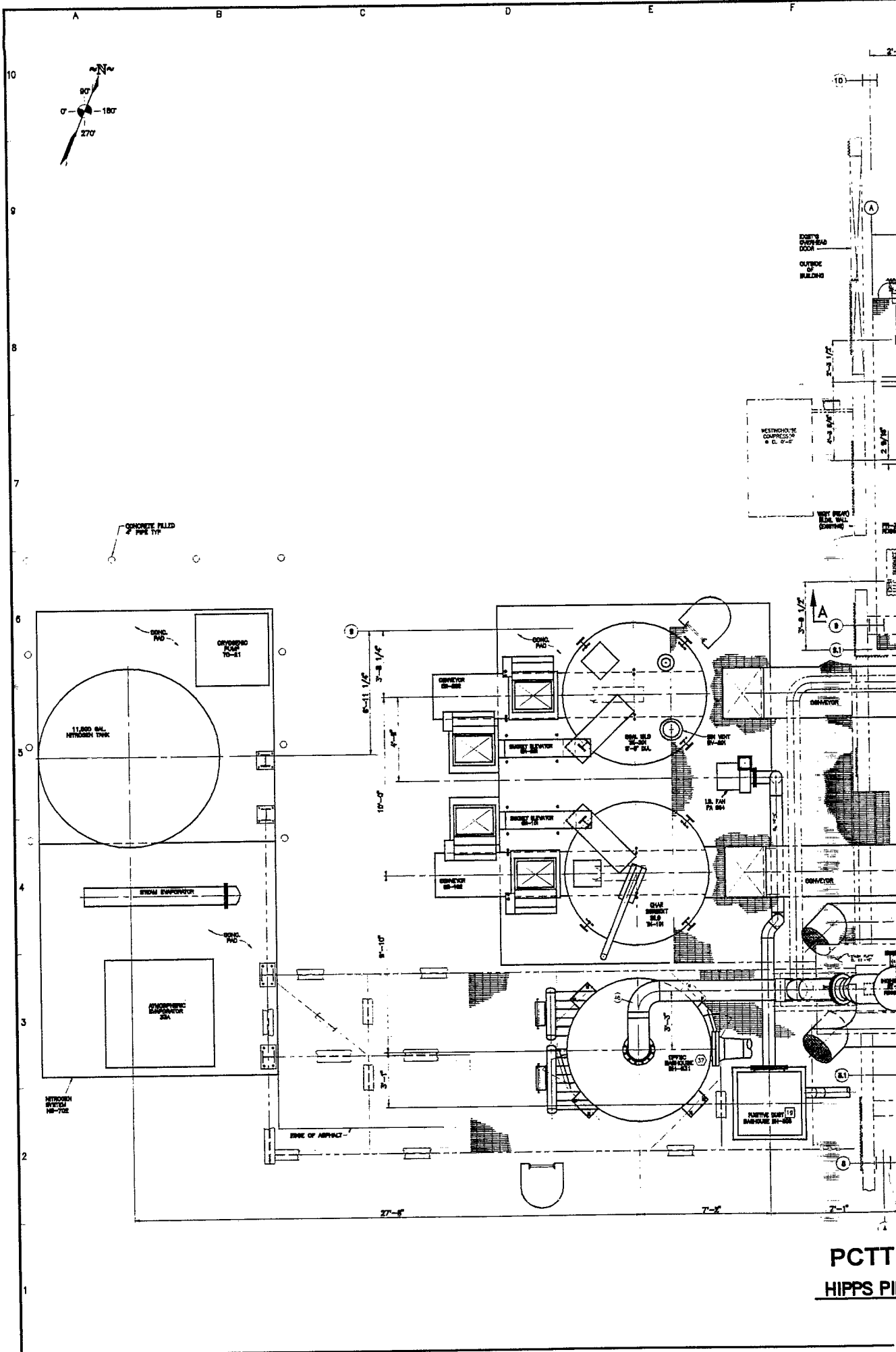ST-506
SEE DWG.
RD920-8

EL. 59'-0"

TROLLEY

MAX
EL. 60'-0"

ALKALI PROBE
SAMPLES

DOOR TO ROOF

EL. 66'-0"

TDS
EL. 50'-0"

NOTES

TROLLEY

(2) TROLLEYS N.S. & F.S.
ONE (2 TON)
ONE (5 TON)

PRIMARY
CYCLONE

PYROLIZER

EL. 44'-1 1/16"

TDS
EL. 40'-5"

EL. 37'-0"

SPRAY
TOWER

COAL
LOCK
HOPPER
RD900-15
CL-801

TDS
EL. 30'-4 1/4"

EXISTG BLDG. STEEL
EL. 28'-0"

EL. 28'-0"

EL. 21'-0"

SCREW
CHAR
LOCK
HOPPER
RD900-16
SL-401

TDS
EL. 21'-1 3/4"

CARBONIZER
COLLECTION
HOPPER
CH-332
RD900-86

ROCHESTER
KNOCKOUT
POT
DH-401

EL. 16'-10 1/4"

TAR/OIL
DRAIN HOPPER
LH-401
RD900-38

COAL
FEED
HOPPER
RD900-16
CL-802

SCREW
CHAR
FEED
HOPPER
RD900-17
SL-402

F.O.F.
EL. 9'-2 1/8"

PASTE
STORAGE
TANK
RD900-18
SN-801

HS-3030

F.O.F.
EL. 8'-0 3/8"

SCREW FEEDER
RD900-30
FS-303

EL. 4'-1 11/16"

HS-2050

DIPLEG
COOLER
RD900-44
SL-303

EXIST'G
OVERHEAD
DOOR

PRELIMINARY
7/30/96

SCREW FEEDER
RD901-37
FS-152

HS-1049

LH-303
HOP

EL. 3'-7"

EL. 2'-7"

SCREW FEEDER
281-3

55 GAL
DRUM

HS-8039

HOPPER
STRUCTURE

COAL & SORBENT/
CHAR SCREW FEEDER
OUTLET ADAPTOR
RD913-188

23'-9"

11'-4"

9

14'-9"

9.5

4'-7"

GRADE
EL. 0'-0"

EQUIPMENT
GENERAL ARRANGEMENT
ELEVATION A-A & B-B
HIPPS
PILOT PLANT
PHASE 2

RD950-81

REV
A*

DRAWING NUMBER

SCALE: 3/8" = 1'-0"

DRAWN BY: B. KULCSAR 12/11/95

CHECKED BY:

APPROVED BY:
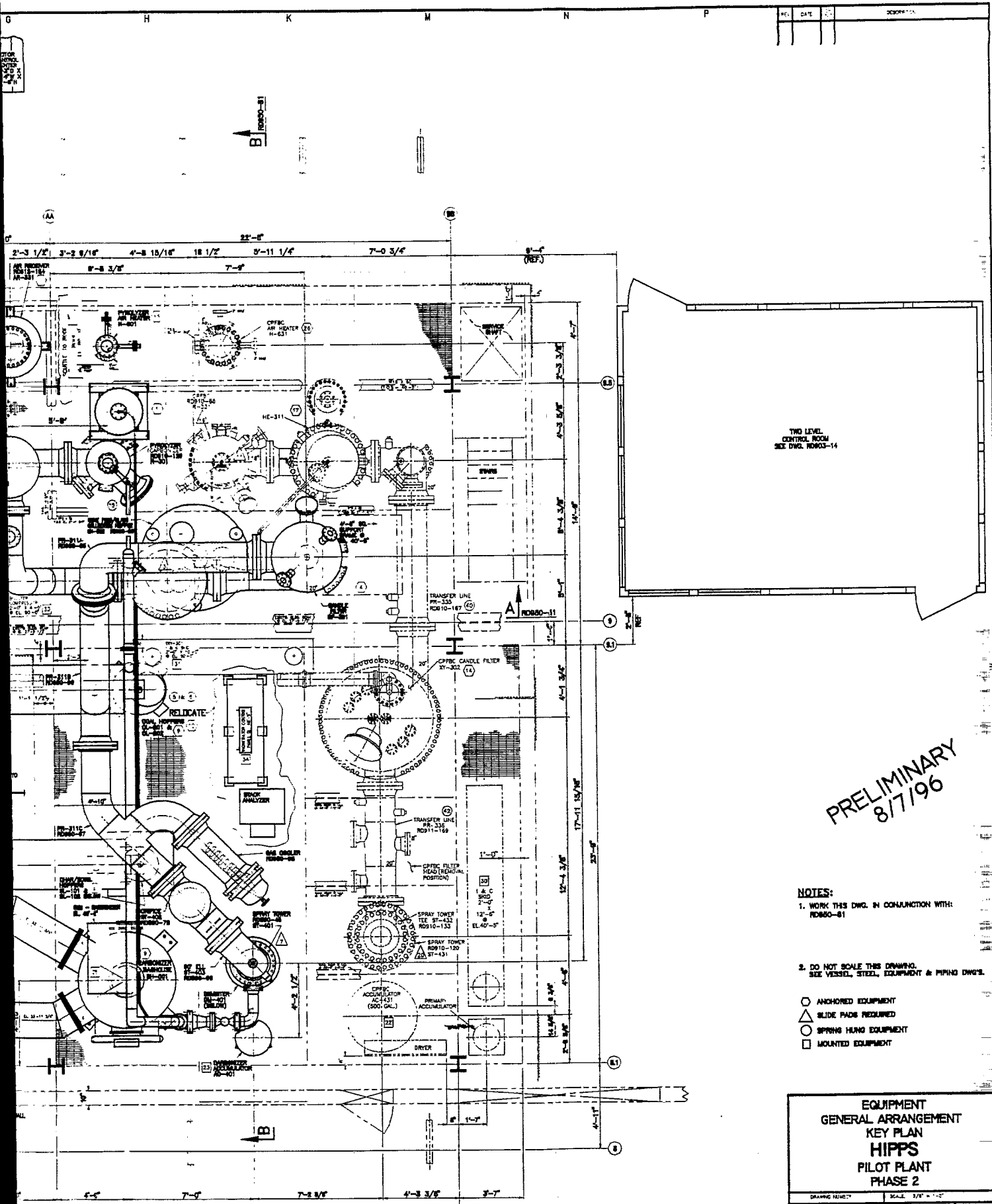
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12 PEACH TREE HILL RD. LIVINGSTON, N.J.

FW

e 6

nt Elevation

ELEVATION B-B

~N~

90°

0° — 180°

270°

10

9

8

7

6

5

4

3

2

1

A   B   C   D   E   F

CONCRETE FILLED 2" PIPE TYP

CONC. PAD

CRYOGENIC PUMP TO-21

11,000 GAL. NITROGEN TANK

STEAM EVAPORATOR

CONC. PAD

ATMOSPHERIC EVAPORATOR 30A

NITROGEN SYSTEM NS-702

EDGE OF ASPHALT

WESTINGHOUSE COMPRESSOR @ EL. 0'-0"

DOSTA OVERHEAD DOOR

OUTSIDE OF BUILDING

CONC. PAD

CONVEYOR BN-308

BUCKET ELEVATOR BE-93

BUCKET ELEVATOR BE-101

CONVEYOR BN-102

BH VENT BV-801

SOLID BLD TK-201 8'-0" DIA.

CONVEYOR

I.D. FAN FN-801

CHAR ELEMENT BLD TK-101

CONVEYOR

FUSTIVE DUST BAGHOUSE BH-805

CRYOGENIC BAGHOUSE BH-801

27'-6"   7'-2"   2'-1"

PCTT

HIPPS PIL

TWO LEVEL
CONTROL ROOM
SEE DWG. RD903-14

PRELIMINARY
8/7/96

NOTES:
1. WORK THIS DWG. IN CONJUNCTION WITH:
   RD950-81

2. DO NOT SCALE THIS DRAWING.
   SEE VESSEL, STEEL, EQUIPMENT & PIPING DWG'S.

○ ANCHORED EQUIPMENT
△ SLIDE PADS REQUIRED
○ SPRING HUNG EQUIPMENT
□ MOUNTED EQUIPMENT

Figure 7
...t Plant Plan

...ANT LIVINGSTON, N.J.
...EY PLAN

EQUIPMENT
GENERAL ARRANGEMENT
KEY PLAN
HIPPS
PILOT PLANT
PHASE 2

DRAWING NUMBER   SCALE 3/8" = 1'-0"

RD950-76   REVISION

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Figure 8 Arch Firing

will probably be provided on magnetic media. Bids were received and evaluated for job-shop drafting and design services, to assist in the IST design effort.

As part of the original plans to replace outmoded parts of the CFFF central data acquisition and control system with a distributed control system (DCS), specifications were completed, and competitive bids were requested. This DCS will be used both for coal processing in support of the PCTT and for the IST. To mitigate time as well as dollar costs of the IST, specifications require compatibility of the DCS with the Foster Wheeler control system being used for the PCTT. UTSI is coordinating acquisition of the DCS with FWDC to ensure that the system satisfies test and compatibility requirements.

A letter was received from the Tennessee Air Pollution Control Division stating that the IST test does indeed qualify under Title V of the CAAA as "conditional major", thus allowing the CFFF to be

classified as a "pilot plant" and "insignificant emission unit". This places the CFFF in a categorically exempt status and allows testing to proceed with only limited air permit oversight and emissions monitoring. However, they are requiring updated permit applications to assure that the test does not significantly impact air quality. Regular emissions monitoring, although not continuous monitoring, will still be required.

UTSI is reviewing alternatives for supplying high-pressure steam to the pyrolyzer and feeding sorbent to the pyrolyzer. The CFFF's current steam capability is 1.03 MPa (150 psig) steam, which is insufficient for pyrolyzer operation. Required steam pressure is expected to be about 1.38 MPa (200 psig). Bids for both new and used boiler systems providing high-pressure steam for injection into the pyrolyzer during the IST were examined. Replacement of the CFFF low-pressure 5.89 MW (600 bhp) boiler was also considered, as an alternate option. An existing location with appropriate foundation and building was identified, but these boilers would still require a preheat tank as well as utility installation. A decision will wait for completion of contract negotiations and continued availability of the used boilers, which seem the best option at present.

An investigation of the coal-feed system indicated a possible requirement for 2.07 MPa (300 psia) driving pressure, well above the 1.38 MPa (200 psi) available from the existing CFFF nitrogen system. Thus budgetary pricing was received for a new N_2 booster compressor. The estimated cost for a new compressor and alternate costs to rebuild an existing unneeded CFFF air compressor to accommodate N_2 are being compared.

Existing components of the CFFF coal-preparation system were reviewed for use in the HIPPS sorbent feed system and some components will be usable. Existing components of the UTSI Energy Conversion Facility coal system were also examined but found unsuitable for use in the sorbent-feed system because of pressure rating. Also, plans were initiated to examine the seed-injection system being excessed by a DOE facility in Montana in July. Preliminary indicates it may provide a satisfactory system.

Task 4 - Subsystem Test Unit Construction

The equipment-purchasing process has started for all three pilot plants.

REFERENCES

1. Foster Wheeler Development Corporation, "Engineering Development of Coal-Fired High Performance Power Systems- Technical Progress Report 2 (October through December 1995)", DOE PETC under contract DE-AC22-95PC95143, February 1996, pp 5-11.

2. Foster Wheeler Development Corporation, "Engineering Development of Coal-Fired High Performance Power Systems- Technical Progress Report 2 (October through December 1995)", DOE PETC under contract DE-AC22-95PC95143, February 1996, pp 18-30.